## APPENDIX No. 14.

ON THE FLEXURE OF PENDULUM SUPPORTS.

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#### HISTORICAL.

The fact that the rate of a pendulum might be largely influenced by the elastic yielding of its support was first pointed out by Dr. Thomas Young in his article on Tides in the Encyclopædia Britannica, where he gave a correct mathematical analysis of the problem. Kater made use of the noddy, or inverted pendulum of Hardy, to assure himself that its support was sufficiently steady.

Hardy's noddy is a pendulum turning with a reed spring and provided with an adjustable bob. It differs from an ordinary pendulum, first, in being upside down, that is, having its center of mass above its point of support; and second, in having a spring so strong as to act a little more strongly than gravity. The force tending to bring the pendulum to the vertical is then the excess of the force of the spring over the moment of gravity. In this way the noddy is easily adjusted so as to have the same period of oscillation as the pendulum used to determine gravity, while its moment of inertia is very small. In a note at the end of this paper I give the mathematical analysis of this state of things, from which it will be seen that Kater might have constructed his noddy in such a manner as to detect any amount of flexure sufficient to have a serious effect upon the period of his pendulum.

Bessel, at the end of §3 of his great memoir on the length of the seconds' pendulum at Königsberg, states that he also used Hardy's noddy, and that he swung his pendulum again after stiffening the support. He adds that the effect on the period would probably be the same for his long pendulum as for his short one—a very just remark—which made it less necessary for him to attend to the rigidity of the stand.

The construction of English pendulum supports, that of Bassevi, for example, shows that in that country this source of error was never overlooked. It is noticed even in brief accounts in English of the process of measuring gravity. Thus, a writer in the Encyclopædia Britannica proposed to make use of two different reversible pendulums of the same form but of different weights, in order to take account of the flexure, an idea lately borrowed by M. Cellerier.

When the reversible pendulum came into use the study of the writings of the older observers seems to have been neglected,\* and the grave errors due to flexure were never suspected until Albrecht found a value of gravity at Berlin differing by nearly 2 millimeters from that of Bessel. So little was the true cause of this discrepancy at first suspected that it was paradoxically attributed to the neglect of a buoyancy correction.

In 1875, however, General Baeyer gravely suspected that the period of a pendulum swinging upon a Repsold tripod was affected by the oscillation of the latter, and in a circular addressed to the members of the committee on the pendulum of the International Geodetic Congress, he wrote: "The necessity of suspending the pendulum from a stand is a source of error, since a pendulum swinging on a stand sets the latter into oscillation and so influences the rate of the former. The effect could be diminished by the use of a shorter pendulum and smaller stand; but whether it would be rendered entirely insensible is open to question."

<sup>\*</sup>Thus, Bessel's idea of directly measuring the position of the center of mass was supposed by the Swiss savans to belong to M. Cellerier.

It was at this time that I first received the Repsold apparatus from the makers, of whom it had been ordered two years before, on the occasion of my first being charged with the pendulum operations of the Coast Survey. Becoming acquainted with General Baeyer's doubts, I determined to settle the question by measuring the flexibility of the Repsold tripod at the earliest opportunity. This I did at Geneva, where, though I only made a rough measurement, I found that the flexure was fully sufficient to account for the discrepancy between the determinations of Bessel and of Albrecht.

On September 25 of the same year I communicated my result to the standing committee of the Geodetical Congress. At the same sitting the reports of the different members of the pendulum committee were read. Dr. Bruhus said: "The question whether the stand is set into oscillation, and whether the rate of the pendulum is influenced thereby is, in my opinion, well worth investigation. But I should suppose that the stand could be made so stiff as to eliminate this source of error for a pendulum used only as a relative instrument." The views of M. Hirsch, who is so much occupied with the going of time-keepers, are interesting. He said: "The fear that the tripod of suspension may also enter into oscillation, unless it be a fact established by direct observations, seems to me unfounded. Indeed, it cannot be supposed that there are any true oscillations of a body of such a form resting on three points. Besides, the movement of the pendulum whose mechanical moment (moment mécanique) is slight on account of its small velocity, could only be communicated to the tripod by the friction of the knife on the supporting plane. Now, this friction is insignificant, as the slowness of the decrement of the amplitude shows, this being almost entirely due to the resistance of the air." It may be observed that the rolling friction of the knife edge is, in truth, very slight, but the amount of the sliding friction is sufficient to hold the knife in place on the supporting plane. Dr. vou Oppolzer, the designer of the Repsold tripod in its definitive form, said that the construction of the stand rendered any serious flexure a priori improbable; but he did not support this opinion by any calculations.

During the spring of 1876, having already measured the flexibility of the tripod in Paris, I remeasured it in Berlin, where my experiments were witnessed by General Baeyer and a party of gentlemen attached to the Prussian Survey.

In October, 1876, at the meeting of the standing committee of the International Geodetical Union at Brussels, the result of my experiments was announced by General Baeyer. M. Hirsch described certain experimental researches undertaken by him to ascertain whether there was any such flexure in the case of the Swiss tripod. He had, in the first place, employed an extremely sensitive level, which had not entered into oscillation while the pendulum was swinging upon it. It is not clear why M. Hirsch employed a very sensitive level, the natural time of oscillation of which would differ much more from the period of the pendulum than that of a less sensitive level would do. He also used an artificial horizon in the same way. M. Hirsch's conclusion is that "there remains no doubt that the Swiss stand is free from every trace of such oscillations." Dr. von Oppolzer entirely agreed with the views of M. Hirsch.

In the following summer I addressed to M. Plantamour a paper upon the subject, to be submitted to the next meeting of the Geodetical Congress. In this note, which is reprinted at the end of the present report, I first give a mathematical analysis of the problem. I next show experimentally that the motion of the knife-edge support is not a translation, but is a rotation, so that different parts of the head of the tripod, only a few centimeters distant from one another, move through very different distances. Consequently, measures of the flexure made anywhere except at the center of the knife-edge plane require an important correction before they can be used to correct the periods. This is confirmed by experiments with a mirror while the pendulum is in motiou. I next give a brief résumé of my statical measures of the flexure. I then give measures of the actual flexure under the oscillation of the pendulum, and show that the statical and dynamical flexibilities are approximately equal. Finally, I swing the same pendulum upon the Repsold support and upon another having seven times the rigidity of that one, and I show that the difference of the periods of oscillation agrees with the theory.

Immediately upon the reception of my manuscript, MM. Hirsch and Plantamour commenced new researches, designed to form an "étude approfondie de ce phénomène." These were embodied in a paper by M. Plantamour, which was read to the Geodetical Congress, and which has since been

expanded into a memoir entitled "Recherches expérimentales sur le mouvement simultané d'un pendule et de ses supports." M. Plantamour finds fault with me, first, for having measured the flexure with a force five or ten times that of the deflecting force of the pendulum; and second, for measuring the elasticity statically instead of dynamically. The reply to the first objection is that the properties of metals are known to a great extent, that elasticity is not "une force capricieuse," and that no fact is better established than that an elastic strain is proportional to the stress up to near the limit of elasticity, which limit was not approached in the author's experiments. As to the second objection, I had shown by experiment that the statical and dynamical flexures are nearly equal; and I am willing to leave it to time to show whether this will not be assumed in future measures of the flexure of future pendulum supports. M. Plantamour caused a fine point fixed into the head of the tripod to press against a little mirror, mounted on an axis; and then observed the reflection of a scale in a telescope. The length of the path of light from the scale to the telescope divided by the distance of the bearing point from the axis of the mirror he calls the grossissement; so that had he used a fixed star in place of his scale, the grossissement would have been virtually infinite. From the given length of the lever it would appear that a movement of 04.03 in the point would turn the mirror 4". The aperture of the mirror is not stated, but it cannot be supposed that the error of observation would be less than this. It does not seem to me that the use of this mode of measurement, which magnifies the motion but little more than my method, is conducive to accuracy, especially in investigating the difference between statical and dynamical flexure. A certain finite force presses together the point and the lever. Dividing this force by the minute area of pressure, we find the pressure upon the metal is very great, approaching the crushing pressure. Now, the behavior of metals under great pressure is greatly influenced by the time. But my objection is not merely theoretical; I have myself made experiments upon this method, and, making them as skillfully as I could, I still found great uncertainty in the results.

The following table exhibits M. Plantamour's results:

M. Plantamour's flexure experiments.

	Flexure under swinging pen- dulum.	Flexure when weight is raised and lowered.	Statical flex- ure.
Support on floor, comparator removed	$\mu$ 3. 26 $\pm$ . 05	μ 3. 17 ± . 09	μ 3. 27 ± . 04
On Geneva pier, comparator removed		3. 29 ± .08	3. 48 ± . 04
On Geneva pier, comparator in place		2.50 ± .05	2.76 + .04
On Berlin pier, comparator in place		2.90 ± .04	3. 24 ± .03
On wooden table, comparator in place		3. 26 ± .04	3. 67 ± .02
On wooden table, comparator removed	4.42 4. 13	4. 53 ± . 04	4. 98 ± . 05
Excess:		2.00	4. DC = . DD
Geneva pier over floor	09 ± .06	+ .12 ± .12	+ .21 ± .06
Berlin over Geneva pier	+ .10 ± .08	+ 40 ± .06	
Table over Geneva pier, comparator in place.	$+ .78 \pm .07$	+ .76 ± .06	
Table over Geneva pier, comparator removed	+1.16 ± .14	$+1.36 \pm .10$	
Effect of comparator:			,
Geneva pier	76 ÷ 07	_ 79 ± 40	72 ± .06
Table	-1.23 + 14	$-1.27 \pm .06$	$-1.31 \pm .05$
Excess table over pier			- 1.51 ± .05 59 ± .08

The table used is the same one shown in Fig. 26 of the Coast Survey Report for 1877. The numbers in the last line above should show the effect of the weight of 3 kilogrammes in diminish ing the flexure of this table under a horizontal force of 100 grammes. The weights used in obtaining the first two numbers were about 100 grammes; but the last column is one-tenth the deflection produced by 1,000 grammes. It seems quite incredible that 3 kilogrammes, laid on the table, should really have an effect of this magnitude, so closely proportionate, too, to the deflecting force. It is highly desirable that this result should be confirmed by purely optical experiments; and until this

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is done, we must suspect that these large numbers indicate some error to which the method of observation is liable. It is certain that the comparator did not act as a brace to stiffen the instrument, and equally so that its weight is not sufficient to alter the modules of elasticity of the brass of the support. It would seem, however, that the effect might be due to a film of some semi-elastic substance under the feet of the tripod. When the tripod is on the floor, no such effect is observed; when it rests on the Geneva pier the dynamical flexure is the same as when it is on the floor, but the statical flexure is much larger. On the Berlin pier the excess of the statical flexure over that on the Geneva pier is five times the dynamical excess. On the other hand, the excess of the dynamical flexure on the table over that at Berlin is half as great again as the statical excess.

#### MEASURES OF FLEXURE.

My own measures form two series, those made previous to, and those made subsequent to the publication of M. Plantamour's memoir.

In the first series, I was simply occupied in measuring the flexure of the Repsold tripod, as well when properly put up as when the nuts of the bolts were not tightened, of the Geneva support as mounted at Hoboken, and of my "stiffest" support. All the precise measures are statical, and, being made with a filar micrometer, are superior in accuracy to the subsequent ones.

In the second series, the flexures are always measured dynamically as well as statically, and the statical flexure is always found to be the greatest. On the excessively flexible Repsold tripod the difference is sufficient to affect the length of the second's pendulum by 10<sup> $\mu$ </sup>. Nevertheless, as the axis of motion is different for the two kinds of flexure, there are points at which the motion is less for dynamical than for statical flexure. And in point of fact, when the Geneva support rests on the Geneva tripod, the dynamical flexure of the center of the knife-edge is greater than the statical flexure.

Experiments were also made upon the effect of leaving the nuts of the Repsold tripod entirely loose, of tightening them as much as possible by the hand, and of tightening them by a wrench. It is found that there is little difference between leaving them loose and tightening them by hand, but the effect of the wrench is to produce a stiffening equivalent to a shortening of the pendulum by 20 microns.

Experiments were also made upon the effect of placing a weight of 6 pounds, and afterwards of 25 pounds, upon the head of the Repsold support. The first weight produced absolutely no effect; the second moved the axis of motion a little, and thus caused a slight difference of flexure at some points.

Experiments were also made upon the effect of resting the Repsold support upon blotting-paper, upon blocks of oak, and upon blocks of india-rubber. In every case the difference between the statical and dynamical flexure was much increased.

The pendulum has also been swung on all these different supports and the period of oscillation determined with a view of ascertaining whether the statical or dynamical flexure should be used in calculating the corrections to the periods. The result, as might have been predicted from the mathematical theory, shows that a value intermediate between the two is to be taken. But the best way is to make the support so solid that the difference of the two kinds of flexure must be inconsiderable.

## EXPERIMENTS TO DETERMINE THE FLEXURE-CORRECTION.

## A .- Flexure of the Repsold stand.

To determine the flexure, a known force was applied statically to the stand, and the resulting deflection was measured. The principal experiments were made in the cellar of the Stevens Institute at Hoboken. The floor of the cellar is of brick laid down in cement directly on the solid ledge. The floor having been carefully cleaned, the three brass pieces which support the screwfeet of the Repsold tripod were laid down upon it, and the tripod itself was set up. The binding-screws of the feet were screwed up very tight. The pendulum, comparator, and meter were not placed on the tripod, but a mass of iron about equal to them in weight was placed on blocks on the lower part of the tripod in order to ballast it. To apply the force, a silken cord was wound round

the tongue upon which the pendulum usually rests, just in the slot over which is the middle of the knife-edge, in such a manner that the cord when stretched horizontally was exactly at the level of the knife-edge. The cord passed horizontally and perpendicular to the knife-edge to a pulley-wheel over which it passed, and from which it hung down vertically; and to its extremity was attached a kilogramme. The pulley-wheel was one which belonged to an Atwood's machine; it turned with very little friction and its rim was accurately plane and perpendicular to the axis. This wheel rested on a stout wooden tripod; its axis was carefully adjusted to be parallel to the knife-edge and the upper part of the rim was brought to the level of the knife-edge. The usual position of the knife-edge is here referred to; but the pendulum was not actually in position. In the measurements of flexure, one person gently raised and lowered this weight alternately. The measurement of the deflection was made by another person, as follows: A micrometer scale on glass was fixed, either to the tongue or to an arm solidly fixed to the tongue, in such a way that the direction of measurement was parallel to the force applied to the tripod. This micrometer scale was observed by a microscope magnifying about fifty diameters and provided with a filar micrometer. microscope was mounted on a separate, very stiff, iron stand resting on the floor, and carrying at its head a brass apparatus for holding the microscope. The optical axis of the microscope was made exactly parallel to the knife-edge and the filar micrometer screw was made parallel to the force applied to the stand, and the microscope was focused on the micrometer scale. Each division of the scale usually employed was about 12". The filar micrometer wire (which was vertical) was made to bisect one division of the scale and the micrometer was read; it was then made to bisect another division, by turning the screw through about one revolution, and the micrometer was read again. Thus, the value of the revolution was obtained. The weight was then put on, and pointings were made upon the same two divisions. Then, the whole process was repeated until the weight had been put on five times. This made one set of experiments.

The following experiments were made to determine the position of the axis of rotation of the knife-edge support during flexure.

Hoboken, March 10, 1877. Ther. 13° C.—The micrometer scale, attached to an arm, was placed on the line of the knife-edge  $53^{\rm mm}$  in front of the anterior extremity of the tongue. The following were the readings of the filar micrometer on one of the lines of the scale with the weight alternately on and off ( $\rho$  throughout signifies a revolution of the micrometer screw):

Weight off,	Weight on
ρ 10. 077	$\rho$
10. 955	11. 324
. 968	. 320
. 978	. 324
M 10 005	14.000
Means 10, 967	11.323

Difference,  $+0^{\rho}.356$ .

The arm was now lengthened so that the scale was 318<sup>mm</sup> in front of the end of the tongue. The following readings of the filar micrometer were now made:

Weight off.	Weight on.
$\rho$	ho
10. 344	10. 762
. 350	. 776
, 341	. 793
. 335	. 778
. 330	. 772
W Common Company of the Common	
Means 10, 340	10. 776

Difference, +0p.436.

The micrometer-scale was next carried over to the other side of the instrument so as to be  $496^{mm}$  behind the front end of the tongue. The following readings were now made:

Weight off.	Weight on.
ρ	ρ
10. 106	10. 324
. 120	. 334
. 141	. 334
.~124	. 346
. 136	. 340
Means $10.125$	10, 336

Difference,  $+0^{\circ}.211$ .

It will be understood that in all these experiments the arm to which the scale was fixed was attached to the tongue on which the pendulum rests, and that this arm was subjected to no force.

The above results are satisfied by supposing that the axis of rotation cuts the level of the knife-edge 1<sup>m</sup>.258 behind the end of the tongue. The following table shows the agreement of the observations with this supposition.

Distance forward of	Flexure.	
end of tongue.	Obs.	Calc.
m	ρ	ρ
+0.318	0.436	0.433
+0.053	0.356	0.361
<b>-0.496</b>	0. 211	0, 212

The scale was next (March 12, 1877, observer, Edwin Smith) fixed at 395<sup>mm</sup> vertically below the end of the tongue. The following measures were then made:

Weight off.	Weight on
ho	$\mu$
13. 739	13. 260
. 700	. 247
. 710	. 261
. 700	. 260
. 702	. 243
. 710	
Means 13, 710	13, 254

Flexure,  $+0^{\rho}$ .446.

The filar micrometer was here in the reverse position from its usual one, and hence the reading with weight off is greater than with weight on.

The scale was next placed  $44^{\rm cm}$  above the point of support and the following measures were made:

Weight off.	Weight on.	
ρ	ρ	
<b>10. 52</b> 3	10. 737	
<b>. 45</b> 3	. 645	
. 400	. 578	
10. 450	10.650	
10. 459	10. 653	

Deflection,  $-0^{\rho}$ .196.

The filar micrometer was so shaky in this position that accurate measures could not be obtained, but the above answers the purpose.

The scale was next fixed on the end of the tongue and the three measures given below (series 18, 19, 20) were made. The mean of these gives a flexure of 00.340. These measures show that the axis of rotation cuts a vertical from the end of the tongue at a height of 1.07 meters above the level of the knife-edge. Thus we have on this hypothesis:

Distance below	Flexure.			
knife-edge.	Obs.	Cale.		
m	$\rho$	$\rho$		
-0.44	0.196	0.196		
0.00	0.340	0.332		
+0.395	0.446	0.452		

A large series of experiments were made at Hoboken to determine the amount of flexure. Of these, the following are chiefly relied upon:

HOBOKEN, March 7, 1877. Ther., 590.15 F. 3h 12m P. M.

Möller's glass scale of hundredths of millimeters was fixed 3 millimeters above the end of the tongue. The filar micrometer wires remained fixed, and readings of the micrometer scale were made on the two wires, alternately with weight off and on.

	FIRST SERIES.			
	Wei	ght off.	Wei	ght on.
	$844^{\mu}$	$893^{\mu}$	$878^{\mu}$	931
	843	894	879	930
	844	894	879	931
	845	895	879	930
	844	896	879	931
Means,	844.0	894. 2	878.8	930. 4
Distance of v	wires, 50. 2		51	. 6
Flexure,	3	$\overset{\mu}{4.4}$	36.2	
Mean,		3 <b>5</b> . 3		

The following readings were then taken with the filar micrometer (temperature 59°.24 F.). The wire was set between lines 80 and 81, and between lines 90 and 91 of the scale.

•	SECOND SERIES.				
7	Weight off.	Weight on.			
90-91	80-81	90-91	80-81		
9.347	10. 312	9. 694	10. 655		
344	. 309	. 693	. 660		
348	. 315	. 692	. 664		
348	. 322	. 699	. 663		
352	. 336	. 713	. 684		
Means, 9. 344	10. 319	9, 698	10. 665		
1 millimeter,	975	0.5	967		
Flexure,	0.354	0.346			
Mean,	0.550 = 36.1				
was considered of in	forior goonnoor	÷			

This last set was considered of inferior accuracy.

HOBOKEN, March 10, 1877. 0h 15m P. M. Temp., 110.9 C.

A scale on glass by Rogers was observed in the same position as above. Each division is  $\frac{1}{2000}$  of an inch (=127\*). The micrometer wire was placed between the first and second and between the tenth and eleventh lines. The observations were made alternately with the weight off and on.

## THIRD SERIES.

	Weight off.		Weight on.		
	1-2	10-11	1-2	10-11	
	9. 715	10. 849	10. 060	$11.\stackrel{ ho}{193}$	,
	. 720	.855	. 058	. 185	
	. 723	. 845	. 049	. 186	
	. 715	. 846	.055	. 185	
	. 714	. 841	. 050	. 176	
Means,	9. 719	10. 847	10, 054	11. 185	
$\frac{9}{2000}$ inch,	<b>1.</b> 1	28	1. 13	$31 \qquad \therefore \frac{1}{10}$	mm.=0.988
Flexure,		$0.\overset{0}{3}35$	0.338		
Mean,		0.3	$36=3\overset{\mu}{4}.1$		

This series occupied seven minutes. The whole apparatus was readjusted and a new set was made, as follows:

## FOURTH SERIES.

	Weight off.		Weight on.		
	1-2	10-11	1-2	10-11	
	$9.\overset{\circ}{1}17$	10. 239	9. $^{ ho}_{459}$	10. $^{ ho}_{580}$	
	.122	. 236	. 449	.586	
	.~125	. 241	. 464	. 581	
	.120	. 244	. 456	.584	
	. 128	. 234	. 456	. 579	
Means,	9. 122	10. 239	9. 457	10. 582	
2000 inch,	1. i	17	1. í	25	$_{10}^{1}$ mm.=0. $982$
Flexure,		0.335	0. 343		•
Mean,		0.	339 = 34.5		

At  $2^{\rm h}$  55<sup>m</sup> P. M. another set of experiments were made, giving the following results (temperature, 12°.2 °C.):

## FIFTH SERIES.

	Weight off.		Weight on.		
	1-2	10-11	1-2	10-11	•
	9. <b>64</b> 1	10. <b>74</b> 5	9. 980	11. $082$	
	. 619	. 748	, 968	. 084	
	. 612	. 745	. 962	. 075	
	. 616	. 735	. 963	. 089	
	. 626	. 754	. 976	. 104	
Means,	9. 623	10. 745	9, 970	11. 087	
$\frac{9}{2^{\frac{9}{00}}}$ -inch,	$1.\mathring{1}22$		1. 1		$\frac{1}{10}$ mm.=0. 980
Flexure,		0. 347	0. 342	,	
Mean,			0.344 = 35.1		

After this set the focus was readjusted and two more sets were taken, as follows (temperature,  $12^{\circ}.2$ ):

## SIXTH SERIES.

	Weight off.		Wei		
	1-2	10-11	1-2	10–11	
	9, 600	10. 730	$9.^{'}953$	11.083	
	. 602	.742	. 956	. 080	
	.605	. 736	.945	. 075	
	. 594	. 740	. 953	. 076	
	. 602	. 734	. 951	. 071	
Means	s, 9.601	10. 736	9. 952	11. 077	
20°00-inch,	1. i	35	1. 1	25	$\frac{1}{10}$ mm.=0. $\frac{6}{989}$
Flexure,		$0. extstyle{351}$	0.341		
Mean,		. 0.	346 = 35.0		
		SPVPNT	H SERIES		

## SEVENTH SERIES.

## Temp., 13° C.

			,		
	Weig	ht off.	Wei	ght ou.	
	1-2.	10-11.	1-2.	10-11.	
	ρ	ρ	ρ	ρ	
	9.582	19. 711	9. 929	11.046	
	.575	. 706	. 921	. 040	
	.570	. 703	. 922	.042	
	$.\ 575$	. 700	. 918	. 038	
	. 561	. 697	. 912	. 033	
Means,	9. 573	10. 703	9.920	11.040	
	_	ρ	•	ρ	ρ
$\frac{9}{2000}$ inch,	1.	130	1.	120	$\therefore \frac{1}{10} \text{ mm.} = 0.984$
731		ρ	ρ		
Flexure,		0.347	0.337		
Mean,			342=34.8		•
mean,		0.	342=34.8		

Three sets were then taken, placing the micrometer wire between the second and third lines, instead of the first and second. The light had now become fainter. Temp., 13°.1 C.

## EIGHTH SERIES.

	Weig	ht off.	Wei	ght on.	
	2-3.	10-11.	2-3,	10-11.	
	9. $710$	10. 724	10. 061	11. 061	
	. 712	.721	. 057	. 059	
	. 719	. 711	.052	. 053	
	. 713	. 721	. 052	. 058	
	. 725	. 727	. 060	. 055	
Means,	9. 716	$\overline{10.721}$	10.056	11. 057	
2000 inch,	1.0	) 005	1. (	)01	$\therefore \frac{1}{10} \text{ mm.} = 0.987$
Flexure,		$0.\overset{\rho}{340}$	0. 3 <sup>6</sup>		
Mean,		0.	338=34.2		

After this set the focus was changed. Thermometer still 13°.1 C.

		NINTI	I SERIES.		
	Weig	ht off.	, Wei	ght on.	
	2-3.	10-11.	2-3.	10-11.	
	9. $669$	10. 677	$10.\overset{ ho}{0}27$	11. 022	
	. 696	. 696	. 030	. 035	
	. 696	. 693	. 039	.045	
	.694	. 695	. 032	. 038	
	.685	. 693	. 032	. 035	
Means,	9. 688	10. 691	10.032	11, 035	
$\frac{S}{2000}$ inch,	1. (	003	1. (	ρ <b>00</b> 3	$\therefore \frac{1}{16}$ mm.=0. 987
Flexure,		0. 344	$0. \overset{\rho}{344}$		
Mean,		0.	344 = 34.9		

It was noted that this set ought to have double weight. The following set was then taken temperature, 13°.1 C.:

TENTH SERIES.

	Weig	ht off.	Wei	ght on.	•
	2-3.	10-11.	2-3.	10-11.	
	9, 688	10. <b>6</b> 89	10,019	11. 029	
	.675	. 674	, $025$	. 031	**
	.674	. 676	. 014	. 020	
	. 670	. 675	. 014	. 020	
	. 662	. 660	. 016	. 016	
Means,	9. 674	10. 675	10. 018	11. <b>0</b> 23	
- 8 inch,	1. (	001	1. (	) 005	$\therefore \frac{1}{10} \text{ mm.} = 0.987$
Flexure,		0.344	0. 3 <b>4</b> 8		
Mean,	*	0.	346 = 35.1		

This set was also assigned double weight at the time.

Collecting the foregoing results, we have for the deflection of the end of the tongue under one kilogramme's weight—

	Diff. from mean.
$\mu$	μ
1st set, March 7, 1877 35.3	+0.4
¬2nd set, March 7, 1877 36.1	+1.2
3d set, March 10, 1877 34.1	-0.8
4th set, March 10, 1877 34.5	-0.4
5th set, March 10, 1877 35.1	+0.2
6th set, March 10, 1877 35.0	+0.1
7th set, March 10, 1877 34.8	-0.1
Sth set, March 10, 1877 34.2	-0.7
9th set, March 10, 1877 34.9	+0.1
10th set, March 10, 1877 35.1	+0.2
Mean 34.9	$\pm 0.1$

The middle of the knife-edge being  $30^{\rm mm}$  behind the end of the tongue, which is  $1^{\rm m}.258$  forward of the point where the axis of rotation crosses the knife-edge produced, it follows that  $\frac{3.0}{1.258}$  of the flexure observed at the end of the tongue, or  $0^{\mu}.8$ , has to be subtracted from that quantity to get the flexure of the middle of the edge. The latter is, therefore,  $34^{\mu}.1$ .

Measures of the flexure were also made on the 8th and 12th of March, by Sub-assistant Edwin Smith. The following are his results:

## ELEVENTH SERIES.

		1 <sup>b</sup> 15 <sup>m</sup> p. m.	Temp., 60°.41	F.	
	$\mathbf{Wei}_{!}$	ght off.	Weight on.		
	2-3.	7-9.	2-3,	7-9,	
	6. 970 . 956 . 963	7. 599 . 571 . 581	7.305 .298 .295	7. 940 . 940 . 930	
	. 950	. 573	. 281	. 915	
Means,	6, 960	7. 581	$\overline{7.295}$	7. 931	
2000 inch,	0.	621	0,	636	
Flexure,		0. 335	0. 350		
Mean,	0.342 = 34.5				

## TWELFTH SERIES.

1<sup>h</sup> 45<sup>m</sup> p. m. Temp., 60°.37 F.

	Weigl	at off.	Weight on.		
	2-3.	10-11.	2-3.	10-11.	
	<b>P</b>	P	_P	ρ	
	6. 921	7. 930	7.245	8. 256	
	. 911	$.\ 915$	. 248	. 249 🍍	
	. 915	. 917	. 248	. 253	
	. 906	. 913	. 248	<b>. 24</b> 8	
	. 902	. 907	.247	. 240	
Means,	6. 911	$\overline{7.916}$	$\overline{7.247}$	8. 249	
8 inch,	1. 0	05	1. 0	02	
Flexure,		0.336	0. 333		
Mean,		0.	334=33. 9		

## THIRTEENTH SERIES.

2<sup>h</sup> 05<sup>m</sup> p. m. Temp., 60°.52 F.

	Weig	Weight off.		ght on.
	2-3.	10-11.	2-3.	10-11.
	ρ	ρ	ρ	ρ
	<b>6. 94</b> 3	7. 949	7.271	8.282
	. 945	. 946	. 279	. 278
	. 941	. 941	. 271	. 275
	. 932	. 934	. 270	. 273
	<b>. 93</b> 8	. 939	. 271	. 273
Mean	s, 6. 940	7, 942	$\overline{7.272}$	8.276
2000	inch, 1.0	1.002		04
Flexu	ıre,	0, 332	0. 334	
Mean	,	0.	333=33.7	
S. Ex. 49——47				

## FOURTEENTH SERIES.

2<sup>h</sup> 25<sup>m</sup> p. m. Temp., 60°.27 F.

	Weigl	nt off.	Weight on.		
	2-3.	10-11.	2-3.	10-11.	
	ρ	ρ	р	٩	
	6. 942	7.943	<b>7.</b> 280	8, 288	
	. 939	. 945	. 277	.287	
	. 942	. 949	. 281	. 281	
	. 944	. 950	. 279	. 280	
	. 940	. 941	.279	. 283	
Means,	6, 941	7. 946	7. 279	8. 284	
and inch,	1. 0	05	1.0	05	
Flexure,		<b>0.</b> 338	0. 338		
riexure,		v. 306	V. 556		
Mean,		0.	338 = 34.2		

#### FIFTEENTH SERIES.

2<sup>h</sup> 40<sup>m</sup> p. m. Temp., 60°.23 F.

	Weigl	at off.	Weight on.		
	.1-2.	9-10.	1-2.	9-10.	
	ρ	ρ	ρ	P	
	6.825	7.825	7. 159	8. 159	
	。. 823	. 826	. 159	. 161	
	.822	. 829	. 158	. 158	
	. 819	. 820	. 157	. 160	
	. 821	. 823	. 157	. 160	
Means,	6.822	7.825	7. 158	8. 160	
0 31	1 0	·/ <b>&gt;0</b>	P	200	
2000 inch,	1. 0	03	1. (	002	
Flexure,		0.336	0. 335		
Mean,	0.336=34.0				

## SIXTEENTH SERIES.

 $2^{h}$  55<sup>m</sup> p. m. Temp., 60°.18.

	Weigl	at off.	Weight on.		
	1-2.	9-10.	1-2.	9-10.	
	P	P	P	ρ	
	6.822	7.822	7, 153	8. 160	
	. 824	. 826	. 158	. 156	
	. 826	. 827	. 157	. 158	
	. 821	. 823	. 160	. 161	
	. 820	. 823	. 161	. 161	
Means,	6. 823	7.824	7. 158	8. 159	
8 inch,	1. 001		1.001		
Flexure,		0. 335	0. 335		
Mean,		0.	335=34.0		

## SEVENTEENTH SERIES.

3և 10 <sup>m</sup> թ.	m.	Тешр.,	$60^{\circ}.04$	F
-----------------------	----	--------	-----------------	---

Weight off.		Weight on.		
1-2	9-10	1-2	9-10	
р 0.050	P 047	, , , , , , , , , , , , , , , , , , ,	۹	
		7.172	8. 175	
. 849	. 8 <b>4</b> 8	. 179	. 183	
. 849	.852	. 186	. 183	
. 849	. 851	. 189	. 190	
. 850	. 850	.180	. 181	
6. 849	7. 850	7. 181	8. 182	
1.	001	1. 601		
	<b>0.</b> 332	0. 332		
$0.\overset{\circ}{332}=33.\overset{"}{7}$				
1877, MARCH 12.				
	1-2 6. 850 . 849 . 849 . 849 . 850 6. 849	1-2 9-10 6. 850 7. 847 . 849 . 848 . 849 . 851 . 850 . 850 6. 849 7. 850 1. 001 0. 332 0. 332	$ \begin{array}{cccccccccccccccccccccccccccccccccccc$	

# Ther., 14°.1 C.

	Weig	ght off.	Weight on.		
	12-13	20-21	12-13	20-21	
	ρ	ρ	ρ	ρ	
	8. <b>43</b> 2	9. 452	8. 807	9.805	
	. <b>45</b> 3	. 467	. 807	. 805	
	. 448	. 457	. 797	. 801	
	.445	. 462	.792	. 800	
	. 440	. 450	. 780	. 779	
Means,	8. 444	9. <b>45</b> 8	8. 797	9. 798	
<sup>8</sup> / <sub>2000</sub> inch, 1.014		1.	<b>0</b> 01		
Flexure,		0. 353	0. 340		
		_			

# Mean, 0. After this the apparatus was readjusted.

## NINETEENTH SERIES.

Ther., 14°.2 C.

		,				
	Weig	ght off.	Weig	Weight on.		
	12-13	20-21	12-13	20-21		
	ρ	P	ρ	ρ		
	9.421	10. 431	9. 755	<b>10.758</b>		
	. 419	. 435	. 753	. 750		
	.421	. 428	. 763	. 770		
	. 428	. 428	. 757	. 770		
	. 419	. 423	. 763	. 763		
Means,	9. 422	10. 429	9. 758	10. 762		
2000 inch	h, 1. 007		1. 004			
Flexure,		0. 336	0. 333			
Mean		0. 334	=34.0			

#### TWENTIETH SERIES.

Ther., 14°.2 C.

	Wei	ght off.	Weight on.		
	12-13		12-13	20-21	
	9. 383	10. 390	9. 726	10. 744	
	. 400	. 402	. 730	. 738	
	. 395	. 399	. 728	. 725	
	. 396	. 405	. 735	. 743	
	. 398	. 405	. 735	. 730	
Means,	9. 394	10, 400	9. 731	10, 736	
8 inch,	, 1. 006		1.	005	
Flexure,		o. 337	0. 336		
Mean,		0. 337:	=34. 1		

During the last two sets the illumination was very poor.

Mr. Smith's results, being collected, are as follows:

, 0		Flexure.	Difference from the mean.
		μ	μ
11th set, 1877, March	8	34.5	+0.4
12th set, 1877, March	8	33. 9	<b>-0.</b> 2
13th set, 1877, March	8	33. 7	-0.4
14th set, 1877, March	8	34. 2	+0.1
15th set, 1877, March	8	34.0	-0.1
16th set, 1877, March	8	34. 0	-0.1
17th set, 1877, March	8	33. 7	-0.4
18th set, 1877, March	12	34. 9	+0.8
19th set, 1877, March	12	34.0	-0.1
20th set, 1877, March	12	34.1	$\pm 0.0$
•		<del></del> _	
Mean		34. 1	$\pm 0.1$

It will be seen that there is a rather large difference between the results of the two observers. It will, of course, be understood that the discordances of single readings are due mainly to relative movements of the micrometer and the pendulum-support. As most of the sources of constant error tend to make the observed values too small, the larger result has been preferred. When the binding screws of the feet were not perfectly tight the flexure was still greater, as is shown by the following means of sets of observations made under those circumstances:

	Flexure.	Diffence from the mean.
	μ	· µ
21st set, 1877, February 17	36. 5	-0.4
22d set, 1877, February 17	37.9	+1.0
23d set, 1877, February 17		<b>-0.</b> 8
24th set, 1877, February 17	37.8	+0.9
25th set, 1877, February 17		0.0
26th set, 1877, February 19		-1.6
27th set, 1877, February 19		+0.7
, , ,		
Mean	36.9	$\pm 0.3$

The flexure of the Repsold stand was also measured in Geneva, Paris, and Berlin. In Berlin the microscope was mounted on a wooden stand, which rested on the same pier as the tripod. This was to avoid including the flexure of the pier, which is best measured separately. The micrometer scale was simply fixed to a piece of wood, which was laid on the brass pieces at the ends of the

tongue. This wooden piece projected 35<sup>mm</sup> beyond the tongue, and consequently 1 .7 has to be subtracted from the observed results to get the flexure at the middle of the knife-edge. The following are the means of sets of ten measures:

	Flexure.	Difference from mean.
	$\mu$	$\mu$
1876, May 24, a. m	. 35.8	+0.1
1876, May 24, p. m	35.7	0.0
1876, May 24, p. m		+0.1
1876, May 24, p. m		+0.2
1876, May 25, a. m		-0.2
1876, May 25, a. m		<b>-0.</b> 3
Mean	35. 7	$\pm 0.1$

This result agrees well with that obtained at Hoboken. Thus:

Flexure of mid- dle of knife- edge under 1 kilogramme	
$\mu$	μ
Hoboken (C. S. P., observer) 34, 1	215.2
Berlin 34.0	214. 5

The same value was found in a rough measure made at Geneva, September 13, 1875.

Much larger values were obtained in Paris, which agree with those found at Hoboken when the binding-screws were not tight. Thus we have

	Flexure with binding-screws loose.		
Hoboken	• • • • • • • • • • • • • • • • • • • •	36.1	227.8
Paris, 1876, January 18	· - • · • • · · ·	36.3	229. 1
Paris, 1876, March 7			234.1

B.—Flexure of the stiffest support.

This support was constructed in order to test the calculated effect of the flexure of the stand. The following table shows the results of measures of its flexure:

Distance of scale forward of center of knife.	Distance below knife.	of scale level of	Deflection + in the direction of the force, - in the opposite direction.
mm.	m	m.	μ
+ 30	_	33	+ 5.2
+ 30		33	+ 5.2
+ 30	+1	003	-42.5
+335	+1	003	-36.7

It follows from this that the axis of rotation cuts the line of the knife-edge 166<sup>mm</sup> behind the center of the edge, and cuts the vertical from that center 68<sup>mm</sup> below the edge. Also, that the deflection of the middle of the edge under a force of 1 kilogramme's weight is 3<sup>m</sup>.1. This includes the flexure of the pier.

## C .- Flexure of the Geneva support and pier.

In measuring this flexure, instead of a low-power microscope and filar micrometer a high-power microscope and eye-piece micrometer were used. A stage micrometer was always observed, and the value of the divisions of the eye-piece micrometer determined by it. In the following experiments the scale was 254<sup>mm</sup> above the level of the knife-edge and 25<sup>mm</sup> forward of the middle.

## 1878, OCTOBER 1.

#### FIRST SET.

	Weight off.	Weight on
	10. 4	5.8
	10. 2	5. 6
	10.3	5. 4
	<b>10.</b> 2	5. 5
	10. 0	5. 4
Means,	10. 2	5. 5
Flexure,		4. 7

It was observed that 18.5 of eye-piece micrometer equals 9 of stage micrometer. 1 division of latter =  $7^{\mu}.34$ . . . observed flexure =  $16^{\mu}.8$ .

#### SECOND SET.

## (Higher power.)

	Weig	ht off.		Weig	ht on.	
	12.4	39. 2	1	l8.7	<b>45.</b> 8	
	12. 3	39.2	1	19.0	<b>4</b> 5. 9	
	12.3	39. 3	1	8.8	46.0	· -
	12.4	39.3	1	18.8	46. 1	
	12. 2	39. 4	1	18.7	<b>46.</b> 2	
Means,	$\overline{12.3}$	39. 3	1	8.8	46.0	
66	$\mu.1 = 27$	. 0	• .	27	. 2 1	$div. = 2^{\mu}.44$
Flexure,		6.	5 6.7			
Mean,			$6.6 = 16^{\mu}$	.1		

#### THIRD SET.

	Weight off.	Weight on.
	39.8	46. 4
	<b>39.</b> 8	46. 5
	39. 9	<b>46.</b> 5
	40. 0	<b>46.</b> 6
	40. 1	46. 7
Means,	<b>39.</b> 9	46. 5
Flexure,	6.	6

Nine spaces of stage micrometer were equal to 27.1 of eye-piece micrometer. Hence, observed flexure  $=16^{\mu}.1$ .

## FOURTH SET.

	Weight off.	Weight on.
	39. 9	46. 4
	40.0	46.6
	40. 1	46.6
	40. 0	46.8
	40. 2	46.8
Means,	40.0	46.6
Flexure,	6.	$6 = 16^{\mu}.1$

## 1878, OCTOBER 19.

The scale was fixed 111cm below the knife-edge, and three sets of 10 gave for the deflection

$$\begin{array}{r}
-48.8 \\
-47.5 \\
-48.1 \\
\hline
-48.1
\end{array}$$
Mean,  $-48.1$ 

#### 1878, OCTOBER 21.

The scale was fixed 244 millimeters above the knife-edge, and 356 millimeters forward of the middle. Two sets gave as the deflection

$$+13.5$$
 $+12.2$ 
 $+12.8$ 

From these measures we find the flexure at the middle of the knife-edge to be 4#.05.

## EXPERIMENTS AT PENNSYLVANIA GRAVITATION STATIONS.

#### ALLEGHENY.

Statical flexure of Geneva support on iron bars. Weight and pulley employed. Weight= $2^k$ ;  $f_a$  denotes flexure produced by a horizontal force equal to the weight of pendulum ( $6^k$ .308).

## 1879, FEBRUARY 18.

Scale 3-inch (=2cm) above, and 12.5 inches (=32cm) forward of middle of knife-edge; 22.4 div. of scale=100r. C. S. P., observer.

Onel.	readinas
1111111	T CHUTHUS

Wt. off.	Wt. on.	Diff.	$f_{ m o}$
• • • •		1.2	
		1.2	15.1
• • • •	· · · ·	1.1	
		div.	
		1.17	

The following measures were made under a higher power of microscope; 58 div. of scale = 100\*.

37. 0 36. 7	34. 0	2, 85	16
35. 7 36. 0	33. 1 33. 4	2.6	15
37. 7 37. 9 38. 0	35. 0 35. 0 35. 2	3. 0	16
38. 6	5.22 <b>. 2</b>		

Mean,

16

In the following measures, 37 div. of scale=100"; otherwise same as preceding.

24.1 22.2 1.9 16 4

## 1879, FEBRUARY 20.

Scale  $\frac{1}{2}$  inch (=1<sup>em</sup>) above, and 12 $\frac{1}{2}$  inches (=32<sup>em</sup>) behind middle of knife-edge; 23.5 div. ot scale=100\*.

Soule	readings.
Boute	remaine.

Wt. off.	Wt. ou.	Diff.	$f_n$ .
6.3	11. 4	div.	μ
6, 5	11. 2	4.7	63
6. 6	11.2		
7. 0			

In the following, 28.3 div. of scale=100"; otherwise the same.

35, 4	40. 5		
35. 6	40. 4	4.8	54
35. 7	40. 4		
35. 7			
63. <b>4</b>	68. 4		
63. 6	68. 9	5, 0	56
<b>64.</b> 0	68. 9		
<b>64.</b> 0			

Mean,

55

In the following, 41.1 div. of scale=100"; otherwise same.

20.8	25. 0	4. 4 rej.	
30, 5	35, 0		
71.5	78. 0	·	
71.6	<b>78.</b> 1	6. 5	64
71. 6	78.0		
71. 4	78, 8		

## 1879, MARCH 4.

Scale 1 inch (=2<sup>cm</sup>.5) above, and 13½ inches (34<sup>cm</sup>.5) behind middle of knife-edge; 26.7 div. of scale=100\*. H. Farquhar, observer.

5, 4 64

Scale next put 2em above and 14 inches (=35em) forward of middle of knife-edge; 26.7 div. of scale=100e.

1.0 12

## 1879, MARCH 6.

Scale on level of knife-edge and 15 inches (=38cm) forward; 38.5 div. of scale=100\*.

1.9 16

Scale next put 55 inches (=140cm) below middle of knife-edge; 33.3 div. of scale=200\mu.

The following is a summary of the above. Fo here and elsewhere denotes the flexure at the middle of the knife-edge under a horizontal force equal to the weight of the pendulum. A=dis-

tance from middle point of knife-edge to its intersection with axis; B=distance from middle point of knife-edge to the intersection of axis with vertical line.

#### C. S. P.'s observations.

3 inch above,  $32^{\rm cm}$  forward of knife-edge,  $f_0=16.1$  ½ inch above,  $32^{\rm cm}$  behind,  $f_0=55^{\rm cm}$ .7  $\cdot$ . Flexure at  $2^{\rm cm}$  above= $35^{\rm cm}$ .9  $A=58^{\rm cm}$ .

#### H. F.'s observations.

2°m.0 above, 35°m forward of knife-edge,  $f_0=12^{\mu}$  2°m.5 above, 34°m.5 behind,  $f_0=64^{\mu}$   $\therefore$  Flexure at 2°m above=38 $^{\mu}$ .18 A=51°m.

140°m directly below middle of knife-edge,  $f_0=156$ .  $\therefore$  F<sub>0</sub>=40 $^{\mu}$ .66; A (mean)=54°m, B=29°m.

#### EBENSBURG.

At this station the Repsold tripod stood on a hard floor of clay. Statical flexure measured by means of weight of 1.0818<sup>k</sup>. C. S. P., observer.

#### 1879, SEPTEMBER 26.

The two screw-taps binding the front legs of the tripod to top of the stand were first tightened by hand. Scale on the level of the point of suspension, 50cm to the right, and 18cm 4 forward of middle of knife-edge. 21.6 div. of scale=.001 inch.

#### Scale readings.

Wt. off.	Wt. on.	Mean diff.	$f_0$ .
8.4	61. 3		μ
9.4	<b>61.</b> 3	<b>52.</b> 3	3 <b>5</b> 8
9.6	61. 6		

P. M.—Taps wrench-tightened; scale 18cm.4 directly forward of knife-edge; 20.7 div. of scale=.001 inch.

	35. 0 35. 0 34. 5 34. 6	82. 3 81. 8 82. 0 81. 6	<b>47.</b> 2	337
Taps next loosene	d.			
	25.5	82.0	*	
	28. 0	<b>82.</b> 3	54. 7	391
	28.3	82.1		
	28.3	82, 5		
Taps next hand-ti	ghtened.		•	
	16.0	67. 6		
	16. 0	67. 5		•
	16. 2	67. 6	51.5	368
	16.0	67.7		
	16. 7			

S. Ex. 49—48

## 1879, SEPTEMBER 27.

Scale 18cm.4 forward of middle of knife-edge, as before; 21.8 div. of scale=.001 inch. Taps tightened by Mr. F. about as tight as during last four days of pendulum swinging. (*Note.*—It had been discovered that during these days the taps had only been tightened by hand.)

Scale 1	eadings.
---------	----------

Wt. off.	Wt. on.	Mean diff.	fo.				
19.0	47.8 rej		μ				
18.4	69. 0	50. 5	343				
18. 2	69. 0						
In the following r		.6 div. of s	cale = .001	inch. Mic	croscope re	focussed.	
21.0	71.5						-
20. 9	71.5	<b>50.</b> 4	3 <b>4</b> 6				
21.0	71.0						
20. 5	71. 0						
			r	osition of lines read on eye-p	oi stage microm iece micrometei	ieter, r.	
21.8	72.8			0. 1	0.5		
22.0	72. 9	<b>50.</b> 9	349	21.8	21.9		
22. 0	73. 0			<b>4</b> 3.3	<b>43.4</b>		
22.0				64. 7	64. 9		
				86.5	86.8		
7	Mean interva	a.]		21.55	21. 55		
Screws now some				21.00	21.00		
		d by Mr. 1	r.				
2. 5	<b>55. 5</b>			<b>13.</b> 2	12.5		
2. 5	<b>55.</b> 5	<b>53.</b> 0	3 <b>6</b> 3	34. 7	34.5		
2. 3	<b>55.</b> 4			77. 7	77. <b>4</b>		
				99. 5	99. 1		
Ŋ	Mean interva	d	•	21. 55	${21.58}$		
Screw-taps now ti	ghtened as t	ight as por	ssible wit	h fingers;	21.6 div. of	scale = .001	inch.
18. 5	69. 6			7.0	4.6		
18.6	69.6	50.8	348	28.5	26.3		
18. 6	69. 0		0.20	49.8	47.6		
20. 0				71.5	69. 5		
				93. 2	91. 0		
		-					
	lean interva				21.6		
Screw-taps now tig		ı wrench b	y Mr. P.;	21.2 div. o	of scale $= .0$	01 inch.	
20.0	67. 4		•	16.5	<b>20.</b> 0		
20. 0	<b>67.</b> 3	<b>47.4</b>	331	37. 7	40.9		
19.6	67. 2			<b>59. 0</b>	<b>62.</b> 2		
				81.6	83. 5		
M	lean interva	1		21.25	$\overline{21.2}$		
New set. Screws	entirely loos	e; 21.6 div	of scale	=.001 inch			
21.5	<b>75.</b> 8			16.6	<b>11.</b> 2	10.8	•
22.0	76.0	54. 0	370	38. 5	33. 2	32. 7	
21.8				59. 7	<b>54.</b> 8	54. 6	
21.7	10. 1			<i>uo.</i> 1		Ú4. 1)	
	75.7			99. 1			
	15. 1			81. 7	76. 6 97. 9	76. 1 97. 4	

Screws now tightened by hand of Mr. F.	'about right"; 21.6 div. of scale=.001 inch.
--	--

	_					- arit or source	•••
			S	cale readings	3.		
	Wt. on.	Wt. off.	Mean diff.	f <sub>0</sub> . P	osition of lines o	f stage micrometer.	
	17.0	67. 6		μ	3. 4	1.8	
	17.1	67. 3	50, 2	344	24. 9	24, 0	
	17.0	67. 0		_	46. 4	<b>45</b> , 3	
	17.2				68. 1	67. 2	
					89. 5	88. 6	
		Mean inter	val		${21.55}$	21.66	
Screws ag			l "about rig				
	20.4	70. 0		, ,	8.5	5. 0	
	20.4	70. 0	49.8	341	30. 6	26.8	
	20.0	69. 9			57.8	48. 5	
	20.1				73.3	69. 9	
					94. 9	91. 4	
	1	Mean interv	val	4	${21.52}$	$\frac{-}{21.58}$	
Screws aga	ain tigh	tened by ha	and "about	right"; 2		scale=.001 inch	
	11.5	61. 5		,	0. 7		
	11.4	61. 2	50. 0	344	22.9		
	11.2				43. 9		
					65. 7		
					87. 0		
•							
	Ŋ	Iean interv	al		21. 52		
a last sat of							

The last set of measures were not regarded as being so satisfactory as the preceding.

Head of stand taken off, put on again, and tightened with wrench. Scale 18cm.6 forward of middle of knife-edge; 21.4 div. of scale=.001 inch.

36. 4	83.4			2.6	7. 9
36. 4	83. 1	46.8	324	13. 4	18. 7
36. 4	83.1			23.0	29.2
36. 4				34.5	40.0
			•	45. 5	<b>50.</b> 8
				<b>56.</b> 1	61.6
				<b>66.</b> 9	72.1
			•	77.4	82.6
				87.8	
	Mean interv	al	· · · · · · · · · · · · · · · ·	${21.47}$	$\frac{-}{21.41}$

Another set; 21.4 div. of scale=.001 inch. N. B.—In this and following sets the positions of several, generally three, lines of the stage micrometer are read off on the eye-piece micrometer, between all the changes of the weight. This explains the separation of the numbers in the first two columns into groups.

37.4	84. 2			4.0	5.3
<b>32. 3</b>	79. 4			13. 5	10.7
26.6	74. 1	47.1	327	24.3	16.0
				<b>35.</b> 0	21.3
<b>36.</b> 9	84. 2			46. 2	26.4
31.8	79.3			56, 6	31. 9
26.4	<b>74.</b> 1	47. 5		67. 7	36, 9
				78. 0	
				88. 8	
	35	_			
	Mean interv	al		21.43	21 07

Another set; 21.5 div. of scale=.001 inch.

Wt. on.	Wt. off.	Mean diff.	$f_{\theta}$ .	Position of lines o read on eye-pi	f stage micrometer ece micrometer.
28.3	74.5			32, 1	31. 1
33, 2	80.0	46.6		37.9	37.1
38. 4	<b>85.</b> 3			43.3	42.3
• -		٠		47.9	47.0
27. 6	74.5			54.0	53.0
33, 0	80.0	47.0		59, 2	58.6
38. 2	<b>85.</b> 3			64.8	63 <b>. 9</b>
00.2	00.			70. 0	68. 9
27. 6	74. 6			75. 1	74.5
33. 0	80, 0		ш	80.8	79.8
38, 2	84. 9	46.9	$3\overline{23}$	85. 8	85. 0
28, 0	74, 6	Mea	an inter	val. 21. 51	22, 55
32.9	80. 0				
38. 0	84. 9	46. 9.			
28, 0	75. 1				
32. 9	79. 9				
38. 0	85. 1	47.1			
27. 8	74. 1				
33, 0	79. 9				
38. 0	85. 1	46.8			

Stand reversed. Scale 42cm.5 behind middle of knife-edge; 21.6 div. of scale=.001 inch. Scale readings.

	Wt. off.	~Wt. on.	Mean diff.	$f_0$ .		Lines of scale.	
	3.0	28.8			5. 0	18.4	6, 6
	24. 4	50, 6	26. 0		26.8	39.5	27.5
				μ	37.2	61.5	<b>48.</b> 8
	3. 7	29, 1		177	47, 9	83. 0	70, 6
	25. 0	50. 9	25. 6				92.6
	46.5	72, 0					
			Mea	n interval.	21.6	21.6	21.5
Microsco	pe refocus	ssed. Sca	le 21. 6 div.=	=.001 inch.			
	- 5. 6	31. 0	<b>25.</b> 7		13.0		
	26. 7	<b>5</b> 2. 7			34.0		
					<b>5</b> 6. 0		
	7.4	32.4		176	77. 2		
	28.4	<b>54.</b> 0	25.7				
	48.8	75.4	Mea	n interval.	21.55		
Again, 2		f scale=.00	1 inch.				
,	12.5	38.6			17.0	18. 6	
	33. 4	60. 2			38.6	40. 2	
	54. 0	81.4			60. 0	61. 6	
					81.5	82. 9	
	14.0	39.5	26.8	181 rej.			
	35. 0	61. 2	Mea	n interval.	21, 47	21. 4	
	56. 0	82. 5	26. 1				
	14. 4	39.[5					
	35. 6	61. 2					
	56.5	82. 5	25, 6				

Screws next entirely loose; 21.5 div. of scale=.001 inch

Scale readings.

Wt. on.	Wt. off.	Mean diff.	$f_0$ .	Lines of scale
52.0	68.6			5, 0
62.7	79. 6		μ	16.1
73.2	90.4	16.9	115	28.8
				37. 5
52.5	68.4			48.0
62.9	79.3			59. 1
73, 2	90.0	16. 4		69, 6
				80, 6
				91. 1•
				21.5

Screws now hand-tightened "about right," by H. F.; 21.6 div. of scale=.001 inch.

Again, 21.6 div. of scale=.001 inch.

Three bricks were next put on the bottom of the stand; weight, 4 pounds 5½ ounces, 4 pounds 6¾ ounces, 4 pounds 11½ ounces, respectively. The following measures were taken at 41cm.4 behind the middle of knife-edge, and 0cm.7 above level of support. Screws hand-tightened, as in last observations; 21.4 div. of scale=.001 inch.

16. 5 16. 5	34. 4	17. 9	124	2. 2 87. 8
Refocussed. Scale	as above.			21:4
20. 2 20. 2	38.5 38.6	18. 4	127	9.5 73.7 21.4

Screws next tightened with wrench; 21.7 div. of scale=.001 inch. Measures taken 1°.4 above level of support.

67.3	93. 0			2.4	5. 6
67. 4	92.4			88.8	92.7
67.3	92.6	25.2	172		
67.8	92. 7			21, 6	21.7

Measure taken 4<sup>cm</sup>.1 below level of support; 21.6 div. of scale=.001 inch. Stage micrometer fixed to the top of the tripod, but not to the tongue on which the pendulum rests.

10. 8 12. 2 13. 6	41. 0 40. 6 42. 8	<b>29.</b> 3	201	$   \begin{array}{r}     18.6 \\     83.5 \\     \hline     21.6   \end{array} $
Again, 21.7 div. of	scale=.001	l inch.		
16. 0	<b>45.</b> 0			11,8
16. 9	46.0	29, 0	197	98. 5
18, 0	46, 8			
				21.7

Again, 21.6 div. of scale=.001 inch.

TIE WILL,	21.0 (111. 01	Source 100	inon.		
			Sca	ale readin	gs.
	Wt. off.	Wt. on.	Mean diff.	fo.	Lines of scale
	20, 7	50. 0	29, 2	201	7. 0
	21.0	50. 2		201	93. 4
					$\overline{21.6}$
Again,	scale $21.5 d$	liv <b>.=.0</b> 01 i	nch.		
	23. 0	<b>51.</b> 7	29. 1	200	22, 2
	52.5	20. 1	200	86. 8	
					$\frac{-}{21.5}$

The bricks were now removed from base of support, and pendulum suspended heavy end down; 21.4 div. of scale=.001 inch. Screws wrench-tightened.

	8.0	37.4	90.0	മെ	16. 0
	8, 5	37.5	29, 2	202	80. 3
Again, 21	.6 div. ot	f scale=.00	l inch. "C	Jood."	$\overline{21.4}$
	7. 5	37.5	· mon,	300 <b>u</b> .	7. 9
	7. 5	36.8	29.9	206	94. 3
		38. 0			${21.6}$

The following is a summary of the observations with weight and pulley at Ebensburg, on Repsold stand.  $F_0$ =flexure at middle point of knife edge, under a horizontal force equal to the weight of the pendulum; A=distance from middle point of knife-edge to intersection of axis with knife-edge.

Arrangement.	18cm.4 forward, 0cm.7 above.	42cm. 5 back, 0cm. 7 above.	f <sub>0</sub> .  10° from axis of rotation.	Δ.	$\mathbf{F}_{0}$ .
	μ	μ	μ	Cm.	μ
Front taps wrenched up	337	177			
	<b>33</b> 1	176			
•	324		1		
	327				
	323		:		
	328	176. 5	24.9	113. 5	283
Front taps hand-tightened	343	137			
	349	138		:	
	344				
	341	, <b></b>			
	344			1	
4	342	137. 5	33. 6	83, 4	001
m	3 <b>42</b> 3 <b>6</b> 3	187. 3	ee. 0	83. 4	281
Taps somewhat loose	391	115	45, 3	67. 9	309
Front taps loose	991	110	<b>40.</b> 0	01. 9	309
		41cm.4 back.	Calculated from above without bricks.		
:		μ	μ		
Tripod loaded with bricks; taps hand-tightened		124		İ	
		127		1	
·		125. 5	141		
Taps wrenched		172	179	1	
жаре итополос.		With bricks.	W thout bricks.	1	
Flexure of tripod without that of tongue; taps wrenched.		201	202		
Proxime of preport without there of conduct, sales we on one		197	206		
		201			
		200			
			904	* :	
		200	204		

It will be seen that the effect of loosening the front taps is to increase the angular flexure about the instantaneous axis. But this axis is at the same time brought forward, and the consequence is that the flexure at the middle point of the knife-edge is not much changed. That the flexure of the tripod alone, without that of the tongue supporting the pendulum, appears, when measured,  $40^{\rm cm}$  behind the middle point, to be greater than the combined flexure of the two, is no doubt due to the axis of flexure of the tongue cutting the level of the knife-edge only a short distance behind the middle point. The effect of loading the base of the tripod with bricks was to make it slightly stiffer when wrench-tightened, and considerably stiffer when hand-tightened. All of these measures of flexure seem, however, to be in error, and it seems likely that the position of the scale, when in front of the stand, was not really  $184^{\rm mm}$  as recorded, but perhaps  $584^{\rm mm}$ . With that change, these measures would agree with others, which they do not now do.

The following are dynamical measurements. The pendulum swung heavy end down; 21.4 div. of scale=.001 inch. Arc expressed in ten-thousandths of the radius. Screws wrench-tight-ened; scale 41° 4.4 behind, 0° 7.7 above knife-edge.

		1879,	SEPTEMBI	ER 27.	
	Arc.	Scale r	eadings.	Diff.	$f_{0}$
	292	7.6	13. 7	6.1	$17\overset{{\scriptscriptstyle \mu}}{8}$
	291	9.0	14.8	5.8	169
	290	6. 7	12.8	6. 1	179
Another swing.	381	6. 6	14. 7	8.1	179
	379	7.8	16.0	8.2	183
	377	6.6	13.8	7.2	162
	373	6. 9	14.4	7. 5	171
	370	0.0	7.8	7.8	179
		Mean	•••••••		175.0

1879, SEPTEMBER 28.

Scale 1.0 to 87.6=.004 inch. Other conditions same as before.

Arc.	Scale	readings.	Diff.	$f_0$
520	5. 0	15. 9	10, 9	175
512	4, 0	14.8	10.8	178
506	0.0	10. 7	10.7	178
<b>50</b> 3	0. 5	11. 3	10.8	181
496	7.7	17.8	10, 1	170
492	3.8	14. 3	10. 5	177
487	2.3	12.0	9. 7	166
479	2.6	12.8	10. 2	177
465	8. 7	17.8	9.1	163
459	5. 1	14.8	9. 7	178
455	2. 2	11.4	9. 2	168
290	3.9	10. 0	6. 1	175
	Mean			173.8
l again.				
200	9 6	11 7	Q 1	172

Stopped, and started again.

	Mean	• • • · · · · · · · · · · · · · · · · ·		173. 0
383	0.3	8. 2	7. 9	173
392	3. 6	11.7	8.1	173

The following are statical measures of the flexure produced by drawing the pendulum to one side over a measured arc; steel tongue used instead of wooden strip before employed. Scale 44<sup>cm</sup>.4 behind middle of knife-edge, and 2<sup>cm</sup>.5 below its level; 1.0 to 87.6 div. of scale=.004 inch.

		Scale readings; pend. vertical.		ngs; pend. ined.	Arc.	Mean diff.	$f_0$ .	
	9. 7	74. 5	16, 6	81.1	474	6. 75	$23\overset{\mu}{3}$	
	10.4	74.8	16.0	80. 9	465	5.85	207	
Again.								
	10. 7	<b>75.</b> 3	16. 9	81.6	<b>474</b>	6.25	217	
	11.3	76. 3	16. 1	80. 7	475	4. 60	159	
	11.6	76.5	15, 8	80.4	438	4. 05	151	
	10.8	75, 6	16. 1	80.8	<b>486</b>	5.25	177	
	11. 1	76. 0	16. 2	80.7	489	4. 90	164	
	10.8	<b>75.</b> 8	16.2	80.1	429	4.85	186	
•	9. 6	74. 3	15.9	80.8	450	6. 40	233	
	7.3	71.9	12.9	77.7	501	5. 70	187	
	9. 5	74. 5	15.0	79. 7	500	5.35	175	
	10.1	<b>74.</b> 8	16.0	80. 6	515	5.85	187	
	9. 6	74.2	<b>15.</b> 0	79, 7	490	5.45	182	
						•	100	
							189	

Dynamical measurements; 21.6 div. of scale=.001 inch.

Arc.	Scale	readings.	Diff.	$f_0$ .
496	8.8	19. 7	<b>10.</b> 9	185
493	1.6	11.3	9. 7	166
490	12.3	22.6	10.3	177
488	13. 6	23.9	10.3	178
<b>485</b>	8. 7	19.4	10.7	186
482	0.9	11.0	10.1	177
480	12.6	22.7	10.1	177
477	13.7	23.8	10. 1	179
474	9. 0	19.3	10.3	182
472	0.9	10.8	9. 9	177
469.	<b>12.</b> 5	22.7	10.2	183
466	13.8	23. 7	9.9	179
463	3.8	13.7	9. 9	180
460	0.7	10.6	9. 9	181
458	2.4	11.9	9.5	174
<b>455</b>	3.5	13. 3	9.8	181
				178.8

Tongue readjusted. Scale 44cm.6 behind middle of knife-edge, and same height as before. Screws hand-tightened by H. F.; 3.3 to 89.2 div. of scale=.004 inch.

				μ.
466	10.6	18.9	8.3	150
<b>46</b> 3	11.6	19.9	8.3	150
461	3.6	12. 2	8.6	157
460	5. 4	13.7	8.3	151
457	0. 4	8.8	8.4	155
450	1.7	9. 9	8.2	154
447	4.9	13. 3	8.4	158
445	5. 7	13.9	8. 2	155

153.8

Statical	flexure	with	same	arrangement.
----------	---------	------	------	--------------

	Scale readings; pend. Scale readings; per vertical. inclined.			Arc.	Mean diff.	,fise	
	6.2	89. 6	8. 2	94. 3	464	4. 80	174
	2.3						
	3.6	89.8	7.8	93. 9	471	4.15	149
	2.3	88. 6	7. 9	<b>94.</b> 1	<b>482</b>	5.55	195
	2.5	88. 2	5. 9	<b>93.</b> 2	475	4. 70	167
	2.3	88. 4	6.9	93. 0	488	4. 60	159
	0.8	87.8	6. 1	91. 9	494	4. 20	144
	1.4	87. 6	6. 2	92.0	488	4. 60	159
Conorma n	ow looser	an bus box	roin tiahta	ned by han	a		158. 7
Screws II	OM 100861	ieu, anu ag	am ugue	nea oy nan			μ
	4.6	78.9	8.8	83. 4	482	4. 35	151
	4.6	79. 0	9.4	83.8	507	4. 80	160
	4.6	79. 1	8.9	83.5	<b>4</b> 73	4.35	155
	4.7	79.2	8. 9	83. 8	482	4. 45	155
							<del>155.</del> 3

Dynamical measures with last arrangement; 21.3 div. of scale=.001 inch.

Arc	Scale	readings.	Diff.	$f_0$ .
. 519	0.3	9. 4	9. 1	151
508		10. 5	9.2	154
500	6 1.6	10.8	9.2	155
504	<b>1</b> 2.7	11. 7	9.0	152
503	3.7	12.4	8.7	147
500	4.8	13.6	8.8	150
498	0.2	8.9	8.7	149
490	6 1.1	9.8	8.8	150
498	5 1.8	10.7	8.9	153
•				$\frac{151.2}{1}$
Screws retightened by H	. F.			μ
520	5.6	<b>15.</b> 3	9. 7	159
510		<b>16.</b> 3	9. 6	158
513	8.7	13.1	9.4	156
519		9.7	9.3	155
509		15.7	9. 7	162
50'		16.6	9. 3	156
50-	9.0	18. 4	9.4	159
50:		9. 9	9.2	156
				${157.6}$

Statical flexure. Same arrangement; 21.6 div. of scale=.001 inch.

Pend. v	ertical.	Pend. it	iclined.	Arc.	Mean diff.	$f_0$ . $\mu$
18. 6	83. 3.	23, 1	87.7	464	. 4.45	162
19.6	83. 5	23. 4	88. 1	514	4. 70	153
18.6	83. 5	23. 3	87.8	488	4. 50	155
18.4	83.4	23. 3	87.9	467	4. 70	170
18.6	83.5	23.6	88.3	513	4. 90	162
						160. 4

Focussed. Same arrangement.

Pend. v	ertical.	Pend. in	nclined.	Arc.	Mean diff.	$f_0$ .
20.0	84. 7	24. 4	89. 3	483	4. 50	157
20.1	84.8	<b>24.</b> 9	89. 3	504	4. 65	155
20.6	85. 1	25. 1	89. 2	506	4. 30	143
21.1	85.6	<b>25. 1</b>	88. 9	509	• • • •	
20.6	85. 4	25. 5	89. 9	478	4. 70	165
						$\frac{-}{155.}$

The stand turned around; tongue now projects in front of middle of knife-edge 33cm.9; height approximately as before; nuts wrenched up; dynamical.

Are.	Scale readings.		Diff.	$f_{o}$ .
472	7.3	23. 4	16. 1	$^{\mu}_{287}$
470	5.6	21.3	<b>15.</b> 7	281
467	4.0	20. 3	16.3	293
<b>465</b>	7.6	22, 9	<b>15.</b> 3	277
<b>462</b>	5. 7	20.9	<b>15.</b> 2	277
460	7. 7	22.9	<b>15.</b> 2	278
457	5.8	21.0	<b>15.</b> 2	280
•				281. 9

Statical flexure. Same arrangement; 21.4 div. of scale=.001 inch.

Scale read vert	ings; pend. ical.	Scale readi incli	ngs; pend.	Arc.	Mean diff.	$f_{o}$ .
<b>5.</b> 7	80. 2	13.0	87.1	398	7. 20	3 <b>0</b> 7
15.4	79. 6	23.8	88.1	460	8, 45	312
15.4	79. 7	23.7	87. 9	476	8.25	294
15.4	79. 6	23. 7	87.7	485	8, 20	287
14.8	79.3	23.6	87. 9	496	8. 70	297
14.9	79.3	23. 7	88. 0	480	8. 75	309
14.6	79. 1	23.4	87. 6	488	8.65	300
14.7	79.2	23.5	87. 5	491	8.55	<b>35</b> 3
14.7	78.9	23.5	87. 6	500	8. 75	297
14.6	78. 7	23. 5	87. 7	$\boldsymbol{504}$	8. 95	302
						300. 5

Dynamical flexure; 21.4 div. of scale=.001 inch; nuts tightened by hand of H. F.

Arc.	Scale r	eadings.	Diff.	$f_{0}$
426	6. 1	22. 6	16. 5	. 328
<b>422</b>	2. 2	18.6	16.4	330
420	6. 3	22.6	16.3	329
417	4. 7	21.2	16.5	336
414	5. 5	21.7	16, 2	332
412	3.8	19.8	16.0	329
410	6. 3	22. 2	15. 9	329

Statical flexure; las	t arranger	nent.				
Scale reading vertic			ings; pend. ined.	Arc.	Mean diff.	$f_0$ . $\mu$
13.9	78. 3	23. 9	87. 8	<b>49</b> 0	9. 75	340
14.0	78.1	24.1	88. 3	502	10. 15	345
14. 1	78.3	24.3	88. 4	511	10. 15	338
14.5	<b>78. 5</b>	24. 4	88. 4	503	9, 90	334
14.5	<b>78.4</b>	24. 0	87. 9	480	9, 50	336
14.5	78, 6	24.8	88. 7	521	10. 20	333
14.2	<b>78.</b> 2	23.5	87. 7	476	10. 40	370
14.0	78.4	23.6	87. 7	485	9. 45	331
14.0	77.9	23. 9	87. 9	504	9. 45	319
<b>14.</b> 5	78. 6	23. 9	88. 0	490	9. 40	326
NT 4 11 4 1 5	1 3 . G.		mamant na h	a fama		333.6
Nuts readjusted by			gement as b		8. 80	341
24.1	88. 2	32. 9	97. 0	437 504	10. 15	341
24. 2	88.3	34. 3	98. 5	504 475		372
23. 5	88.3	33.8	97.8	475 466	10. 40 9. 50	346
23.8	87.7	33. 1	97.4	466		340
23. 5	87.5	32.5	96. 7	455	9. 10	343
23. 5	87. 6	33. 4	97. 7	496	10. 00 10. 90	343 341
23.6	87.6	34.4	98.6	542	9. 70	$\frac{341}{346}$
23.4	87. 5	33. 2	97.1	475	8. 90	350
$egin{array}{ccc} 23.4 \ 23.5 \end{array}$	87.6 $87.6$	$32.3 \\ 32.9$	96. 5 97. 0	$\frac{433}{474}$	9.40	336
						${342.7}$
Dynamical flexure.	Arc.	rangement Scale re		Diff.	$f_{0}$ .	<i>()</i> 12. •
	384	0.7	<b>15.</b> 7	15. 0	$3\overset{\scriptscriptstyle{\mu}}{3}1$	
	382	6.6	21.7	15. 1	334	
	380	8.3	23.0	14. 7	329	
	379	10.3	24.7	14. 4	322	
•	484	8.5	27. 7	19. 2	336	
	479	6. 1	24.7	18.6	329	
·	474	6.6	24.8	18.2	326	
	471	8. 2	26, 6	18.4	331	
					${329.8}$	
Screws again tighte	ned by ha	nd of H.	F.			
	412	3.0	19.4	16.4	337	
	410	4.1	20.6	<b>16.</b> 5	341	
	<b>40</b> 8	2, 6	<b>19.</b> 3	16, 7	347	
	407	4. 2	20. 4	16.2	337	
•	405	3.2	<b>19.</b> 3	16.1	337	
	404	4.4	20. 2	15.8	332	
	523	<b>0.</b> 5	21. 5	21.0	340	
	520	0. 6	21.6	21.0	342	
	518	5. 5	25. 9	20.4	334	
	514	3, 8	24.3	20.5	338	
•	511	1. 0	21. 3	20.3	336	
	506	5. 5	25.7	20.2	338	
	504	5. 4	24.8	19. 4	328	
					337. 4	

Statical flexure; same arrangement; 21.5 div. of scale=.001 inch.

Scale readi	ngs; pend.	Scale readi	ngs; pend.	Arc.	Mean diff.	$f_0$ .
verti	cal.	incli	ned.			
10.6	75.4	20.9	85. 1	-492	10, 00	346
10.9	75. 1	20. 1	84.4	457	9, 25	344
10.6	74. 9	20.3	84.7	485	9. 75	341
10.4	74.9	20.8	82.0	509	10. 25	342
10.5	74. 9	21.2	85.9	505		
10.6	75.3	21.0	84. 2	494		
10.6	<b>75.</b> 3	21.0	84.8	501	9, 95	337
10.4	75, 0	21.3	85. 7	517	10, 80	355
10.4	74.6	20.4	84. 9	496	10. 15	347
10.3	74.7	20.5	84.9	500	10. 20	346
					10 <sup>div</sup> , 03	${344.7}$

Summary of observations with pendulum, dynamical and statical, made at Ebensburg on Repsold stand.

## Nuts wrenched.

	Dynam.	Stat.
44cm.4 behind knife-edge,	$f_0 = 178  ^{\mu}.  8$	189 4.0
33cm.9 forward,	$f_0 = 281  \mu.  9$	300 ±, 9
	∴ $F_0 = 237  \mu.  2$	252 ±. 7
	$A = 182^{em}.9$	$176^{\mathrm{cm}}.5$

## Nuts hand-tightened.

44cm.6 behind knife-edge,	$f_0 = 154  \mu.  2$	158 <sup>\(\mu\)</sup> . 6
33cm.9 forward,	$f_0 = 332^{\mu}.4$	339 4.4
,	∴ $F_0 = 255  \mu$ . 4	261 <sup>µ</sup> . 3
	$A = 112^{cm}.5$	113cm. 4

## YORK.

H. F., observer. All observations at this station, made in two positions, carefully brought to the level of the knife-edge plane. Ebensburg weight=1\*.0818.

## 1879, NOVEMBER 8.

Scale 47cm in front of middle agate; 76.2 div. of scale=.003 inch; Geneva support.

Scale readings.					
Wt. on.	Wt. off.	Diff.	$f_0$ .		
28.5	37.0				
28.0	40.0		μ		
28.0	38, 0	10.3	58.7		
28.0	38. 5				
28.5	39. 0				
29.0	39.0				
23.5	34.0				
24.0	35.0				
24.0	35. 0	10.6	60.4		
24.0	34.0				

22.5	33.5	6. 0	18. 0		
23.9	33. 0	6.0	16.5		
23.0		6.5	17. 0	10.6	60.4
		7. 0	18.0		
		7.0	17.5		
		7.5	18.0		
		7.5	18.0		
		8.0			

Scale put on 46cm behind middle of agate; 85.5 div. of scale=.003 inch. Measures not very good, on account of jarring of machinery.

Scale readings.

	•••••		
Wt. on. 31. 0	Wt. off. 49, 5	Diff.	$f_0$
31.0	49.5		
30. 5	49. 0		μ
30. 0	48.0	18. 4	93.5
$29.\ 5$	48. 0		
29.5	47.0		
29.0	47.0		
28. 5			
13. 5	32.5		
12.0	31.5		
13.0	31. 5		
14.0	32.0	18. 7	95, 0
14.0	32.5		
14.0	33.0		
14.0	<b>3</b> 2, 0		
12.5			

1879, NOVEMBER 9.

Sunday. Shops all still. Scale 46cm.6 behind the middle of the knife-edge; 91.87 div. of scale=.003 inch.

Scale readings.

0,003	inch.	Wt. on.	Wt. off,	Diff.	$f_0$ .
91.8	91.8	29.5	11.2		
92.0	91.7	29.7	11, 1		
<b>92.</b> 1	92.0	29. 9	12.1		
92.1	91. 9	31.3	13.0		
92. 1	91.8	30, 9	13.0	a.	μ
	92.0	31.7	14, 0	18.1	85.4
		32.7	14.2		
	•	33.3	14.9		
		32. 9	15, 1		
		33.8	16. 1		

Again.	93.47 div. of sca	le=.003 in		7.		
	.003 inch.		Wt. on.	readings. Wt. off.	Diff.	ſ
	93. 9		41. 7	22. 8	Din.	$f_0$ .
	93. 5		41. 9	22. 7		
	93. 3		41. 7	22. 9	18. 85	87. <b>4</b>
	93. 4		41.2	22. 5	101 00	01. 1
	93. 3		41. 5	22. 9		
	93. 4		41.8	22. 0		
Again.	93.51 div. of sca	le= 003 in				
	93.4	10	37. 3	18. 7		
	93. 3		37.2	18. 1		
	93. 8		36. 9	18. 0		
	93. 7		36.6	18.1	18.8	87.2
	93. 4		36. 6	17. 7	10.0	01.2
	93. 3		36. 7	18. 0		
	93. 7		37. 5	18. 3		
	<i>70.</i> 1		36. 9			
			36. 9	18. 1		
Soala nu	it on 460m 6 in fro	nt of midd		ifo adms. 70 C	9 Air e	
iscate pu	it on 46cm.6 in fro	nt or midd		ne-eage; 10.0 readings.	o aiv. or s	cale=.002 mcn
	.002 inch.		Wt. on.	Wt. off.	Diff.	$f_{0}$
	70.8		27.0	37.0		
	70. 3		<b>26.</b> 6	37. 9		μ
	70. 9		26.8	37.0	10.4	42.6
	<b>70.</b> 6		26.4	37.0		•
	70. 9		26.7	37.0		
	70. 3		07.0			
			27.8	38. 2		
			27.7	<b>38. 0</b>		
A *	50 F5 1: 0 1	000 *	27.9			
Again.	70.57 div. of scal	e=.002 m		00.4		
	70. 8		22.7	33. 1		
	70. 2		24. 7	35. 4		
	<b>70</b> . <b>7</b>		25. 4	35.8		
	70. 7		25. 9	36. 3		
	70. 4		25. 3	35. 9		
	70. 6		25. 5	<b>36.</b> 1	<b>10</b> . <b>24</b>	<b>42.</b> 0
			26. 0	36. 2		
			26. 4	<b>36. 1</b>		
			26. 2	<b>36. 5</b>		
			26.0	<b>35.</b> 9		
			<b>25.</b> 8	<b>36.</b> 8		
			27.0	<b>37. 0</b>		
			27.5	•		
Again.	Draw-tube shorte				ach.	
	.004 inch.		Vt. on.	Wt. off.	Diff.	$f_0$ .
,		96. 8	24.1	32. 1		
		14	25. 0	31. 9		
		97. 2	24.1	31.3	# AO	44.0
			24. 0	31. 3	7.49	44. 6
		97. 7	23.8	31.5		
		97. 7	23. 9	31. 8		
			24. 6	31. 9		
	97. 0		24. 2 23. 0	31. 7		
			- e = 1.8			

# 1879, NOVEMBER 13.

5.30 p. m., and machinery stopped. Scale put on 46cm.6 in front of agate; 98.38 div. of scale=.004 inch.

inch.		Scale r	eadings.		
	. 004 inch.	Wt. on.	Wt. off.	Diff.	$f_{o}$ .
	98. 3	2.6	10.0		
	98, 3	3.4	9.2		
	98. 5	2, 8	9. 0		
	98. 2	2. 2	8.8		
	99. 0 98. 2	3.1	<del></del>		,,
	98. 2	2.8	8.9	6. 43	37.8
	98. 3	2.1	8.9		
		2.0	8. 6		
		2.9	8.3		
		2.8	9. 0		
		2.0	10. 0		
		3.1	9. 1		
		3.0			
Again.	Draw-tube lengthened;	7.84 div. o	f scale=.00	3 inch.	
	.003 ineh.	Wt. on.	Wt. off.	Diff.	$f_{\alpha}$
	97.8	26.8	<b>36.</b> 7		
	97. 6	28. 2	38.1		
	98. 1	27.8	37.2		
	98. 0	28.0	<b>36.</b> 8		
	97. 9	27. 9	37.6	9.01	40.0
	<b>97.</b> 8	29. 2	<b>37.</b> 2		
	97. 6	28. 0	37. 4		
	97. 9	<b>29.</b> 7	37.2		
		29.8	39, 1		

Again.

17.4	26. 2		
18. 1	27.7		
19.0	27. 2		
18.4	27.0	8.65	38.4
<b>18.</b> 1	26. 9		•
17.8	26. 6		
17.8	25. 9		
17. 9			

36.8

## 1879, NOVEMBER 16.

Morning. Draw-tube=1.35; 82.28 div. of scale=.003 inch.

## $Scale\ readings.$

. 903 inch.	Wt. on.	Wt. off.	Diff.	$f_0$ .
82, 0	4. 0	· 10. 0		•
82.8	4.3	10.5		
82. 2	4. 5	10. 2		
82. 1	4.8	9. 9		
82. 0	4.1	9, 8	5, 93	31.3

			Scale 1	readings.			
	.003 inch.		Wt. on.	Wt. off.	Diff.	$f_0$ .	
	82. 0		4.6	10. 4			
	82. 5		4. 1	10. 2			•
	82.5		4, 3	<b>10.9</b>			
	<b>82. 4</b>		4. 1	10. 9			
*******	3 3		5. 2	1 000			
Draw-tu	be lengthened	to 5.5; 7t					
	.002 inch.		Wt. on.	Wt. off.	Diff.	$f_0$ .	
	<b>75.</b> 8		10. 9	19.8			
	76. 5		11.0	20. 2			
	<b>76.</b> 3		11. 1	20.3			
	76. 1		11.3	20.3		μ	
	<b>76.</b> 2		12. 2	21. 9	8. 94	34.0	
	<b>76.</b> 0		13.8	24. 0			
	76. 1		13. 2	22.0			
			13. 2	20.8			
			13.0				
Again.	Draw-tube 4.0	•	v. of scale=	002 inch.			
	.002		Wt. on.	Wt. off.	Diff.	$f_0$ .	
	67. 2	<b>67.</b> 8	27. 2	<b>34.</b> 8			
	67. 0	67. 7	27.3	<b>35.</b> 0			
	67. 4	<b>66.</b> 9	26. 9	<b>34.</b> 2	7. 70	33. 1	
	67. 3	67. 2	27. 2	35. 1			
	67. 5	<b>67.</b> 9	27. 7	35. 1			
			27.2	35. 9			
A main			28. 0				
Again.			28.9	37.0			
			28. 5	o <b></b>			
			28.0	36.0			
			28.1	36. 0		•	
			28. 2	37. 0			
			28.3	37.8			
	,		29.9	38. 0			
			30.0	38. 1	8, 23	35. 4	
			30.0	38. 1	0, 20	00, 1	
			30. 1	38. 0			
			30. 5	38. 0			
			29. 1	37. 9			
			30.0	40.0			
			32.3	41. 0			
			33. 0	41.1			
			33. 2			,	
Scale put	on 46cm.6 behind	nd middle	of knife-ed	ge. Draw i	tube=0;98	3.02 div. of a	scale=.004 inch.
	.004 inch.		Wt. on.	Wt. off.	Diff.	$f_0$ .	
	97. 9		38. 2	22.4			
	97. 7		37.8	22. 2			
	97. 9	•	38.2	22. 2			
	98. 3		38. 4	22.7	15. 46	91. 3	
	98. 4		38. 2	23. 0			
	98. 2 97. 0		38. 1	23. 1			
	97. 9 97. 9		38.1	23. 2			
	ð í , ð						

Again, tube=5.4; 37.94 div. of scale=.001 inch.

	. 002	. 002 inch.		Wt. off.	off. Diff.	
	76. 0	75.9	33. 1	10. 1		
	76.2	$\dot{7}5.5$	33.2	9. 0		μ
	75. 9	76. 1	33.0	9.7	23, 20	. 88.5
	76. 2	76.2	33.1	9.1		
		76.6	33. 1	10.8		
•	•		33. 7	10.7		
•			33. 9			
Again.						
	75. 1		41.6	18.3		
	75.3		41.5	18.1		
	75.8		41.6	18.0	23.40	89. 2
	76.0		42. 0			
	76. 1		42.2	19. ð		

1879, NOVEMBER 19.

Scale 46	cm.6 behind agate.	Tube 0; 96.96 di	v. of scale=	=.004 inch.	
		Scale readings.			
	.004 inch.	Wt. on.	Wt. off.	Diff.	$f_0$ .
	97. 0	28. 9	14.4		•
	96. 9	29. 3	14.7		
	97. 1	29.1	15.0		и
	96.8	29.8	15.8	14.44	86.3
	97. 0	30, 1	15.7		
		30.3	16.0	•	
		30.4	15.7		
		30.2	15.8		
		30.0			
Again, t	ube=5.4; 74.69 div.	of scale=.002 in	nch.		
1	.002 inch.	Wt. on.	Wt. off.	Diff.	$f_0$ .
	74. 0	22.7	0.2		•
	74. 7	22, 9	0.0		•
	<b>74.</b> 8	22.8	0.9		
	74. 9	23.1	1.2	22.36	86. 6
	74. 9	23.3	0.3		
•	<b>74</b> . 7	23.2	1.0		
		22.3	0.5		
		23, 2	, 0.8		
•		23.2	A		
Again, t	ube 3.6; 98.7 div. of	scale=.003 inch	١.		
	.003 inch.	Wt. on.	Wt. off.	Diff.	$f_0$ .
	98. 7	36.9	17. 1		•
	98. 7	36, 8	16.8		
	98.7	36.3	16.8		
	,	35.9	<b>16.</b> 3	19.80	87.2
	•	36.0	16. 2		
		36.1	15.9	•	

35, 0

15.3

S. Ex. 49——50

Again, tube 1.8; 85.78 div. of scale=.003 inch.

.003 inch.	Wt. on.	Wt. off.	Diff.	$f_0$
85. 7	36. 1	19. 1		<b>J</b> (1)
86.0	36.0	19. 0		
85.8	36.8	19. 9	17.01	8 <b>6.</b> 1
85. 5	36. 1	19. 0		
<b>85.</b> 9	<b>36.</b> 1	19, 0	•	
	36. 0			

Note.—When a wagon passes by on the street (ground somewhat frozen), the agitation of the apparatus is so violent that the lines wholly disappear. Tremor estimated at  $5^{\rm div}$  (sometimes even  $12^{\rm div}$ , when a wagon is moving very rapidly and is exactly opposite);  $1^{\rm div} = 0^{\mu}.889$ .

Scale 46cm.6 in front. Tube 1.8; 85.08 div.=.003 inch.

85.0	30, 0	<b>37.</b> 0		
85. 3	29. 1	37.3		
85. 1	29. 0	36. 7		
85.0	29. 1	<b>36.</b> 8		
85. 0	28.8	35, 9	7.81	39.9
	28. 0	36. 7		
	29.0	36. 1		
	28.8	36. 7	9	•
	28.8	37.3		
	29. 2	* •		
	_			

Again. Tube 4.8; 70.94 div. of scale=.002 inch.

.002 inch.	Wt. on.	Wt. off.	Diff,	$f_0$ .
70. 9	7.3	16. 9		
71.0	7.0	<b>16. 2</b>		
70. 7	6. 2	15.0		•
71.1	5. 9	14.3		
71.0	5.9			
	6. 0	<b>15.</b> 9	9.38	<b>38.</b> 3
	<b>5.</b> 8	14. 4		
	4.3	<b>14.</b> 2		
	<b>5.</b> 3	14.3		
	<b>5.</b> 2	<b>15.</b> 2		
	4.3	14.3		
	5. 6	<b>14</b> , 2		
	5.0			

1879, NOVEMBER 23.

Flexure apparatus readjusted. Pieces of hoop iron substituted for heavier strips. So much agitation that experiments were postponed.

1879, NOVEMBER 26.

Scale 43cm.5 forward of middle of knife-edge, and 111cm below. Tube 5, with  $\frac{2}{3}$  objective; 98.9 div. of scale=.009 inch.

	Scale readings.		•	
009 inch.	Wt. on.	Wt. off.	Diff.	$f_0$ .
98.8	11.1	23.0		
98.4	10.9	23. 7		
98. 9	9. 7	22.2		
99.0	9. 2	20.7	12.73	167. 6
99.0	7. 9	20.0		
99. 1	6.5	19. 4		
99, 0	6, 2	18.9		
99. 0	5.7			

Summary of statical observations with weight and pulley made at York upon Geneva support.  $F_0$ =flexure at middle point of knife-edge; A=distance of middle point to intersection of axis with knife-edge; B=distance in a vertical line from middle point to axis.

46<sup>cm</sup>.6 forward of knife-edge,  $f_0 = 38.6$ ; Nov. 9, 43.1; Nov. 13, 38.7; Nov. 16, 33.5; Nov. 19, 39.1. 46<sup>cm</sup>.6 back of knife-edge,  $f_0 = 87.7$ ; Nov. 9, 86.7; Nov. 16, 89.7; Nov. 19. 86.6. 43<sup>cm</sup>.5 forward, 111<sup>cm</sup> below,  $f_0 = 167.6$ .

 $F_0$ : A=10<sup>-4</sup> 0.527 A=119.8.

 $F_0: B=10^{-4} 1.148 B= 54.9.$ 

#### 1879, DECEMBER 7.

Dynamical flexure. Scale 52<sup>cm</sup>.5 behind middle of knife-edge. In these and the following experiments the silver are is always carefully placed with its zero exactly under the pendulum point at rest; 90.39 div. of scale=.003 inch.

. 003 inch.	Arc.	Scale re	eadings.	Diff.	$f_{0}$ .
90.0	. 0350	9. 2	13.7	4. 5	77. 6
90. 7	346	10.1	14.6	4. 5	78. 3
90. 5	<b>341</b>	10.3	13, 9	3.6	63. 2
90. 0	330	13.0	<b>16.</b> 8	3.8	69. 2
90. 4	328	12.4	16.5	4. 1	75. 2
90. 9	325	12.4	<b>15.</b> 8	3.4	62. 6
90.2	319	<b>12.</b> 3	16. 1	3.8	71. 6
	296	<b>16.</b> 2	20.0	3.8	77.1
	294	14.8	18.2	<b>3.4</b>	69. 2
	292	15.1	18.8	3. 7	76. 5
	290 rej.	13.8	18. 2	•	
	285	16.3	19. 1	2.8	
	282	<b>15.4</b>	18.4	3. 0	<b>6</b> 3. 8
	280	17.0	20. 2	3. 2	68.6
	278	17.0	20. 2	3. 2	71.6
	269	17.0	20. 4	3.4	<b>75.</b> 9
					${71.5}$

1879, DECEMBER 14.

Scale 52° ... 5 behind; 1 div. of scale=0".843; dynamical.

Arc.	Scale r	eadings.	Diff.	$f_{o}.$
. 0374	29. 5	24. 7	4.8	77. 1
<b>369</b>	<b>32. 3</b>	27.8	4.5	73 <b>. 4</b>
<b>365</b>	6.8	2.7	4. 1	67.4
3 <b>61</b>	8. 9	4. 2	4.7	78.9
352	11. 9	8. 2	3.7	63, 2
348	13, 1	8.8	4.3	74.0
3 <b>44</b>	14. 7	<b>10.</b> 8	3.9	68.0
341	<b>16.</b> 2	<b>12.</b> 3	3.9	68.6
321	<b>11</b> . <b>2</b>	7.2	4.0	74. 6
319	11.6	7.8	3.8	71.6
317	11. 3	7. 5	3.8	72.2
314	11.3	7.3	4.0	76.5
310	11.0	7.1	3.9	<b>75.</b> 9
309	12.0	7. 9	4.1	80.1
308	10.8	7.0	3.8	74.0
306	10.8	7.2	3.6	70. 2
305	10.8	$7.2$ $\left.\begin{array}{c} 7.2 \\ 7.1 \end{array}\right\}$ $\left.\begin{array}{c} \mathbf{good} \end{array}\right.$	3. 7	72.8
		•		

72.9

Again, evening; 81 div. of scale=.003 inch.

.003 inch.	Arc.	Scale	Scale readings.		$f_0$ .	
80.9	. 0353	4. 2	0.3	3.9	73. <sup>"</sup> 9	
81.0	351	3. 9	0. 2	3.7	70.6	
81.0	350	4. 9	1.1	3.8	73.2	
	348	5. 0	1.2	3.8	73. 2	
	345	5. 0	1.4	3.6	69. 9	
	342	13.3	9. 7	3.6	70.6	
	339	13. 1	9.8	3. 3	<b>65.</b> 2	
	335	<b>13.4</b>	9.8	3, 6	71.9	
	333	<b>13.</b> 6	10.0	3.6	72.6	
	330	14.4	10.7	3.7	75.3	
	328	14.0	11.0	3.0	61. 2	
	325	<b>14.</b> 8	11.2	3.6	74. 6	
					${71.2}$	

Statical flexure. Same position as before; 35 div. of scale=.001 inch. Readings taken in two positions of pendulum, zero and .0370 out.

		Scale rec	idings.		
.002 inch.	Arc.	Zero.	Out.	Mean diff.	$f_{0}.$
70. 0	. 0370	0. 7	3.2		
70. 0		0.2	2.9		
		0. 2	2.3		
	•	0.0	3.0	div.	
		0. 2	3.0	2.50	70, 0
		0. 1	2.4		
		0.3	2.9		
		0. 2	2.3		
		-0.1	2.0		
		-0.1	1.9		
		-0.9	1.7		
		<b>-0.</b> 8	1.6		
		<b>-0.</b> 9			

Scale 46cm.0 in front of knife-edge; 69.74 div.=.002 inch.

.002 inch.	Arc.	Zero.	Out.	Mean diff.	$f_{0}$ .
69.8	. 0370	27. 0	25. 1		., .
70.4		26, 3	24.9		
69.1		<b>25.</b> 8	23.9		
69. 7		<b>25.</b> 3	23.8		
69. 6		25.3	23.2	div.	μ
69. 2		23.9	22.2	1.65	46.6
70. 2		23.2	22.2		
69.9		23. 1	21.6		
<b>69.</b> 8		23.0	21.0		

21.5

20.8

20.9

23.0

22.6

22.4

 $Scale\ readings.$ 

Dynamical flexure. Scale 46cm.0 in front of middle of knife-edge; 34.72 div. of scale=.001 inch.

002 inch.	Arc. Scale rea		adings.	Mean diff.	$f_0$ .	
69. 6	. 0363	25.3	22, 3	3.0	43.2	
<b>69.</b> 3	359	24.8	21.8	3.0	43.7	
69. 4	356	24.3	21.4	2.9	42.6	
	350	22.6	20.0	2.6	39.0	
	343	21.9	19.7	2. 2	33, 5	
	334	20.1	17.9	2. 2	34.5	
	329	19.5	17.0	2.5	39.7	
	- 323	18.8	16.3	2. 5	40.5	
	319	18.5	15.7	2.8	<b>45.</b> 9	
	309	25.0	21.8	3, 2		
	305	22.0	19.3	2. 7	<b>46.</b> 3	
	293	22. 9	20.3	2. 6	46. 4	
	279	22.0	20. 2	1.8	33.7	
	276	21.1	19. 5	1. 6		
	274	21.7	19.4	2.3	<b>43.</b> 9	
	270	23.0	21.2	1.8	34. 9	
	261	23. 7	21.6	2.1	42. 0	
					$\frac{1}{41.0}$	

1879, DECEMBER 15.

Statical flexure. Scale 46cm.0 in front of middle of knife-edge; 34.57 div. of scale=.001 inch.

		Scale re	adings.		
. 002 inch.	Arc.	Out.	Zero.	Mean diff.	$f_0$ .
69. 1	. 0370	<b>16.</b> 0	17.3		,
69. 1		13. 5	14. 7		
69. 2		12.7	13.4		
		11. 5	12.8		
		11.0	12.1	*	*
		10.8	11. 9	điv.	
		9. 2	10.5	1.125	32.0
		9.8	11.0		
		9.1	10.0		
		8. 2	9. 1		
		7.6	8.7		
		22.9	24. 1		
	•	20.9	22.2		
		20. 1	21.2	•	
		18.9	20.0		
		17.8	19.0		

Dynamical flexure; 46cm.0 in front; 34.57 div. of scale=.001 inch.

Arc.	rc. Scale readings.		Diff.	$f_{0}$ .
. 0356	18.0	20.7	2.7	39. 8
352	<b>16.</b> 3	18.8	2.5	37.3
348	14, 0	16.8	2.8	42.3
341	13.8	16.3	2.5	38. 5
338	14.9	17.1	2.2	34. 2
334	12.8	<b>15.2</b>	2. 4	37. 7
323	13. 5	16.0	2.5	40, 6

Arc.	Scale re	eadings.	Diff.	$f_{0}$ .
321	12.5	14.8	2.3	39. 5
319	$12.0^{\circ}$	14.4	2.4	43.6
313	11.2	13.8	2.6	39.1
309	11.6	13.9	2.3	39. 6
305	10.9	13.2	2.3	36.9
299	9.9	12. 0	2, 1	
				39 1

Dynamical flexure; 52cm.5 behind middle of knife-edge; 35.35 div. of scale=.001 inch.

.002 inch.	Arc.	Scale readings.		Diff.	$f_0$ .
70. 7	. 0355	11.9	16.7	4.8	69.3
70. 7	351	10.4	<b>14.</b> 8	4.4	64.1
	347	10.1	14.4	4.3	63. 6
	345	9.9	14.2	4.3	64. 1
	341	9, 2	13.7	4.5	67.7
	338	9.6	13.9	4.3	65.2
	334	8.8	13, 3	4.5	<b>69.</b> 3
	331	8.8	13.2	4. 4	68. 2
	316	8.3	12.7	4.4	71.3
	312	8. 7	12.9	4.2	68.7
	310	8.8	12.8	4.0	66, 2
	308	8.0	12, 2	4. 2	<b>69.</b> 8
	305	8.0	12.1	4.1	69, 3
	302	8.5	12.4	3.9	66. 2
					$\frac{-}{67.4}$

Statical flexure. Same conditions as above.

	scute readings.				
Arc.	Out.	Zero.	Mean diff.	$f_0$ .	
. 0370	9.6	7.4			
	9.3	7. 2			
	9.2	6. 9			
	9. 0	6.8			
	8.4	6.4	div.	μ	
	8.4	6.2	2.21	61.3	
	9.3	7.1			
	9.8	7.2			
	7. 0	4.8			
	6. 9	4. 7			
	6.9	4.8			
	6. 7	4.5			

Statical flexure. Scale 52cm.5 behind middle of knife edge; 34.3 div. of scale=.001 inch. Binding-screws all loosened.

Scale readings.					
.002 inch.	Arc.	Ont.	Zero.	Mean diff.	$f_{0}$ .
68. 6	. 0370	<b>27. 4</b>	<b>25.</b> 1		
68. 6		27. 7	25.2	div.	μ
		27. 1	24.6	2, 48	70.9
		26.8	24.3		1.2
		26.8	24.4		
		26. 4	23.7		

Dynamical flexure. Same condition as above.

Arc.	Scale 1	eadings.	Diff.	$f_0$ .	
. 0335	21. 9	26.4	4. 5	. 71.0	
330	21.7	$26.\ 3$	4. 6	73.7	
327	21. 7	<b>26.</b> 1	4.4	71.2	
321	21.2	25.5	4. 3	70, 9	
318	21.3 ·	25.2	3, 9	64. 9	
312	21.0	24.8	3.8	64.4	
				69.4	

Scale 46cm.0 in front of middle of knife-edge; 33.83 div. of scale=.001 inch. Binding-screws loose; dynamical flexure.

.002 inch.	Arc.	Scale readings.		Diff.	$f_0$ .
67. 6	. 0362	15, 0	17.7	2. 7	40. 0
67.7	358	12.7	15.8	3.1	46. 4
	335	6. 7	8.8	2.1	33.6
	334	4.7	7. 0	2.3	36. 9
	329	2.1	4.7	2.6	42, 4
	325	3.0	5.7	2.7	44.5
			l <sub>en</sub>		40.6

Statical flexure under same conditions.

Scale readings.

Arc.	Out.	Zero.	Mean diff.	$f_0$
. 0370	29. 0	29. 9		-
	23. 9	25.0		
	19. 7	20.6	div.	gi.
	19.7	19. 2	1. 10	31. 9
	17.1	18.1		
	15. 6	16. 7		
	14.4	<b>15.</b> 6		

Summary of observations, dynamical and statical, made at York, with pendulum on Geneva support.

### Screws wrenched.

 $\therefore \mathbf{F}_0 = 59 + .9$ 

 $F_0: A = 0.292$ 

 $A=192^{cm}$ 

52 µ.7

133cm

0.396

The statical measures are evidently unreliable. The dynamical measures show that the binding screws have no effect.

1879, DECEMBER 21.

Repsold stand; the three legs hand-tightened both above and below. Statical observations with weight and pulley; weight used= $1^k$ ; 27.78 div. of scale=.001 inch. Scale  $56^{em}$ .7 in front of knife-edge.

			Scale	readings.			
	3 inch.		Wt. on.	Wt. off.	Wt. on.	Mean diff.	$f_{0}$
82.8	83. 2		16.6	80. 7	16.8	div.	μ
83.8	83. 3		10. 1	73. 9	9.9	63. 95	368.8
84.1	83. 2		4.0	67.5	2.8		
83. 0			4.2	67. 9	3. 8		
	•		<b>13.</b> 2	76. 7	12. 4		
Now wren	ch-tightened	l below ar			· ·	iv. of scale=	.001 inch.
			16. 7	<b>76.</b> 9	15.3	div.	μ
			12.2	73.9	12.8	61. 16	352.7
			8.1	<b>6</b> 9 <b>.</b> 2	7. 9		
Microscop	e refocused;	27.47 div	. of scale=	.001 inch.			
	.003 inch.		Wt. on.	Wt. off.	Wt. on.	Mean diff.	$f_0$ .
82. 2	82. 6	82.6	20.3	80.4	19.1	div.	μ
81.9	82.4	82.7	19.8	80.0	20. 2	60. 2	351.2
			20.7	80.2	19.9		
Nuts again	n wrench-tig	htened bo	oth above a	nd below;	27.77 div. o	f scale=.001	inch.
.003 inch.			Wt. on	Wt. off.	Wt. on.	Mean diff.	$f_0$ .
83.3			27.2	88.8	<b>27.4</b>		
83.3			27.2	88.4	26.8	div.	μ
83. 3			23.4	84. 2	23.4	61. 22	353. 3
			20.6	81.8	20.0		
						•	
			19.1	79.8	18. 7		
Scale 50cm.	1 behind mi	ddle of k				1tened; $27.7$	4 div. of a
inch.	1 behind mi	ddle of k				itened; 27.7	4 div. of
	1 behind mi	ddle of k				ntened; 27.7	4 div. of
inch.	1 behind mi	ddle of k	nife-edge.	Nuts still	wrench-tigl	•	•
inch. 83. 2	1 behind mi	ddle of k	nife-edge. 14.3	Nuts still	wrench-tigh	atened; 27.79 div. 28.82	4 div. of a
83. 2 83. 2	1 behind mi	ddle of k	nife-edge. 14. 3 15. 1	Nuts still 43. 5 44. 0	wrench-tigh 14. 9 15. 9 16. 5	div.	μ
83. 2 83. 2 83. 3	1 behind mi	ddle of k	14. 3 15. 1 16. 3	Nuts still 43. 5 44. 0 45. 4	14. 9 15. 9	div.	μ
inch. 83. 2 83. 2 83. 3 83. 2	1 behind mi		14. 3 15. 1 16. 3 17. 5 19. 0	Nuts still 43. 5 44. 0 45. 4 46. 3 47. 9	14. 9 15. 9 16. 5 17. 5	div.	μ.
inch. 83. 2 83. 2 83. 3 83. 2			14. 3 15. 1 16. 3 17. 5 19. 0	Nuts still 43. 5 44. 0 45. 4 46. 3 47. 9	14. 9 15. 9 16. 5 17. 5	div.	μ.
inch. 83. 2 83. 2 83. 3 83. 2			14. 3 15. 1 16. 3 17. 5 19. 0	43. 5 44. 0 45. 4 46. 3 47. 9 bove.	14. 9 15. 9 16. 5 17. 5 19. 0	div. 28, 82	μ.
inch. 83. 2 83. 2 83. 3 83. 2	ghtened belo		14. 3 15. 1 16. 3 17. 5 19. 0 sightened al	Nuts still 43. 5 44. 0 45. 4 46. 3 47. 9 bove. 58. 3 36. 2	14. 9 15. 9 16. 5 17. 5 19. 0 27. 3 3. 7	div. 28, 82 div.	166. 5
inch. 83. 2 83. 2 83. 3 83. 2			14. 3 15. 1 16. 3 17. 5 19. 0 sightened at 26. 7 3. 7 2. 3	Nuts still 43. 5 44. 0 45. 4 46. 3 47. 9 bove. 58. 3 36. 2 34. 8	14. 9 15. 9 16. 5 17. 5 19. 0 27. 3 3. 7 2. 5	div. 28, 82	μ.
inch. 83. 2 83. 2 83. 3 83. 2	ghtened belo		14. 3 15. 1 16. 3 17. 5 19. 0 sightened at 26. 7 2. 3 1. 8	Nuts still 43. 5 44. 0 45. 4 46. 3 47. 9 bove. 58. 3 36. 2 34. 8 34. 0	14. 9 15. 9 16. 5 17. 5 19. 0 27. 3 3. 7 2. 5 1. 6	div. 28, 82 div.	166. 5
inch. 83. 2 83. 2 83. 3 83. 2	ghtened belo		14. 3 15. 1 16. 3 17. 5 19. 0 sightened at 26. 7 3. 7 2. 3	Nuts still 43. 5 44. 0 45. 4 46. 3 47. 9 bove. 58. 3 36. 2 34. 8	14. 9 15. 9 16. 5 17. 5 19. 0 27. 3 3. 7 2. 5	div. 28, 82 div.	166. 5
inch. 83. 2 83. 2 83. 3 83. 2  Wrench-tig	ghtened belo	w, hand-t	14. 3 15. 1 16. 3 17. 5 19. 0 sightened at 26. 7 3. 7 2. 3 1. 8 1. 4 1. 1	Nuts still 43. 5 44. 0 45. 4 46. 3 47. 9 bove. 58. 3 36. 2 34. 8 34. 0 33. 2 33. 0	14. 9 15. 9 16. 5 17. 5 19. 0 27. 3 3. 7 2. 5 1. 6 1. 2 1. 1	div. 28, 82 div. 32, 05	166. 5
inch. 83. 2 83. 2 83. 3 83. 2  Wrench-tig	ghtened belo	w, hand-t	14. 3 15. 1 16. 3 17. 5 19. 0 sightened at 26. 7 3. 7 2. 3 1. 8 1. 4 1. 1	Nuts still  43. 5 44. 0 45. 4 46. 3 47. 9 bove.  58. 3 36. 2 34. 8 34. 0 33. 2 33. 0 below; 27.7	14. 9 15. 9 16. 5 17. 5 19. 0 27. 3 3. 7 2. 5 1. 6 1. 2 1. 1 2 div.=.00	div. 28, 82 div. 32, 05	166. 5
inch. 83. 2 83. 2 83. 3 83. 2  Wrench-tig	ghtened belo	w, hand-t	14. 3 15. 1 16. 3 17. 5 19. 0 sightened al 26. 7 3. 7 2. 3 1. 8 1. 4 1. 1	Nuts still  43. 5 44. 0 45. 4 46. 3 47. 9 bove.  58. 3 36. 2 34. 8 34. 0 33. 2 33. 0 below; 27.7 57. 2	14. 9 15. 9 16. 5 17. 5 19. 0 27. 3 3. 7 2. 5 1. 6 1. 2 1. 1 2 div.=.00 25. 1	div. 28. 82 div. 32. 05	166. 5
inch.  83. 2 83. 2 83. 3 83. 2  Wrench-tig  Evening.  83. 1 83. 2	ghtened belo	w, hand-t	14. 3 15. 1 16. 3 17. 5 19. 0 sightened at 26. 7 3. 7 2. 3 1. 8 1. 4 1. 1 above and 1	Nuts still  43. 5 44. 0 45. 4 46. 3 47. 9 bove.  58. 3 36. 2 34. 8 34. 0 33. 2 33. 0 below; 27.7 57. 2 56. 0	14. 9 15. 9 16. 5 17. 5 19. 0 27. 3 3. 7 2. 5 1. 6 1. 2 1. 1 2 div.=.00 25. 1 23. 8	div. 28. 82 div. 32. 05	166. 5 185. 2
inch. 83. 2 83. 2 83. 3 83. 2  Wrench-tig	ghtened belo	w, hand-t	14. 3 15. 1 16. 3 17. 5 19. 0 sightened al 26. 7 3. 7 2. 3 1. 8 1. 4 1. 1	Nuts still  43. 5 44. 0 45. 4 46. 3 47. 9 bove.  58. 3 36. 2 34. 8 34. 0 33. 2 33. 0 below; 27.7 57. 2	14. 9 15. 9 16. 5 17. 5 19. 0 27. 3 3. 7 2. 5 1. 6 1. 2 1. 1 2 div.=.00 25. 1	div. 28. 82 div. 32. 05	166. 5

Nuts wrench-tightened below, also on hind leg above, but the two front legs hand-tightene Feet tightened very slightly. 27.72 div. of scale=.001 inch.

Wt. on. Wt. on. Mean diff.  $f_{\rm o}$ 

	. 003 inch.	Wt. on.	Wt. off.	Wt. on.	Mean diff.	$f_{\alpha}$ .	
		11.5	36. 7	11.5			
		10.4	35.2	10.0	div.	μ	
		10.1	34.9	9, 9	25.08	145.0	
		9. 0	34.2	9.2			
		8. 7	33, 9	8. 7			
Again.	Wrench-tighte	ened above	and below	; 27.58 div	of scale=.0	001 inch.	
	82. 6	10.1	41.4	9. 7			
	82. 9	9.8	41.1	9, 8	div.	μ	
	82. 7	9.0	40.3	9.2	31, 20	181.2	
		9, 2	40.2	9.2			
		9, 0	40.1	9. 2			
Again.	Binding-screw	s of the th	iree feet qui	te loose.	27.58 div. of	scale=.001 i	neh.
		6.3	39.2	7.1			
		6.3	38.2	6.3	div.	**	
		5.0	36, 9	4.4	32. 24	$187.\overset{\circ}{3}$	
		4.0	36. 1	4.0			
		3.7	36. 0	3.3			
Binding	screws tight as	possible.					
	C	29. 9	59.8	29. 7			
		29. 1	59. 0	28. 5	4.		
		27. 6	57. 4	27. 8	div. 20. 04	173.9	
		$\frac{27.3}{27.3}$	57.1	26. 9	29. 94	140. 9	
		26.5	<b>56.</b> 3	26. 5			
Weight	of 2.7 kilos put			20. 0			
" OIGH	or 2.1 knos put			41 -			
		12.4	42.1	11.5	div.	173. 4	
		9.2	38. 9	9. 0	29. 84	173. 4	
		8, 0	37. 5	7.2			
		6.0	35. 3	5. 2			
es . 1. =0	an =	3, 8	33. 4	3.4			
Scale 50	cm.7 in front o			ightened.	Same weigh	it on top of	stand; 27.83
	83. 6	20. 9	81. 0	19. 7			
	83.3	13.8	74. 5	13. 6	div.	μ	
	83, 6	11.0	71. 7	10, 6	<b>60, 6</b> 6	349, 2	
		6. 4	66. 4	5, 2			
***	• • • • • • • • • • • • • • • • • • • •	1.3	61. 4	0, 9			
Weight	removed from t	op of stan			above; 27.83	div.=.001 ii	ich.
		32.0	92. 4	31. 4			
		26.8	87. 4	25.8	div.	μ	
		23.1	83. 7	22, 3	<b>60.</b> 90	350, 6	
		13.3	73. 8	12. 7			
		10.8	71. 5	10, 4			
Binding-	screws loosened			=.001 inch.			
	84, 6	30.0	95. 1	28.2			
	84. 0	27. 2	$92.\ 5$	26, 2			
	84. 3	23. 6	88. 7	22.4	div.	μ.	
	84. 2	19.8	84. 9	18.6	6 <b>5. 6</b> 6	374. 6	
	e	17. 1	82. 3	16.5			
		<b>15.</b> 1	80. 4	14. 5			
~ ~-		13.6	78.8	13. 4			
S. Ex.	49——51						

Binding screws moderately tightened (about as in earlier experiments).

Wt. on.	Wt. off.	Wt. on.	Mean diff.	$f_0$ .
$32.\ 5$	<b>93.</b> 2	31. 9		
31.4	92. 0	30.8		
30.4	91.0	30.2	div.	μ
28.1	88. 9	27.7	61.03	348.1
<b>25.</b> 8	86.7			
27.0	88.4	26.8		
26. 7	87.7	26. 5		

Nuts at top of two front legs hand-tightened; 28.11 div. of seale=.001 inch.

.003 inch.	Wt. on.	Wt. off.	Wt. on.	Mean diff.	$f_0$ .
84.0	15.4	88.6	15.2		
84.7	13.4	87.2	13.6	div.	щ
84.3	16.5	90.2	16. 1	73. 76	420.5
84.3	15. 7	88.7	14.7		
	13.6	87.8	13. 2		

Note.—The weight put on stand was a very heavy paper weight. Observer troubled all day by tremor; every passer by on the street, every one entering the building, or even the adjoining building, agitates the apparatus so as to make it impossible to read the scale.

Summary of observations with weight and pulley, on Repsold stand, at York.

	Nuts hand-tightened above and below.		Nuts wrenched above and below.	Binding-screws extra tight.	Binding-screws loose.	Weight on.
56cm.4 forward,	368 4, 8	351 +. 9	350 #. 7	3 <b>5</b> 0 %. 6	374 +.6	$349$ $^{\mu}$ . $2$
50cm.1 behind middle j	ooint, 185 ". 4	185 4.2	173 4, 9	173 4. 9	187 +.3	173 4.4
50cm.1 behind middle 1	ooint, 271 + 4	263 4.6	256 <sup>µ</sup> , 9	256 ±, 8	275 4.2	255 4.9
$\mathbf{A}$ ,	$158^{\mathrm{em}}$	$168^{\mathrm{cm}}$	$156^{\mathrm{em}}$	$156^{\mathrm{em}}$	$147^{em}$	$156^{\mathrm{cm}}$
$\mathbf{F}_0: \mathbf{A}_{\bullet}$	1.72	1, 57	1, 65	1.65	1.75	1.65

1879, DECEMBER 23.

Evening. Still very tremulous; especially when vehicles pass, as they frequently do. Scale 56cm.6 in front. All tightly wrenched up. No weight on top of stand. Measures all very uncertain. 27.89 div. of scale=.001 inch. Dynamical flexure.

.003 inch.	Arc.	Scale 1	readings.	Diff.	$f_{lpha}$
83, 5	. 0461	4. 2	28.8	24.6	$34\overset{\circ}{8}$
83.8	450	5. 7	29, 2	23.5	340
83.7	433	5.2	27.8	<b>22.</b> 6	340
	427	5.3	27.7	22.4	342
	420	6, 0	27.5	21.5	333
	393	6, 5	27.0	20.5	340
	377	6. 2	26. 1	19.9	344
	368	5. 3	24.7	19. 4	343
	353	7.3	25.8	18, 5	341
	347	7.3	<b>25.</b> 6	18.3	<b>34</b> 2
	. 323	7. 6	24.4	16.8	339
					342. 4

Statical flexure, with same arrangement.

	Scale rea	dings.		
Arc.	Zero.	Out.	Mean diff.	$f_0$ .
. 0500	3.2	16.6		
	3.7	17.0		
	2. 9	16.2		
	27.2	39, 9		μ
	25.9	38. 7	13.08	340, 4
	26.3	39. 0		
	24.9	38.4		
	24.8	38. 0		
	26.4	39, 2		

1879, DECEMBER 25.

Morning. Arrangement same as in last observation. Scale 56cm,6 in front of middle of knifeedge; 25.67

ge; 25.67 div. of scale=.00		Statical flex		Scare 30cm.0 1	n front of
		Scale re	eadings.		
. 003 inch.	Arc.	Zero.	Out.	Mean diff.	fu.
77. 2	. 0500	22, 3	23.8		
77. 0		21.3	32.2		
76. 9		22.2	<b>33.</b> 8		
76, 9		20.8	32.1	div.	и
		21.4	32. 9	11, 53	326.4
		20, 6	32.5		
		20, 7	32. 5		
		27.4	39.0		
		20. 5	32. 2		
		20.8	32. 3		•
Dynamical flexure, with	same arra	angement.			
	Are.	Scale re	adings.	Diff.	$f_0$ .
	0485	17. 7	39. 4	21. 7	$3\overset{\mu}{16}$
•	478	17. 1	38.8	21.7	321
	440	<b>17.</b> 3	38. 4	21. 1	339
	431	19. 2	39.0	19.8	325
	325	19. 7	<b>33.</b> 3	13.6	296
•	320	<b>19.</b> 8	34.8	<b>15.</b> 0	332
	315	<b>20.</b> 8	<b>35.</b> 1	<b>14.</b> 3	321
	269	5.4	17.4	12.0	315
•	506	20.8	<b>44.</b> 8	24. 0	335
•	502	22. 3	<b>45</b> . 8	23. 5	331
	500	22, 6	45.1	22.5	318
	497	20. 7	43. 9	23.2	330
	494	21. 9	44.4	22, 5	322
	472	22, 2	43.4	21.2	317
	469	22, 0	43. 9	21, 9	330

465

462

**45**8

21.7

22.8

23.0

323.5

327

318

318

21.5

20.8

20.6

43. 2

43.6

43.6

1880, JANUARY 4.

Same arrangement and position as in last observed;  $25.72 \, \mathrm{div.}$  of scale=.001 inch; statical flexure.

D	7 .		•	
N.C.a	le:	read	1	n (18.

. 300 inch.	Arc.	Zero.	Out,	Mean diff.	$f_{o}$ .
77.3	.0500	21.1	9.8	div.	μ.
77. 1		21.2	10, 0	11, 43	323.9
77. 1		20. 1	8. 4		
•		15.0	3.5		
Statical flexure again.	Scale 50cm.	5 behind n	riddle of k	nife-edge.	
	. 0500	16.5	22.3		
		16.3	22, 3		
		16.4	22. 5	div.	μ
		19.0	25. 2	<b>6.0</b> 9	$172. \ 4$
		18.8	24.8		
		18.8	24. 9		
		19.1	<b>25. 4</b>		

18. 9

Dynamical flexure, with same arrangement.

Arc.	Scale readings.		Diff.	$f_{0}$ .
. 0427	22, 6	13.9	8.7	$1\overset{\prime\prime}{4}5$
419	99. 4	90.3	9. 1	154
386	23.8	15. 7	8.1	149
360	23.8	<b>16.</b> 0	7.8	154
357	23.7	16. 0	7. 7	153
351	74. 5	66.8	7. 7	155
348	74.3	66, 8	7. 5	153
480	<b>75.</b> 3	64. 9	10. 4	154
474	75. 1	64. 7	10. 4	$15\overline{0}$
463	74.8	65. 0	9.8	151
455	100.0	90. 3	9. 7	151
448	100. 2	90.8	9. 4	149
441	74.2	65. 1	9. 1	146
438	74. 0	64. 9	9. 1	148
433	73. 6	64. 6	9, 0	148
			•	<del>150.</del> 6

**25.** 1

Two front nuts at top hand-tightened; 25.64 div. of scale=.001 inch. Statical flexure.

### Scale readings.

, 003 inch.	Arc.	Zero.	Out.	Mean diff.	$f_{0}$ .
77. 0	. 0500	18.0	22. 3		
76. 9		18.0	22, 2	div.	μ
76. 9		19. 1	23.5	4. 30	121.7
		71. 2	75. 4		
		19.0	23, 4		

Dynamical flexure, with same arrangement.

Arc.	Scale readings.		Diff.	$f_0$ .
. 0509	22, 8	15. 6	7. 2	$100^{\mu}$
500	23.0	15.5	7.5	106
489	22.3	15.3	7. 0	102
484	22. 4	15.3	7.1	104
479	48. 4	<b>41.</b> 3	7.1	105
467	48.2	41.3	6.9	105
460	48.1	41.2	6. 9	106
				104 1

Stand wrench tightened above, two front feet loosened below; 25.54 div. of scale=.001 inch. Statical flexure.

		Scale red	adings.		
.003 inch.	Arc. . 0500	Zero. 21. 1	Out. 27. 7	Mean diff.	$f_0$ .
<b>76.</b> 6		21.0	27.6		
76. 7		21.0	27.8	div.	μ
		72.9	80. 0	6.81	193.0
		73.0	79.8		
		21. 2	28. 1		
		21.0	27.9		

Dynamical flexure, with same arrangement. •

Arc.	Scale readings.		Diff.	$f_0$
. 0496	48, 2	35, 8	12.4	177
469	24. 3	12. 4	11, 9	180
466	24.7	13, 0	11.7	178
457	24. 7	13.0	11.7	181
<b>450</b>	25, 0	13.8	11. 2	176
442	25. 1	14.2	10.9	175
437	25.2	14. 4	10.8	175
427	25. 4	14.8	10, 6	176
	•			177. 1

All wrench-tightened. Weight of  $2{,}700^{\rm g}$  put on top of stand; otherwise same as preceding. Statical flexure.

Scale readings.				
Arc.	Zero.	Out.	Mean diff.	$f_{\alpha}$ .
. 0500	3.8	10.0		
	30.3	36. 5		
	31.2	36.8		
	6. 2	12.3	div.	μ
	6. 9	12.6	5, 82	164.9
	7. 5	13.0		
	33.6	39, 2	•	
	8.4	14, 2		•
	8.6	<b>14.</b> 3		

Dynamical flexure, with same arrangement.

Arc.	Scale r	Scale readings.		$f_0$ .	
. 0500	14.8	3.8	11.0	156	
497	40.3	29.8	10. 5	149	
491	40.5	30.2	10.3	149	
484	40. 2	30.3	9. 9	145	
<b>465</b>	14.7	5.0	9. 7	148	
460	40.2	30.6	9, 6	148	
451	14.4	4. 9	9. 5	149	
448	14.4	5.1	9. 3	147	
444	39.8	30, 9	8. 9	142	
				148. 1	

Evening. Scale 56cm.8 in front of middle of knife-edge. Statical flexure.

#### Scale readings.

Arc.	Zero.	Out.	Mean diff.	$f_0$ .
. 0500	** 15. 2	3.6		
	<b>15.</b> 3	3 <b>.</b> 6		
	16.0	4.0	điv,	μ
	15.8	4.0	11, 75	332.7
	15.8	3, 9		
•	67. 3	<b>55.</b> 9		
	67.6	<b>55.</b> 8		•
	67. 7	55.9		

Dynamical flexure; with same arrangement.

Arc.	Scale re	adings.	Diff.	for $\mu$
. 0501	29. 2	5. 4	23.8	336
496	28.6	5. 7	22.9	327
486	28. 1	5. 4	22. 7	331
475	27.8	<b>5.</b> 8	22. 0	328
471	27.8	6.0	21.8	328
<b>469</b>	27.7	5.9	21.8	329
<b>462</b>	27.6	6. 1	21. 5	32)
454	26. 7	5.8	20. 9	326
423	26. 6	6, 9	19. 7	330
416	26. 3	7.0	19. 3	329
411	26. 0	6. 9	19. 1	329
				329. 1

Weight taken off from top of stand. Two front feet loosened; 25.63 div. of scale=.001 inch. Statical.

Scale readings.

.003 inch.	Arc.	Zero.	Out.	Mean diff.	$f_0$ .
<b>76.</b> 5	. 0500	23. 6	11.3		
77. 0		23. 5	11.2	div.	μ
76. 9		23.5	11.5	12.17	344.6
		23.4	11.3		
		23.2	10.9		
		22.7	10.7	•	4

Dynamical flexure; with same arrangement.

Arc.	Scale readings.		Di <del>f</del> î.	for
. 0500	34.0	9.8	24. 2	$343^{\mu}$
496	34.7	10. 5	24.2	346
487	34.8	11. 2	23. 6	343
480	34. 7	11.4	23.3	<b>34</b> 3
<b>475</b>	33.8	10. 9	22.9	341
469	33, 8	11.2	22.6	341
456	33.7	11.3	22.4	348
452	33, 6	11, 3	22, 3	349
				344 3

Feet of stand tightened, and two front legs hand-tightened above. Statical flexure; 25.54 div. of scale=.001 inch.

	Scale readings.				
. 300 inch. 76. 6	Arc. . 0500	Zero. 29. 8	Out. 15. 3	Mean diff.	1
76. 7 76. 6		32, 1 32, 3	17. 7 18. 0	div. 14, 34	μ 406, 0
		31.8	17.4		
Dynamical flexure; wit	h same arr	32, 7 angement.	18, 6		

Arc. Scale		readings.	Diff.	$f_{w}$	
. 0512	47. 0	18.2	28.8	398	
506	47. 7	19.0	28.7	401	
500	47.7	19. 7	28. 0	396	
488	47.5	20.1	27.4	397	
475	46.6	20.3	<b>26.</b> 3	392	
468	47.0	20.7	26. 3	398	
463	46. 7	20.8	25. 9	396	
451	46.7	21.4	25.3	397	
443	47.7	22. 7	25, 0	399	
	*			${397.4}$	

Three thicknesses of blotting-paper put under each foot of pendulum stands. All nuts tight Otherwise same as preceding. Statical flexure.

	count rea	wings.		
Arc.	Zero.	Out.	Mean diff.	$f_0$ .
. 0500	21.8	8.7		
	22.0	8, 5		
	22.1	8.6		
	19. 3	6, 6	div.	μ
	21.5	8.1	13, 36	379.6
	21.7	7.9		
	21.1	7.4		
-	22. 3	8.8		
	21.3	8.3		•

Dynamical flexure; with same arrangement...

Arc.	Scale re	adings.	Diff.	$f_0$ .
0, 509	27. 7	1.3	26.4	367
494	53.9	28. 7	25, 2	361
<b>482</b>	54. 0	29.2	24.8	365
477	53.7	29.4	24.3	360
461	53, 3	29.7	23.6	362
457	53. 7	30.0	23. 7	367
446	52.9	30, 2	22.7	360
435	53, 2	30. 7	22.5	366
				${365.0}$

Scale 50cm.5 behind knife edge, with blotting-paper arrangement, etc., as above. Statical flexure; 25.62 div.=.001 inch.

		Scale reading	J×.	
Arc.	Zero.	Out.	Mean diff.	$f_{\alpha}$ .
. 0500	17, 3	24.8		
	16. 7	24.9		
	16.9	25, 3		
	16.6	24.7		
	16.4	24.6	div.	μ
	15.8	24.3	8.28	234. 4
	15.4	24. 1		
	<b>15.</b> 3	23, 9		
	15. 1	23.4		
	14.7	23. 0		

Dynamical flexure, with same arrangement.

Arc.	Scale re	adings.	Diff.	$f_{m_{\mu}}$
. 0478	19. 2	5. 0	14. 2	210
469	18.8	5.2	13.6	205
460	18.8	5.4	13 <b>. 4</b>	206
447	18.7	6.0	12.7	201
439	18.6	5, 8	12.8	207
434	18.4	5.7	12. 7	207
428	17.8	5,3	12, 5	207
418	18.5	5.2	12.3	208
409	16.7	5, 0	11.7	202
400	16. 3	4.8	11.5	204
394	16. 2	4.7	11.5	207
				-

205.9

1880, JANUARY 11.

Flexure in third position, about 52cm.7 in front of knife-edge and 118cm.5 below. All clamped. No weight on stand. Statical flexure. 84.4 div.=.01 inch.

		Scale reading	8.	
Arc.	Zero.	. Out.	Mean diff.	$f_0$ .
. 0500	72.3	66. 6		
	72.8	66.3		
	71.4	66.2	div.	μ
	71.5	65. 6	<b>5.</b> 87	506
	71.2	65. 1		
	71.0	65. 1		
	70.8	65, 0		

Dynamical flexure, with same arrangement.

Arc.	Scale re	eadings.	Diff.	$f_0$ .
. 0484	62.8	73.3	10. 5	467
<b>476</b>	63.9	74. 9	11.0	498
470	64. 1	74.7	10.6	487
466	63.3	73.7	10.4	481
461	63.2	73. 7	10.5	491
454	<b>63.</b> 8	74.0	10.2	485
				484

All tight. Weight of 2,700s on top of stand; otherwise the same. 84.4 div. of scale=.01 inch Statical.

		Scale readings.				
	. 001 inch.	Arc.	Zero.	Out.	Mean diff.	$f_0$ .
	84.7	. 0500	93.6	87.3		
	84, 3		93.0	86.6	div.	μ
,	84. 2		92.5	86.4	6.26	540
	84. 4		92.5	86.3		
			92.4	86.1		
Dynami	cal flexure, with	n same arra	ngement.			
•		Arc.	Scale r	eadings.	Diff.	$f_0$ .
		. 0498	85. 2	98.8	13.6	<b>5</b> 88
		474	86.3	98.0	11.7	532
		469	86.0	98.0	12.0	552
*		462	86.0	98. 0	12.0	560
	•	<b>442</b>	83.0	94.7	11.7	571
		419	86.3	96, 8	10, 5	541
		409	85.3	95.8	10, 5	554
		401	<b>85.</b> 8	96. 0	10.2	547
•						556
Hand-tig	ghtened above;	otherwise	the same.	Statical.		
			Scale re	adings.		
		Arc.	Zero.	Out.	Mean dift.	$f_0$ .
		. 0500	75.2	68. 7		
•			<b>75.</b> 0	68. 1		
			75. 1	68. 0	div.	μ
			80.1	73.7	6. 79	585

Dynamical flexure, with same arrangement.

same and	mgcmcm.			
Arc.	Scale r	eadings.	Diff.	$f_0$ .
. 0424	25.8	38. 7	12. 9	655
417	25.4	38.8	13, 4	692
413	25.3	38.6	13. 3	699
408	25.3	38.1	<b>12.</b> 8	677
400	25. 7	37.0	11. 3	609
388	<b>26.</b> 3	<b>37.</b> 2	10.9	606
379	26. 4	37.9	11. 5	653

77.4

77.7

84.3

656

Statical flexure. Feet unclamped; otherwise same.

	Scale rec	adinys.		
Arc.	Zero.	Out.	Mean diff.	$f_0$ .
. 0500	48.1	41.7		•
	47.4	41.7	div.	μ
	47.4	41.5	6.02	519
	47. 9	41.8		
	47. 7	41.7		
Dynamical flexure, with same an	rangement.			
· Are.	Scale re	Scale readings.		$f_0$ .
. 0414	38. 2	49.6	11.4	593
405	38, 2	48.9	10.7	569
398	37. 7	48.6	10.9	590
389	38.6	48.7	10.1	560
377	<b>38.</b> 6	48. 9	10.3	588
				<del>580</del>

Statical flexure, with blotting-paper under feet; otherwise same; 84.16 div.=.01 inch.

		Scale rec	adings,		
.01 inch. 84. 4	Arc. . 0500	Zero. 73, 2	Out. 65, 8	Mean diff.	$f_0$ .
84. 1	. 0000	72. 9	66. 2	div.	۰ ,
84. 2		73.1	66. 0	7. 00	603
84.1		73.3	66. 1		
84. 0		72.7	66. 1		

Dynamical flexure; with same arrangement.

Arc.	Scale 1	eadings.	Diff.	$f_0$ .
0369	67.8	78. 1	10.3	601
361	67. 7	77. 9	10. 2	610
360	68. 0	77.2	9. 2	552
340	66.8	75. 8	9. 0	571
338	67. 0	76.3	9. 3	582
333	67. 0	<b>76.</b> 0	9. 0	552
328	67.8	<b>76.</b> 2	8. 4	
			•	583

1880, JANUARY 21.

Third position; about 53cm in front of middle of knife-edge, and 120cm below; 84.72 div. of scale=.01 inch. Statical flexure.

	. 01 inch.	Arc.	Scale 1	readings.	Diff.	$f_0$ .
	84.7	. 0500	81. 9	88.7		
	84.3		82. 3	88.6	div.	
	84.6		82. 0	88.4	6.38	546. 4
	85.0		82, 9	88. 9		
	85.0		82.4	88.8		
Dynamic	al flexure; wit	h same arra	ingement.			
		. 0507	82.2	94. 9	12.7	537 <sup>"</sup>
		500	82. 2	96.0	13.8	591
		496	82.3	95.2	12.9	556
		488	82.8	94.8	12.0	526
		478	82.8	94.8	12.0	537
		464	82. 7	94. 7	12. 0	554

550.8

Same as above, except that a weight of 25 pounds is put on top of stand. Statical flexure.

	Arc.	Scale readings.		Diff.	$f_0$ .	
	. 0500	35.5	41.7			
		35.9	41.8	•		
		35.5	41.6	div.	μ	
		34.8	41.3	6. 15	526.9	
		35.0	41.1			
		34.9	41.0			
Dynamical flexure; w	ith same arra	angement.				
•	. 0488	34.8	47.1	12. 3	539	
	<b>481</b>	35.2	46.8	11.6	516	
	476	35. 0	46.8	11.8	531	
	468	35.0	46.9	11.9	544	
	459	34.3	45, 8	11.5	537	
•	439	35.0	45. 4	10. 4	507	
	•				${529.5}$	

1880, JANUARY 23.

Second position;  $50^{\rm cm}.8$  behind knife-edge. No weight on; 26.27 div. = .001 inch. Statical flexure.

	.003 inch.	Arc.	Scale r	eadings.	Diff.	$f_{0}$ .
	<b>78.</b> 9	. 0500	4. 7	10.6		
	79. 0		3.0	7.2		
	78. 7		9.6	16.0		
	78.7		24.7	32. 0		
			25. 1	31, 8		
•			26.5	33. 7	div.	μ.
			6. 0	12.4	6. 23	172.1
			11.0	17. 9		
			13.2	19. 0		
			1.3	6. 8		
			21.3	27.9		
			22.4	28. 3		
			28. 3	34. 3		
			25. 1	30. 9		
			23.2	30, 0		
Dynamical f	dexure; wit	h same arr	angement.			
		. 0329	12. 7	19.1	6. 4	134
		.399	12.4	19. 4	7, 0	121
		304	18. 4	24. 7	6. 3	143
		298	17. 1	24, 3	7.2	166
		259	<b>23.</b> 6	<b>29.</b> 8	6. 2	165
		389	23. 8	33. 7	9, 9	175 .
		338	23, 4	32.3	8.9	181
		319	27.3	35.4	8.1	175
		294	2.5	9.4	6. 9	162
		289	3. 2	9. 2	6.0	143
		278	4.2	10, 4	6. 2	154
		. 276	3.5	10.2	. 6. 7	168
		265	3. 9	10.0	6. 1	159
		259	5. 2	11. 3	6.1	162
•				*		${157.6}$

The mean of the last swinging is 164\*.3.

1880, JANUARY 25.

Second position; 25 pounds weight on stand. Tube lengthened;  $27.44 \, \text{div.}$  of scale= $.001 \, \text{inch}$  Statical flexure.

aucai nexure.					
	Arc.		readings.	Diff.	$f_0$ .
	. 0500	8. 4	2, 3		
		7.4	1.4		
		8. 3	1. 7		
		8.3	1.8	di <b>v</b> ,	. μ
		7. 2	0.1	6.57	173.8
		9. 9	3. 2		
		9. 6	3.2		
		10.0	3.3		
		11.3	4. 5		,
		11. 5	4.7		
Dynamical flexure; wi	th same arr				
	. 0478	7.7	20.6	12.9	178
	<b>467</b>	8.8	20.8	12.0	170
	455	9.4	20. 9	11.5	167
	439	10.4	21. 2	10.8	163
	<b>432</b>	10.2	21.2	11.0	169
•	$\boldsymbol{422}$	10, 2	20. 7	10.5	165
	416	10.3	20.6	10.3	164
	392	11.8	$21.\ 5$	9. 7	163
	384	12.0	22. 2	10.2	176
					${168.3}$
Weight taken off; other	erwise same	. Statical	flexure.		
. 003 inch.	Arc.	Scale re	eadings.	Diff.	$f_{7}.$
82. 4	. 0500	14.1	7.3		
82. 8		14. 5	8.3		
82. 5		<b>15.</b> 2	8.7		
82.3		16.7	9. 6	div.	μ
82. 1		17.0	10.4	6. 59	174.3
82. 2		16.9	10.4		
82. 2		23.4	16. 7		
82. 1		23.1	16.8		
Dynamical flexure, with	h same arra				•
	.0482	18.3	30. 9	12.6	173
	474	19. 2	31.6	12.4	173
	461	18.3	30.6	12.3	176
	457	18.3	29.8	11.5	167
	450	20.0	31.7	11.7	172
	444	19.8	31.1	11.3	168
	437	20. 2	31.2	11.0	167
	424	21.0	31.6	10.6	165
		<del></del>			170. 3

First position: about  $57^{\rm cm}.0$  in front of knife-edge; 26.85 div. of scale=.001 inch. Statical. No weight on.

0.500	15.5	3. <b>4</b>		
	15.8	4.0	điv.	
	16.5	4.5	12.07	326.3
	17. 1	4.9	•	* '
	17.8	5. 5		
	17. 6	<b>5.</b> 6		

Dynamical flexure	with sar	ne arrangement.
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*		ac arrante	semont.			
	.003 inch.	Arc.	Scale r	eadings.	Diff.	$f_0$ .
80.	7 80.3	. 0481	11.3	34. 2	22, 9	$322^{\mu}$
80.	5 80.6	<b>47</b> 3	<b>1</b> 1. 3	33.8	22.5	322
80.	8 80.9	467	12.0	34.0	22. 0	318
- 80.	1 80.6	459	11.8	34.0	22. 2	327
		448	12.2	33. 9	21. 7	327
		441	<b>13.</b> 3	34. 4	21. 1	324
		437	13.4	34, 2	20.8	322
						${322.9}$
Weight on sta	nd. Statical	flexure.	Otherwise sa	ame.		
		. 0500	20.0	<b>32.</b> 3		
			21.8	34. 1	div.	,,
			$22.\ 5$	<b>34.</b> 6	12. 17	329. 0
			22.7	35. 1		
			23. 4	<b>35. 4</b>		
			23. 7	35. 6		
Dynamical flex	cure, with sam	e arrang	ement.			
		. 0483	10.8	34. 1	23. 3	$326^{^{\mu}}$
•		467	11.3	33.8	22.5	326
		<b>458</b>	10. 9	32.4	21.5	317
		446	11.7	33. 3	21.6	327
		438	11. 7	32. 7	21.0	324
		429	12. 7	33.6	20.9	329
		419	12. 6	32. 7	20.1	324
	-	406	13.0	32. 4	19.4	323
	•					${324.6}$

Summary of dynamical and statical observations, with pendulum on Repsold stand, made at York.

## All tight; no load.

56 <sup>cm</sup> .6 forward, 50 <sup>cm</sup> .5 back, 52 <sup>cm</sup> .7 forward, 118 <sup>cm</sup> .5 below,	$f_0$ $f_0$ $f_0$	Statical. 326 **. 4 172 **. 4 504 **. 4	Dynamical. 323 #, 5 150 #, 6 482 #, 9
Inclination of axis to knife-edge,		420.9	490.0
Distance of axis from middle of knife-edge.		1 <sup>m</sup> . 159	$1^{\rm m}$ . $086$
$f_0$ at 1 cm. from axis,		2 p. 114	2 m. 137
	$\mathbf{F}_{0}$	245 °, $0$	$232$ $\mu$ . $2$
The same arrangemen	nt.		
57cm.0 forward,	$f_0$	326 <b>4.</b> 3	322 *. 9
50°m.8 back,	$f_0$	$173  \mu.  2$	<b>167 4.</b> 3
53cm.0 forward, 120cm below,	$f_0$	546 <b>+.</b> 4	550 r. 8
Inclination of axis to knife-edge,		370.0	<b>36</b> 0. 5
Distance of axis from middle of knife-edge,		1 <sup>m</sup> . 040	0 <sup>m</sup> . 993
fo at 1 cm. from axis,		2 <b>4.</b> 359	$2\mu$ . $424$
	$\mathbf{F}_{0}$	245 °. 3	240 <sup>4</sup> . 6

## All tight; load of 2k.7.

Au tight; waa of 22.	1.		
		Statical.	Dynamical.
56cm.6 forward,	$f_0$	332 <b>⊬.</b> 7	329 +. 1
50cm.5 back,	$f_{\mathfrak{o}}$	164 <sup>µ</sup> , 9	148 4.1
52cm.7 forward, 118cm.5 below,	$f_0$		556 #. 0
Inclination of axis to knife-edge,		410.1	400.6
Distance of axis from middle of knife-edge,		1 <sup>m</sup> . 023	0 <sup>m</sup> , 899
$f_0$ at 1 cm. from axis.		2 4. 385	2 4. 597
	$\mathbf{F}_0$	$244 ^{\mu}.  0$	233 ×. 4
All tight; load of 11k	.3.		
57cm.0 forward,	$f_0$	329 m. 0	324 <b>4.</b> 6
50cm.8 back,	$f_0$	173 m. 8	168 m. 3
53°C.0 forward, 120°C below,	$f_0$		529 <sup>µ</sup> . 4
Inclination of axis to knife-edge,	• 0	400.3	390.5
Distance of axis from middle of knife edge,		1 <sup>m</sup> .009	$1^{\rm m}$ . 063
$f_0$ at 1 cm. from axis,		2 4. 226	2 4. 277
Jo we I cm. from and,	$\mathbf{F}_0$	246 <sup>\(\mu\)</sup> . 9	242 °. 0
Front taps above hand-tig	hter	ied.	
_			207 4 4
56cm.6 forward,	$f_0$		3 <b>97                                    </b>
50cm.5 back,	$f_0$	121 #. 7	μ. 1 656 μ. 0
52cm.7 forward, 118cm.5 below,	$f_{\scriptscriptstyle{0}}$	584 #. 9	
Inclination of axis to knife-edge,		580.9	50°, 3
Distance of axis from middle of knife-edge,		0 <sup>m</sup> , 850	0 <sup>m</sup> . 681
$f_0$ at 1 cm. from axis,	$\mathbf{F}_{0}$	3 #, 099 255 #, 7	3 +. 559 242 +. 4
	ΓO	299 F. 1	2427. I
Tight above; binding-screws of fr	ont	feet loose.	
56cm.6 forward,	$f_0$	344 <b>#.</b> 6	344 4. 3
50cm.5 back,	$f_0$		177 4. 1
52cm.7 forward, 118cm.5 below,	$f_0$		580 r. 3
Inclination of axis to knife-edge,		<b>4</b> 3°. 0	370.4
Distance of axis from middle of knife-edge,		1 <sup>m</sup> . 273	0 <sup>m</sup> . 995
$f_0$ at 1 cm. from axis,		2 4. 077	2*. 572
<b>30 20 2 0 20 20 0 20 0 0 0 0 0 0 0 0 0 0</b>	$\mathbf{F}_0$	$264$ $^{\mu}$ . $5$	255 µ. 9
. All tight; blotting paper und	er t	he feet.	
56 <sup>cm</sup> .6 forward,	$f_0$	379 +. 6	3 <b>6</b> 5
50°5 back,	$f_0$	2.5	205 4. 9
52cm.7 forward, 118cm.5 below,	$f_0$	603 ° . 4	580 r. 1
Inclination of axis to knife-edge,	90	35 . 0	380.7
Distance of axis from middle of knife edge,		1 <sup>m</sup> . 282	1 <sup>m</sup> . 179
$f_0$ at 1 cm. from axis,		2 <b>4.</b> 362	2 4. 384
JO WO I OM. HOM WAIDS	Fa	302 +. 9	281 4. 0

These measures are interesting as showing that while the dynamical flexure at the point of application of the force is constantly less than the statical, yet the angular flexure is less in the statical experiments than in the dynamical ones. This fact probably indicates the cause of the difference between the two kinds of flexure, namely, that in the dynamical experiments the flexure-wave has not had time to fully reach the distant parts of the apparatus.

## 1880, MARCH 17.

Top of Repsold stand fastened to an oak plank, which is bolted to top of Geneva tripod. Statical observations with weight (1<sup>k</sup>) and pulley; 76.3 div. of scale=.003 inch. Scale 56<sup>cm</sup>.8 in front of middle of knife-edge.

Scale readings.

	Doute remaining.				
	Wt. on.	Wt. off.	Mean diff.	$f_{ij}$ .	
	4.0	29.0			
	2.3	26.7	div.	μ	
	3, 9	28.3	24, 62	154.7	
	1.6	26.2			
	3.4	28. 1			
Scale 50cm.5 behind k	nife-edge;	76.7 div.=	.003 inch.		
	<b>6.</b> 1	0.3	•		
	1.9	6.7			
	21.3	25.3	div.	μ	
	25. 7	29.8	4. 42	28.9	
	23.0	27.6			
	21. 2	25.8			
	•	. 1880,	APRIL 5.		

Experiments on oak plank support as last used. (It has been removed and put on again since last experiment.) Statical flexure; with weight and pulley. Scale about 50cm.0 behind knife-edge. Filar micrometer, 1 revolution=100r.

1001 = 100	•		
	Micrometer re	eadings,	
Wt. on.	Wt. off.	Diff.	$f_0$ .
4. 186	4, 150	0, 036	
. 218	. 180	. 038	
.242	. 216	. 026	
.291	. 256	. 035	
4. 425	4.390	. 035	. μ
. 451	. 415	. 036	21.9
. 484	. <b>45</b> 2	. 032	
. 523	. 492	. 031	
4. 566	4.528	. 038	
. 595	. 562	. 033	
. 625	. 591	. 034	
. 650	, 611	. 039	
		035	

Scale about 56cm.7 in front of knife-edge; otherwise the same.

3. 200	3,022	0.178	
4. 427	4.247	. 180	
. 405	. 216	. 189	
. 385	.215	. 170	
. 368	. 205	. 163	μ
.345	. 175	. 170	109.1
, 269	.092	.177	
. 231	.064	. 167	
. 236	. 178	. 178	
. 236	. 165	. 165	
. 221	. 178	. 178	
. 210	. 177	. 177	
, 203	. 157	. 157	
		${0.173}$	

Scale about 44<sup>cm</sup> in front, and about 120<sup>cm</sup> below the middle point of knife-edge; otherwise same as before. The flexure is now in the reverse direction from the pull.

	Micrometer	readings.	
Wt. on.	Wt. off.	Diff.	$f_0$ .
3. 735	3.308	0.427	
. 760	. 597	. 163	
.622	. 424	. 198	•
. 808	. 343	. 465	
4. 121	. 624	. 497	
3. 740	. 270	. 470	
$\cdot 448$	. 160	. 288	
. 926	. 277	649	
. 676	. 189	. 487	
. 759	. 386	. 373	
. 6:0	. 480	. 180	μ
.585	. 378	. 207	229.6
4. 785	4.550	. 235	•
5. 913	<b>5.66</b> 8	$.\ 245$	
873	. 384	. 489	
. 639	. 545	. 094	
. 438	. 186	.252	
. 306	4. 567	. 739	
. 073	. 910	. 163	
. 037	. 831	. 206	
. 066	. 661	. 405	
. 112	. 640	. 472	
. 113	. 652	. 461	
4. 943	. 714	. 229	
5. 158	. 607	.551	
, 103	. 756	. 347	
. 187	. 646	. 541	
		0. 364	

1880, APRIL 17.

Repsold stand supported on oak blocks, six inches high, and braced together. Microscope as in former experiments. Scale 56cm.7 in front of middle of knife-edge; 21.93 div. of scale=.001 inch. Statical flexure with pendulum.

.004 i	nch.	Arc.	Scale	readings.	Diff.	$f_{0}$ .
87.7	87.5	. 0500	5.7	<b>24.</b> 0		
87.1	87.9		22.6	40.9		
86.8	88.3		18.9	<b>37.</b> 9		
87.6	87.7		10.9	30.9		
87.9	88. 0		3.5	21.7	div.	. μ
87.6			2.2	20.3	18. 72	619.6
88.0	***		21.8	41.2		*
<b>87.9</b>			17.3	36. 1		•
			15. 0	32.6		
			6.5	25. 2		
			4, 9	23. 7		
			17.8	37. 2		

Dynamical flexure, with same arrangement.

Arc.	Scale	readings.	Diff.	$f_0$ .
. 0494	42.7	9.6	33.1	55 <b>4</b>
490	41.8	7.5	34.3	579
484	40.7	7.0	33. 7	576
454	32.3	2.8	29. 5	538
448	32. 7	3.6	29. 1	538
438	32.4	3 <b>. 3</b>	29. 1	549
433	30. 1	1.2	28.9	552
				<del>55</del> 6
Scale 50cm.4 behind knife-edge.	. Other	wise same.	Statical.	
. 0500	22.3	31.7		
	3.8	13.4		
	5.3	<b>14.</b> 6		
	5.4	14.0		
	10.0	18. 5	div.	μ
	11.0	20. 3	9. 11	301.5
	18.3	<b>26.</b> 0		
	20.0	28. 9		
	22.5	31.4		
	3.4	12. 2		
	4.7	14. 4		
	8.1	17.8		
	10.0	19. 5		
	11. 0	20.6		
Dynamical flexure, with same a	ırrangem	ent.		
. 0489	28. 4	9.6	18.8	318
484	27.7	8.9	<b>18.</b> 8	321
477	<b>26. 2</b>	7.8	18.4	319
470	24.8	6. 9	17.9	296
465	23. 7	6. 7	17.0	303
				$\frac{-}{315}$

1880, APRIL 18.

Repsold stand raised up on pieces of rubber  $2\frac{1}{4}$  inches thick. Scale  $50^{\rm cm}$ .3 behind knife-edge;  $21^{\rm div}.94$  of scale=.001 inch. Statical flexure.

Arc.	Scale re	adings.	Diff.	$f_{t}$ .
.0500	13.8	42.1		
	16.4	39. 3 rej.		
	19.8	48. 7		
	20.9	48.6	div.	μ
	<b>2</b> 3. 7	<b>51.</b> 8	28.36	939. 2
	20.9	49.7		
	23. 1	51.6		
	26.0	54.2		

S. Ex. 49---53

Dynamical flex	cure, with same	arrangement.
- DVHAMICAI UCS	ante, while same	arrangement.

Dynamical nex	are, with sa	ime arrange	ment.			
		Arc.	Scale re	adings.	Diff.	$f_0$ . $\mu$
		. 0460	*49.6	2.4	47.2	849
		452	49.6	2, 8	46. 8	856
		447	49. 3	3. 9	45.4	840
		438	48. 7	4, 6	44.1	833
		426	49. 2	5, 3	43.9	853
•		<b>12</b> ()	20.2			
						846
$57^{\mathrm{cm}}.9$ in front $\epsilon$	of knife-edge	e; 21.9 div.	of scale=.0	01 inch.	Statical flex	ure.
		. 0500	36. 8	2, 8		
		. 0000	50. 0 52. 0	17.5		
			48.6	15. 7		
			45. 6	11. 9		
			43. 5	12. 3		
				8. 3	div. 33, 2	1099
			41.8 $40.8$	6. 7	.,0, 2	3 (71747
			38.6	6. 5		
				5.1		
			37. 2			
			<b>36.</b> 3 3 <b>5.</b> 4	$rac{2}{1}, rac{5}{9}$		
			00. ±	1,		
Dynamical flex	ure, with sa	me arrangei	ment.			
, 00	4 inch.	Arc.	Scale r	eadings.	Diff.	$f_{\alpha}$
88.0	87.7	. 0509	<b>75.</b> 8	14. 0	61.8	1008
88. 4	87.6	498	71.8	12, 7	59. 1	985
87. 9	87.3	487	70. 3	12.4	57.9	987
88.3	87. 2	482	68.4	11. 3	57. 1	984
88.1	87.3	473	66. 8	10.0	56.8	997
	,	467	<b>66.</b> 8	11.0	<b>55.</b> 8	992
	•	458	64. 4	11.2	53.2	964
						989
Third position;	81.1div. of	scale=0.1 i	neh. Stati	cal flexure		
•		. 0500	19. 0	26. 3		
		. 0000	18. 2	26. 3	div.	
			18. 7	26.0	7.9	709, 8
			18. 3	27. 1	,,,,	
			18, 3	26. 5		
Dynamical flex	una mith an					
•	ure, with sa	-			F2.1.40	
. 01 inch.		Arc.	Scale r	eadings.	Diff.	$f_{0}$ .
81, 2	•	, 0320	<b>15.</b> 8	23, 6	7.8	<b>545</b>
80. 9		304	. 15.8	22.8	7.0	514
81.2		288	15. 3	22. 3	7. 0	<b>54</b> 3
		278	15. 1	22. 2	7.1	570
		270	<b>15.</b> 2	21.8	6.6	545
		262	15. 2	21.6	6. 4	545
						<del></del> -
						<b>343</b>

<sup>\*</sup> The first figure in this column is everywhere recorded as 5; the observer, however, notes that it should be 4.

1880, APRIL 21.

Flexu	re observed again on rubber feet.	Scale 50cm.2 behind	knife-edge;	22.17 div. of	scale =
.001 inch.	Statical flexure.				

	. 004 inch.	Arc.	Scale re	adings.	Diff.	$f_{i.i.}$
	88.6	.0500	15.6	38. 9		
	88.8		13, 2	37.8		
	88. S		13.8	38.8		
	88. 6		18.8	43.5		μ
			21.0	44.5	24, 5	803, 8
			19.7	43. 7		
			19.3	44.9		
			18. 9	43.9		
			19.8	43.8		
			19. 2	44.1		
Dynan	nical flexure, wi	ith same arr	angement.			
		. 0472	45.8	5. 2	40.6	707
		463	48.7	8.4	40.3	715
		456	48.7	9, 5	39. 2	707
		450	48. 7	9.4	39, 3	718
		443	48, 0	9, 4	38, 6	716
						$\overline{713}$

 $57^{\rm em}$ .1 in front of knife-edge; 21.99 div. of scale=.001 inch. Statical flexure.

.004 inch.	Arc.	Scale r	eadings.	Diff.	$f_{\alpha}$ .
88. 7	. 0500	47. 9	18. 7	_	
88.6		50.0	20.0		
87. 6	-	48.8	18.2		
88.3		48.0	18.3		
87.5		46. 3	16. 6		
87. 8		44. 0	12.8	div.	μ
87.9		43.0	11. 1	:. <b>0. 1</b>	989.5
88. 0		41.2	12.4		
88. 0		38.7	7. 9		
88. 0		<b>35.</b> 5	6, 2		
		31.2	2.8		
		33. 7	2.4		
		31.1	0.6		

Dynamical flexure, with same arrangement.

				μ
.0495	61.7	6.3	55.4	921
488	61.2	6.8	54.4	918
480	58. 7	<b>5.</b> 3	53 <b>.</b> 4	915
476	59. 4	6.2	53.2	920
467	56. 7	4.0	52. 7	928
459	55, 8	4. 0	51.8	929
				922

Block feet again. Scale 56cm.9 forward of middle of knife-edge; 22.19 div. of scale=.001 inch Statical flexure first.

			Arc.	Scale re	adings.	Diff.	$f_0$ .
			. 0500	22.5	11.3		
				28.5	17. 3		
				31. 2	20, 2	div.	μ
				12. 3	1.1	11.14	365. 2
				14. 9	3.6		
				14.7	3.9		
				13.8	2.5		
·D				<b>13</b> . <b>6</b>	2.5		
Dynai	mical flexu	re, with sa	me.				μ
			. 0459	30. 5	10. 2	20.3	362
			<b>45</b> 0	36.8	17.3	19.5	355
			445	37. 2	17.7	19. 5	359
			439	<b>35.</b> 8	16.4	19.4	362
			430	36. 0	16. 7	19.3	368
			420	33. 7	15.4	18.3	358
			414	34.0	16. 2	17.8	353
				,	•		360
Scale 3	50 <sup>cm</sup> .1 behi	ind middle	point of kni	ife-edge; 2	2.08 div. of	scale = .001	inch. Statical.
	0.004 in		Arc.		eadings.	Diff.	$f_0$
	88. 7	88. 7	. 0500	21.1	31.0		
	88. 2	88.8	•	<b>18.</b> 6	28.8		
	88.7	88, 8		. 17.7	27.2	div.	μ
	87.9			<b>16.</b> 5	26. 4	10.10	331.2
	88. 2			<b>13.</b> 3	23.9		
	88. 2			12.4	22.5		
				11.3	21.8		
Anoth	er set.						
	v		. 0500	12.4	23. 6		
					20.0		
			. 0000		22.4	•• '	
			, 0000	11.4	22.4 21.3	div.	μ 954 1
			. 0000	11. 4 10. 6	21.3	div. 10.8	354. 1
				11. 4 10. 6 11. 5	21.3 $21.8$		354. 1
Dynan	nical flexur	re, with san	ne arrangen	11. 4 10. 6 11. 5 11. 5	21.3		354. 1
Dynan	nical flexur	re, with san	ne arrangen	11. 4 10. 6 11. 5 11. 5 nent.	21. 3 21. 8 22. 3	10.8	μ
Dynan	nical flexur	re, with san	ne arrangen . 0495	11. 4 10. 6 11. 5 11. 5 nent.	21. 3 21. 8 22. 3	10. 8 8. 3	386
Dynan	nical flexur	re, with san	ne arrangen . 0495 489	11. 4 10. 6 11. 5 11. 5 nent. 31. 3 31. 0	21. 3 21. 8 22. 3 8. 0 7. 4	8.3 23.6	386 396
Dynan	nical flexur	re, with san	ne arrangen . 0495 489 483	11. 4 10. 6 11. 5 11. 5 nent. 31. 3 31. 0 29. 7	21. 3 21. 8 22. 3 8. 0 7. 4 7. 3	8. 3 23. 6 23. 4	386 396 380
Dynan	nical flexur	re, with san	ne arrangen . 0495 489 483 478	11. 4 10. 6 11. 5 11. 5 nent. 31. 3 31. 0 29. 7 29. 6	21. 3 21. 8 22. 3 8. 0 7. 4 7. 3 6. 8	8. 3 23. 6 23. 4 22. 8	386 396 380 391
Dynan	nical flexur	re, with sar	ne arrangen . 0495 489 483 478 469	11. 4 10. 6 11. 5 11. 5 nent. 31. 3 31. 0 29. 7 29. 6 28. 8	21. 3 21. 8 22. 3 8. 0 7. 4 7. 3 6. 8 6. 4	8. 3 23. 6 23. 4 22. 8 22. 4	386 396 380 391 392
Dynan	nical flexur	re, with sar	ne arrangen . 0495 489 483 478	11. 4 10. 6 11. 5 11. 5 nent. 31. 3 31. 0 29. 7 29. 6	21. 3 21. 8 22. 3 8. 0 7. 4 7. 3 6. 8	8. 3 23. 6 23. 4 22. 8	386 396 380 391
Dynan	nical flexur	re, with sar	ne arrangen . 0495 489 483 478 469	11. 4 10. 6 11. 5 11. 5 nent. 31. 3 31. 0 29. 7 29. 6 28. 8	21. 3 21. 8 22. 3 8. 0 7. 4 7. 3 6. 8 6. 4	8. 3 23. 6 23. 4 22. 8 22. 4	386 396 380 391 392
			ne arrangen . 0495 489 483 478 469	11. 4 10. 6 11. 5 11. 5 nent. 31. 3 31. 0 29. 7 29. 6 28. 8 28. 7	21. 3 21. 8 22. 3 8. 0 7. 4 7. 3 6. 8 6. 4 6. 8	8. 3 23. 6 23. 4 22. 8 22. 4 21. 9	386 396 380 391 392 385
			ne arrangen . 0495 489 483 478 469 467	11. 4 10. 6 11. 5 11. 5 nent. 31. 3 31. 0 29. 7 29. 6 28. 8 28. 7	21. 3 21. 8 22. 3 8. 0 7. 4 7. 3 6. 8 6. 4 6. 8	8. 3 23. 6 23. 4 22. 8 22. 4 21. 9	386 396 380 391 392 385 388
			ne arrangen . 0495 489 483 478 469 467	11. 4 10. 6 11. 5 11. 5 nent. 31. 3 31. 0 29. 7 29. 6 28. 8 28. 7	21. 3 21. 8 22. 3 8. 0 7. 4 7. 3 6. 8 6. 4 6. 8	8. 3 23. 6 23. 4 22. 8 22. 4 21. 9	386 396 380 391 392 385 388
			ne arrangen . 0495 489 483 478 469 467	11. 4 10. 6 11. 5 11. 5 nent. 31. 3 31. 0 29. 7 29. 6 28. 8 28. 7 e; 22 div. 6 35. 9	21. 3 21. 8 22. 3 8. 0 7. 4 7. 3 6. 8 6. 4 6. 8	8. 3 23. 6 23. 4 22. 8 22. 4 21. 9 01 inch. Solution	386 396 380 391 392 385 388 tatical flexure.
			ne arrangen . 0495 489 483 478 469 467	11. 4 10. 6 11. 5 11. 5 nent. 31. 3 31. 0 29. 7 29. 6 28. 8 28. 7 e; 22 div. o 35. 9 18. 7	21. 3 21. 8 22. 3 8. 0 7. 4 7. 3 6. 8 6. 4 6. 8 of scale=.00 23. 9 8. 3	8. 3 23. 6 23. 4 22. 8 22. 4 21. 9	386 396 380 391 392 385 388

Dynamical flexure, with same.

Arc.	Scale re	adings.	Diff.	$f_{v}.$
. 0484	31.9	11.7	20, 2	342
476	<b>3</b> 3. 6	<b>13.</b> 8	19.8	341
469	<b>34.</b> 3	14, 8	19.5	341
460	36.0	16, 8	19, 2	342
456	36, 8	17.6	19.2	345
				349

1880, APRIL 25.

Observations on wooden stand. Much troubled by tremor, probably due in great measure to the irregular heating of the wooden blocks by the illuminating flame. Third position=55<sup>em</sup> in front of knife-edge and 118<sup>em</sup> below; 71.72 div. of scale=.01 inch. Statical flexure.

Arc.	Scale re	adings.	Diff.	$f_{0}$ .
. 0500	32.8	36. 5 rej.		
	25. 0	<b>30.</b> 3		
	24.8	30, 2		
	32. 3	37.4		
	32.4	37.2		
	32.6	38, 0		
	32. 3	37.8	div.	μ
	35. 2	41.1	5.52	558, 6
	35. 0	41.0		
	<b>36. 4</b>	41.6		
	34. 4	42.1 rej.		
	<b>36.</b> 6	42.3		
	3 <b>6.</b> 2	42. 4		
	37. 0	42.5		
	36. 7	<b>42.</b> 3		
	<b>36.</b> 8	42, 5		
	37. 0	<b>42</b> . 8		
	3 <b>6.</b> 6	41.4		
	<b>36.</b> 9	42. 2		
	<b>36.</b> 9	43. 0		

## Dynamical flexure, with same arrangement.

				j.L
0478	32.3	42. 2	9. 9	524
469	32.0	41.8	9, 8	529
453	<b>30.</b> 3	40. 1	9.8	546
442	30.1	40.1	10.0	572
438	31.2	42.1	10. 9	630
437	31.9	41, 9	10. 0	<b>579</b>
433	31. 1	41.0	9. 9	<b>579</b>
426	31.8	42.0	10. 2	605
419	<b>31.</b> 8	41.2	9. 4	567
416	32.2	41. 2	9.0	546
413	32.3	41.4	9. 1	557
412	32. 2	41.3	9.1	559
409	32.7	42. 1	9. 4	582
403	33. 3	42. 2	8. 9	559
400	<b>3</b> 3. 3	42.3	9. 0	569

567

Scale in front of knife-edge about 56cm.8 (not measured). Statical flexure; 21.67 div. of scale=.001 inch.

1	inch.			25.1.44	
	Arc.		eadings.	Diff.	$f_0$ .
	. 0500	30.0	16.8		
		25. 7	12.6		
		25.4	12.4		
		27. 2	13.7		
		27. 7	14.1		
		30.0	16.6	div.	μ
		32.1	18.4	13.4	448. 7
		33.3	20.1		
		27.4	13.4		
		31.4	18.8		
		32.4	19.7		
		56.4	43.2		
		<b>56.</b> 7	41.9		
	Dynamical flexure, with same ar	rrangemer	ıt.		
	. 0480	<b>55.</b> 4	32, 3	23.1	403
	459	57. 7	35.8	21. 9	399
	451	58. 7	38.8	19. 9	369
	443	<b>59.</b> 3	38.4	20.9	395
	425	<b>59.</b> 3	39.6	19.7	388
	416	60. 7	41.2	19.5	393
	399	63. 7	45.8	17.9	376
		04-41-41	4	•	389
	Scale 50cm.4 behind knife-edge.	Statical			
	. 0500	22.8	15.8		
		24. 7	18.2		
		36. 7	29.8		
		52.8	29. 2		
		16. 3	47.4		
		22.7	10.8	div.	μ 017 0
		23.6	17. 2	6, 49	217 0
		27.6	17. 0		
		27.[4]	21.0		
		31. 4	25.3		
		36, 0	29. 3		
		41.0	33. 3		
		43. 4	<b>36.</b> 8	6.	
	Dynamical flexure, with same a	rrangeme	nt.		μ
	. 0495	<b>46.</b> 3	30.6	15.7	265
	488	<b>47.</b> 7	31.4	16.3	280
	483	49. 3	33. 7	<b>15.</b> 6	270
	478	<b>50.</b> 3	34. 4	<b>15.9</b>	279
	469	<b>51.</b> 3	36. 3	<b>15.</b> 0	268
	463	<b>52.</b> 8	37.3	<b>15.</b> 5	280
	426	<b>52.</b> 8	38. 5	<b>14.</b> 3	281
	414	<b>53. 4</b>	<b>41.</b> 3	12. 1	244
	410	<b>54.</b> 9	41. 0	13. 9	284
	407	<b>56.</b> 0	43.0	13.0	<b>26</b> 7
					272

The above experiments of April, 1889, were made with a view of testing experimentally the question of whether the statical or dynamical flexures should be used in reducing the periods of oscillation, or whether some intermediate value would be preferable. The pendulum was actually swung upon all these supports. Unfortunately, the measures of flexure of the excessively flexible supports are extremely discordant.

Repsold support on rubber blocks.

	$f_0$ forw	ard.	$f_0$ b	ehind.	$f_a$ be	elow,	F	$\mathbf{F}_{d}$
Stat	ical.	Dynamical.	Statical.	ehind. - Dynamical	Statical.	Dynamical.	Statical.	Dynamical.
	μ	μ	$\mu$	$\mu$	μ	μ	$\mu$ .	μ
April 18, 10	99	989	939	846	710	543	1014	913
April 21, 9	89	922	804	713			891	811

From the circumstances of the experiment, the results of the second day are to be absolutely preferred. The pendulum was swung on this support on April 18 and 20. The measurements of the flexure of the Repsold tripod on an oaken support are still more utterly discordant. The second experiment of April 21 seems to be the best, and this gives  $F_0=354^{\mu}$  for the statical flexure, and the dynamical flexure appears to be very little less. For the stiffest support we have  $F_0=62^{\mu}9$ , statical; dynamical not measured.

The details of the experiments to determine the periods of oscillation on these supports will be given in another report. The following are the results:

#### Stiffest support.

	Beigest support.	
	(Method of transits.)	
	Heavy end down,	Heavy end up.
March 31,	$1.\stackrel{\circ}{0}06435$	1, 006473
April 2,	439	178
April 4,	434	483
April 4,	111	475
Mean,	1,006438	1, 006477
(Me	ethod of eye and ear coinciden	ces.)
March 26,	1.006443	1.006468
March 27,	447	475
March 28,	437,	467
March 29,	435	460
Meau,	1. 006440	$\frac{-}{1,006470}$
	Repsold support.	
	(Method of transits.)	
	Heavy end down.	Heavy end up.
April 7,	1, 006499	1, 006506
April 30,	491	485
May 2,	500	528

500

1,006498

3,

May

Mean,

516

1.006509

## Repsold support.

(Method of eye and ear coincidences.)

	ŀ	leavy end down. s.	Heavy end up.
March 19,		1, 006523	1.006523
March 21,		490	473
March 22,		508	527
March 23,		492	533
June 4,		495	505
June 5,		489	483
June 6,		507	511
June 6,		515	505
	Mean,	1. 006502	1. 006508

## Oaken support.

(Method of eye and ear coincidences.)

		Н	eayy end down	-	Heavy end up.
April	24,		1.006545		1,006530
April			542		526
April			536		521
April			538		539
		Mean,	1. 006540		$\overline{1.006529}$
		On In	dia-rubber blo	eks.	•
April	18,		1, 006706		1.006612
April	20,	•	703		610
		Mean,	${1.006705}$		1. 006611

Let us now try the dynamical correction of these periods. For the Repsold support the dynamical  $F_0=237^{\mu}$ ; thence we deduce the corrections for the other supports, as follows:

	Heavy end down.	Heavy end up
Period, Repsold support, Dynamic correction,	1,006500 —84	1.006509 —36
Corrected period,	1. 006416	1.006473
Period, stiffest support, Period, oaken support, Period, on India rubber,	1. 006439 1. 006540 1. 006705	1. 006475 1. 006529 1. 006611
Apparent corrections:	-	k narowania kaka 1907-1904 di Mikaling manadana kanana mahaligai dikilingan
Stiffest support,	23	2
Oaken support,	124	56
India rubber,	289	138
The values of F <sub>0</sub> , calculated from these c	corrections, are as follow	s: ,

Stiffest support,

Oaken support,

India rubber,

65

352

821

13

371

915

Of course, extremely little weight is to be attached to the values calculated from "heavy end up." For the stiffest support and the oaken support the result is in very good accord with the statically observed flexure. For the India rubber support the dynamically observed flexure seems to be indicated, or rather something between this and the results of statical measures.

#### General conclusions.

- 1. The flexibility of almost any pendulum support has an important effect on the time of oscillation, and should be measured.
- 2. The flexure rotates the knife-edge about an axis, sometimes not over 60 cm. distant. It is, therefore, altogether erroneous to measure the flexure at any other point than the middle of the knife-edge, unless it be measured at a number of points and reduced to that point.
- 3. On a properly constructed support the difference between the statical and dynamical flexure should be immaterial. The dynamical flexure is less than the statical, owing to the time required for the transmission of the wave of strain to the more distant parts of the apparatus. The true correction seems to be intermediate between that calculated from the statical and the dynamical flexures, but pretty decidedly nearer to the latter.
- 4. A support like the Repsold tripod will grow more flexible with time, owing probably to the slight loosening of some parts.
- 5. Any dirt, cement, or other elastic film under the feet of such a tripod may greatly increase the flexure, as well as the difference between the two kinds.
- 6. If the flexure is considerable, it is likely to vary from day to day, or even during the course of an experiment.
  - 7. The tightening of the parts may or may not greatly affect the flexure.
  - 8. The loading of the support has no sensible effect.
- 9. Experiments made with weight and pulley give a larger value for the flexure than those made with the pendulum drawn to one side.

#### NOTE ON HARDY'S NODDY.

The theory of Hardy's noddy is very simple. When two pendulums oscillate on the same support in parallel planes, I have shown (Am. Jour. Sci., third series, xviii, 113) that one of the differential equations is

$$\lambda D_t^2 \varphi + D_t^2 s = -\gamma \varphi$$

where

t is the time;

- $\varphi$ , the instantaneous angle of inclination of one pendulum;
- s, the instantaneous linear displacement of its knife-edge from the position of repose;
- $\lambda$ , the virtual length of the pendulum:
- $\gamma$ , the vertical acceleration of each particle, or the constant of force of restoration of the pendulum.

The Hardy's noddy is a pendulum placed on the support of another pendulum so as to oscillate in a parallel plane. Its natural period  $\tau = \odot \sqrt{\frac{\lambda}{\gamma}}$  is as nearly as possible equal to T, the period of the main pendulum; but  $\gamma$ , instead of being gravity, is the excess of the force of a spring over gravity and is made to be as small as possible,  $\lambda$  being correspondingly small, so as to give  $\tau$  the right value. The noddy being very light, the value and changes of s are determined entirely by the main pendulum. We may, therefore, write

$$s = S \cos \frac{t}{m} \odot$$

Substituting this value in the differential equation, the solution of the latter is

$$\varphi = \theta \cos \frac{t - t_0}{\tau} \odot - \frac{1}{\gamma} \frac{S \odot^2}{\tau^2 - T^2} \cos \frac{t}{T} \odot.$$

But the noddy has no oscillation to begin with. This fact is represented by the equations

$$\phi = \frac{1}{\nu} \cdot \frac{S \odot^2}{\tau^2 - T^2}$$

S. Ex. 49-54

We thus have

$$\varphi = \frac{1}{\nu} \cdot \frac{S \odot^2}{\tau^2 - T^2} \left( \cos \frac{t}{\tau} \odot - \cos \frac{t}{T} \odot \right) = \frac{2}{\nu} \cdot \frac{S^2 \odot^2}{\tau^2 - T^2} \cdot \sin \frac{\tau - T}{2\tau T} t \odot \cdot \sin \frac{\tau + T}{2\tau T} t \odot.$$

This equation shows that the noddy oscillates with a period that is a sort of mean between its natural period and that of the large pendulum. The amplitude of oscillation increases from nothing at an initial rate equal to

$$rac{\mathrm{S}\odot^{3}}{\gamma( au+\mathrm{T}) au\mathrm{T}};$$

a rate not much affected by the value of  $(\tau - T)$ . But the amplitude increases more and more slowly, and reaches its maximum when

$$t = \frac{\tau T}{\tau - T}$$

after which it again diminishes and after the lapse of an equal time vanishes. At the beginning, the phase of motion of the support is  $\frac{1}{2}$ , and that of the noddy is 0, so that the support is one quadrant ahead. At the time of the first maximum the phase of the support is

$$\left(\frac{1}{2} + \frac{\tau}{\tau - T}\right)$$
©

and that of the noddy is

$$\frac{1}{2} \frac{\tau + T}{\tau - T} \odot$$

Subtracting the second from the first we see that the two motions are in opposition. When the motion of the noddy vanishes its phase is a quadrant in advance of that of the support. The motion immediately recommences, but

$$\sin \frac{\tau - T}{2\tau T} t$$

is now negative, and this shows that the difference of phase changes to the opposite quadrant, and that the two oscillations again proceed toward opposition.

We have thus far not taken account of the resistance to the motion of the noddy, although this must evidently be large. In consequence of it, the natural motion of the noddy would be of the form

$$\varphi = \Phi \bigcirc^{-\frac{t}{\theta}} \bigcirc \cos \frac{t}{\tau} \bigcirc.$$

From this we easily infer that the differential equation is

$$D_t^2 \varphi + 2 \frac{\Theta}{\theta} D_t \varphi + \frac{\Theta^2}{\tau^2} \varphi = \frac{S \Theta^2}{\lambda T^2} \cos \frac{t}{T} \Theta$$

The solution of this is

$$\varphi = \frac{S}{\lambda} \frac{1}{\theta^{2}(\tau^{2} - T^{2})} + \frac{\tau^{2} - T^{2}}{\tau^{2}} \left( \bigcirc -\frac{t}{\theta} \bigcirc \cos \frac{t}{\tau} \bigcirc -\cos \frac{t}{T} \bigcirc \right) + \frac{S}{\lambda} \frac{\frac{2\tau^{2}T}{\theta(\tau^{2} - T^{2})}}{\frac{\theta^{2}(\tau^{2} - T^{2})}{\tau^{2}}} + \frac{\tau^{2} - T^{2}}{\tau^{2}} \sin \frac{t}{T} \bigcirc \cdot \frac{t}{\theta(\tau^{2} - T^{2})} \right)$$

The signification of this is that the noddy approaches indefinitely toward settling down to an oscillation strictly synchronous with that of the support. Its ultimate amplitude is very little less than half what the maximum amplitude would be without resistance. But the phase may differ very much from that of the motion of the support. Namely, if the noddy is in precise adjustment to the period of the large pendulum, its phase will be one quadrant behind that of the support. If the noddy naturally oscillates slower than the large pendulum, its phase may be anywhere from one quadrant in arrear to opposition; if the noddy naturally oscillates faster than the pendulum, it may be anywhere from one quadrant behind, to coincidence.

If, then,  $\gamma$  is one tenth of gravity,  $\tau^2 - T^2$  one thousandth of a second, and S one tenth of a micron, the amplitude of movement of the noddy will be one thousandth of the radius, a quantity easily measured with a microscope.

ON THE INFLUENCE OF THE FLEXIBILITY OF THE SUPPORT ON THE OSCILLATION OF A PENDULUM.

(Translated from French into English by the author.)

NEW YORK, July 13, 1877.

DEAR SIR: On taking charge of the Coast Survey researches upon gravity, I ordered of Messrs. Repsold a reversible pendulum, to be a copy of that of the Prussian Geodetical Institute. But the instrument makers were at that time so taken up with the construction of instruments for the Transit of Venus, that the pendulum was only ready in the spring of 1875. I then went to Hamburg to receive it; and from Hamburg I went on to Berlin, where I found General Baeyer rather dissatisfied with the results obtained with the Prussian instrument. He specially mentioned the flexibility of the tripod, a source of error which pendulum experimenters have surely never overlooked. The pendulum apparatus that I had carried with me from America having been ruined in transportation, I was under the necessity of employing the new instrument, and therefore undertook to measure and take account of the error in question.

A pendulum support might be rickety, so that the pendulum in its oscillations should throw the knife-edge plane from one position to another, without its undergoing any resistance to the motion other than inertia and friction, between two fixed points. This, however, does not happen in the case of any of the supports that I have examined; for, upon observing their behavior under a high-power microscope, I have always found that they spring back exactly to their original position after every flexure that I have applied to them. In short, the movement with which we have to do is the oscillatory flexure of an elastic body. The amplitude of the oscillation is, at most, about  $\frac{1}{5000}$  of that of the lower knife-edge of the pendulum, so that its square may be neglected.

The plane of support of the knife is itself undoubtedly bent during the movement; but I neglect this and limit myself to the consideration of the movement of its middle point. When to this middle point is applied a horizontal force perpendicular to the knife edge, the latter describes a movement of revolution around an axis which, in the case of the Repsold apparatus, is situated behind and above the tripod at a distance of about a meter from the knife edge. We can neglect the difference between this movement and a translation, until we come to measure its amount. There is also a minute variation in the vertical pressure of the pendulum on the support, but this is very far from producing any sensible effect on the period of oscillation.

Let us denote by

- m the mass of a particle,
- r its distance from the knife-edge,
- $\omega$  the inclination, at rest, to the vertical of the perpendicular let fall from the particle on to the knife-edge,
  - M the mass of the pendulum,
  - l the length of the corresponding simple pendulum,
  - h the distance of the center of mass from the knife-edge,
  - T the period of the oscillation,
  - g the acceleration of gravity,
  - ε the elasticity of the support,
  - $\varphi$  the instantaneous inclination of the pendulum to its position of rest,
- \* the instantaneous displacement of the middle point of the knife-edge from its position of rest.
  - t the time.

Then, the horizontal velocity of a particle will be

$$r\cos(\varphi+\omega)D_{\epsilon}\varphi+D_{\epsilon}s$$

and its vertical velocity will be

$$r \sin(\varphi + \omega) D_i \varphi$$
.

Its living force will, therefore, be

$$\frac{1}{2}mr^{2}(\mathbf{D}_{t}\varphi)^{2}+mr\cos\left(\varphi+\omega\right)\mathbf{D}_{t}\varphi,\mathbf{D}_{t}s+\frac{1}{2}m(\mathbf{D}_{t}s)^{2},$$

and that of the pendulum will be

$$= \frac{1}{2} \mathbf{M} lh (\mathbf{D}_{t} \boldsymbol{\varphi})^{2} + \mathbf{M} h \cos \boldsymbol{\varphi}. \ \mathbf{D}_{t} \boldsymbol{\varphi}. \ \mathbf{D}_{s} s + \frac{1}{2} \mathbf{M} (\mathbf{D}_{s} s)^{2}.$$

The living force of the motion of the support itself may be left out of account since it involves the square of an excessively small velocity.\*

The differential of the potential energy is

Mgh sin 
$$\varphi$$
.  $d\varphi + \varepsilon s$ .  $ds$ .

There is really a third term to be added to this expression dependent on the molecular friction of the matter of the support. But I think we may neglect this term; for its effect cannot be very great, and its coëfficient is, at any rate, unknown.†

From the expressions for the living force and potential we deduce the Lagrangian equations

$$lD_{i}^{2}\varphi + \cos \varphi$$
.  $D_{i}^{2}s = -g \sin \varphi$ 

$$-h\sin\varphi. (\mathbf{D}_{t}\varphi)^{2} + h\cos\varphi. \mathbf{D}_{t}^{2}\varphi + \mathbf{D}_{t}^{2}s = -\frac{\varepsilon}{\mathbf{M}}s,$$

or, neglecting terms of the second degree,

$$lD_t^2 \varphi + D_t^2 s = -g\varphi$$

$$hD_{\iota}^{2}\varphi + D_{\iota}^{2}s = -\frac{\varepsilon}{M}s.$$

[Note.—1882, July 24.—I omit the solution of these equations as originally given, and substitute the following, which is perhaps less inelegant.—Subtracting the second equation from the first, we get

$$(l-h)D_{t}^{2}\varphi+g\varphi=\frac{\varepsilon}{M}s$$

 $\mathbf{or}$ 

$$\mathbf{D}_{t}^{2} \mathbf{8} = \frac{\mathbf{M}}{\varepsilon} (l - h) \mathbf{D}_{t}^{4} \boldsymbol{\varphi} + \frac{\mathbf{M} \boldsymbol{g}}{\varepsilon} \mathbf{D}_{t}^{2} \boldsymbol{\varphi}$$

Substituting this value in the first differential equation, we have

$$\frac{\mathrm{M}}{\varepsilon}(l-h)\,\mathrm{D}_{\iota}^{4}\varphi + \left(l + \frac{\mathrm{M}g}{\varepsilon}\right)\mathrm{D}_{\iota}^{2}\varphi + g\varphi = 0.$$

Separating the operator into factors, and using the abbreviation

$$i=4\frac{Mg}{\varepsilon l}\cdot\frac{1-\frac{h}{l}}{\left(1+\frac{Mg}{\varepsilon l}\right)^{2}}.$$

we get

$$\left[D_{i}^{2} + \frac{\epsilon l + Mg}{2M(l-h)}(1+\sqrt{1-i})\right] \cdot \left[D_{i}^{2} + \frac{\epsilon l + Mg}{2M(l-h)}(1-\sqrt{1-i})\right] \varphi = 0.$$

<sup>\*</sup>It is easy to see that the effect of this would be to increase the last term of the living force; this would affect the second of the differential equations just as if M had been multiplied and h divided by the same quantity. But this would not affect the final result. [1882.]

<sup>†</sup>This is the point to which the greatest objection to my work has been made. [1882.]

The solution of this is

$$\varphi = \mathbf{A}_1 \cos \left( \sqrt{\frac{\varepsilon l + \mathbf{M}g}{2\mathbf{M}(l-h)}} (1 - \sqrt{1-i}) \cdot t + \eta_1 \right) + \mathbf{A}_2 \cos \left( \sqrt{\frac{\varepsilon l + \mathbf{M}g}{2\mathbf{M}(l-h)}} (1 + \sqrt{1-i}) \cdot t + \eta_2 \right)$$

where  $A_1, A_2, \eta_1, \eta_2$  are arbitrary constants. On neglecting the square of  $\frac{Mg}{\epsilon l}$ , this reduces to

$$q = A_1 \cos \left( \sqrt{\frac{g}{l}} \left( 1 - \frac{Mg}{\varepsilon l} \right) \cdot t + \eta_1 \right) + A_2 \cos \left( \sqrt{\frac{\varepsilon}{l} + \frac{Mgh}{\varepsilon l^2}} \cdot t + \eta_2 \right)$$

The second term represents a mere tremor, for its period is very short, owing to the large value of  $\epsilon$ . The period of the first harmonic constituent is

$$T = \sqrt{\frac{l}{g}} + \frac{M}{\varepsilon}$$

From the value of  $\varphi$  and the first equation of this note, we deduce the following value of s:

$$\begin{split} s &= \frac{\mathrm{M}g}{2\varepsilon} \bigg( -\frac{\varepsilon l}{\mathrm{M}g} (1 - \sqrt{1 - i}) + 1 + \sqrt{1 - i} \bigg) \mathrm{A}_1 \cos \bigg( \sqrt{\frac{\varepsilon l + \mathrm{M}g}{2\mathrm{M}(l - h)}} (1 - \sqrt{1 - i}). \ t + \eta_1 \bigg) \\ &+ \frac{\mathrm{M}g}{2\varepsilon} \bigg( -\frac{\varepsilon l}{\mathrm{M}g} (1 + \sqrt{1 - i}) + 1 - \sqrt{1 - i} \bigg) \mathrm{A}_2 \cos \bigg( \sqrt{\frac{\varepsilon l + \mathrm{M}g}{2\mathrm{M}(l - h)}} (1 + \sqrt{1 - i}). \ t + \eta_2 \bigg) \end{split}$$

It thus appears that the amplitude of the principal constituent of s is nearly

$$h \frac{Mg}{\epsilon l} A_1$$
.

while that of the other constituent is nearly  $-lA_2$ .

To find the best way of starting the pendulum so as to make the ratio of  $A_2$  to  $A_1$  as small as possible, we must consider how to make the initial value of s as nearly as possible  $h \frac{Mg}{\varepsilon l}$  times the initial value of  $\varphi$ . Now, it is easy to see that if the pendulum is supported at a point at a distance x from the knife-edge, any yielding of the support will diminish the value of  $\varphi$  as expressed by the equation

$$ds = -x \sec \varphi . d\varphi.$$

Substituting this in the expression for the differential of the potential energy, this last becomes

Mgh sin 
$$\varphi$$
,  $d\varphi - \varepsilon s x$  sec  $\varphi$ ,  $d\varphi$ .

Equating this to zero, we find

$$s = h \frac{Mg}{\epsilon x} \sin \varphi \cdot \cos \varphi$$
.

In order that this should be equal to  $h\frac{Mg}{\epsilon l}\varphi$ , it is only necessary to put x=l, so that in starting the pendulum the finger or trigger should be applied at the lower knife-edge or center of gyration.] The elasticity,  $\epsilon$ , may be measured by observing the deflection, S, of the support produced by a horizontal force equal to the unit of weight. For

$$\epsilon = \frac{g}{S}$$

Substituting this value, we find

$$\varphi = \frac{A}{h} \cos \left( \sqrt{\frac{g}{\left(l + MS_l^h\right)^t}} \right)$$

Accordingly, the effect on the pendulum is to give it a virtual length greater than what it would have on a rigid support by  $\mathbf{MS}^h_{\tilde{I}}$ .

Let us denote the duration of an oscillation by T, and let  $\Delta$  be used to indicate the effects of flexure. Then, since

$$\mathbf{T}^{2} = \mathbf{\mathfrak{O}}^{2} \frac{l}{g}$$

we have

$$\Delta T^2 = \frac{O^2}{g} MS^{\frac{h}{l}}.$$

If we distinguish by subjacent letters the two positions of a reversible pendulum, we have

$$\frac{O^{2}l}{g} = \frac{\mathbf{T}_{d}^{2}h_{d} - \mathbf{T}_{u}^{2}h_{v}}{h_{d} - h_{u}}$$

and

$$\Delta l = MS$$
,

or putting  $\lambda$  for the length of the second's pendulum

$$\Delta \lambda = MS_{j}^{\lambda}$$
.

To determine the flexure, I fasten in the slot in the plane of suspension of the Repsold apparatus a fish-line passing horizontally in the direction of the pendulum's movement over an Atwood's machine pulley, and on the end of this cord I hang a kilogramme. [With a stronger support, the pendulum itself may conveniently replace the kilogramme.] On the extremity of the plane of suspension, or at the end of an arm attached thereto,\* I stick a glass stage micrometer, turned so as to measure in a direction parallel to the impressed force. This scale is looked at by a microscope carrying a filar micrometer, and solidly mounted upon an independent support, the standard of which is a piece of gas pipe about 10 centimeters in diameter.

I now give a brief *résumé* of my results, beginning with the experiments to determine the position of the fixed axis about which the head of the Repsold support rotates during flexure.

#### A.—Experiments made on a level with the suspension plane.

Hoboken, March 10, 1877. Temperature 13° C.

Distance of scale from	Flexure in revolutions of the micrometer screw		
end of plane.	Observed.	Calculated.	
m.			
-0.496	+0.211	+0.209	
+0.053	+. 0356	+0.358	
+0.318	+0.436	+0.431	

The calculated quantities suppose that the axis pierces the suspension plane at a distance of  $1^{m}.355$  behind the forward end of the suspension plane.

<sup>\*</sup>This arm is best made of brass tubing, which may be cut out to make it lighter. [1882.]

B.—Experiments in the vertical of the forward end.

Hoboken, March 12, 1877. Temperature 14° C. Observer, Sub-assistant Smith. + = below; - = above.

Position of the scale relative	Flexure in revolutions	of the micrometer screw.
to the suspension plane.	Observed.	Calculated.
m.		
0.44	+0.196	+0.196

+0.340

 $\pm 0.446$ 

+0.332

+0.454

0.000

 $\pm 0.395$ 

The calculated quantities suppose the axis to pierce the vertical of the forward end of the suspension plane 1".07 above this plane. It is not at all surprising that the instantaneous axis is above the suspension plane. Let us suppose that the flexure existed exclusively in three feet of the support. In this case the movement of the upper end of each foot would be perpendicular to the general direction of the foot, and at the same time perpendicular to the radius of the circle of revolution, so that the foot would be directed directly towards the fixed axis. The axis is without doubt behind the support, on account of the flexure of the plane itself.

I made experiments at Geneva, Paris, Berlin, and New York, in order to determine S numerically. The experiment at Geneva, made the 13th of September, was only a trial. But I had a good pulley which I had borrowed from the workshop of the Geneva society for the construction of physical instruments, and I got as an approximate value-

$$S = 0^{mm}.034$$

The pulley that I used at Paris had considerable friction, to which can be attributed the fact that the numbers found differ sensibly from those obtained with the aid of better apparatus.

These are the figures-

January 18, 1876, at Messrs. Brünner, Temp. 1° C S=
$$0^{\rm mm}.0363$$
 March 7, 1876, at the Paris Observatory, Temp. 9° C S= $0^{\rm mm}.0371$ 

At Berlin I used a very delicate pulley which turned on friction whels, in order to diminish the friction. It belonged to the Physical Cabinet of the Institute of Technology of Berlin, and was put at my disposition by the kindness of Professor Paalzow. The micrometric readings were made alternately with and without the weight, making but one reading each time, in order to avoid any error arising from the support of the micrometer, this being made of wood. In the readings made alternately with and without the weight, I ended with the arrangement with which I began (11 for one, and 10 for the other), in order that the mean instant of the observations should be the same for the two arrangements. The value of 1 division of the micrometer screw was measured separately.

Below are the results of the different series-

At Hoboken (near New York) I obtained, through the kindness of Professor Morton, an excellent pulley, made in the workshop of the Stevens Institute of Technology. I always made a reading on each one of the lines of the scale before changing the disposition of the weight.

The results of the separate series are-

March 7, 1877, Temp. 15° C.,	$S=0^{mm}.0342$
March 10, 1877, Temp. 12°.	$0^{\text{mm}}.0332$
, , , <del>-</del>	$0^{\mathrm{mm}}$ .0337
	$0^{\rm mm}.0343$
	$0^{\mathrm{mm}}.0342$
•	$0^{\rm mm}.0339$
	$0^{\mathrm{mm}}.0334$
These two series should have double	$0^{\mathrm{mm}}.0342$
weight in the reduction,	$0^{\mathrm{min}}.0342$
Mean.	$S = 0^{mm}.0340 + 0^{mm}.000$

In all the experiments made in the different positions of the scale the flexure obtained has been reduced to the center of the knife, and this last is what is called S.

It is to this last value that I give the preference.

It follows, from the experiments described on pages 430-431, made to determine the position of the axis of rotation, that the forward end of the suspension plane is distant from that axis by  $\sqrt{1^{m}.355} \times 1^{m}.07 = 1^{m}.20$ . And, since the movement of this end with the weight of a kilogramme is  $S+0^{mm}.0008$ , the correction +0.0008 arising from the reduction from the center of knife to the forward end, it follows that the torsion of the support by that force is  $\frac{0^{mm}.0348}{1^{m}.20} = 0.0000290 = 5.$ ". 8.

Although there is nothing to be suspected in this result, I wished to check it by a direct experiment. I attached a mirror at the extremity of the suspension plane, and, with the aid of a telescope, I measured the torsion by the reflection of a scale, and I found it 6". This method, of course, is not as exact as the other.

In order to arrive at another confirmation of the theory, I made the following observations on the flexure produced by the oscillation of the pendulum itself in its two positions, using a tolerably high-power microscope (i. e., magnifying 500 diameters). The scale used was made by Mr. Rodgers, of Harvard College Observatory. It is divided with extreme exactness, the interval between two lines being  $\frac{1}{4\sqrt{1000}}$  of an English inch. It was fixed 70 millimeters before the center of the knife, which gives a correction to S of +.0019.

If  $\Phi$  is the amplitude of oscillation on each side of the vertical, the double amplitude of the vibration of the scale should be

$$^{2}$$
 M  $(8 + 0^{min}.0019) \frac{h}{l} \Phi$ 

in which M=6.308 and  $\frac{h}{l}=\frac{17}{56}$  or  $\frac{39}{56}$  according as the pendulum is suspended by the knife nearest or farthest from the center of gravity. I used this formula in calculating the quantities now given.

#### DYNAMICAL FLEXURE.

A .- Pendulum suspended by the knife farthest from the center of gravity.

HOBOKEN, March 20, 1877.

	Ф	Amplitude of the movement of the scale. 1 div. $=\frac{1}{4000}$ inch.		
		Observed.	Calculated	
 ٠	,	Divisions.	Divisions.	
2	32	2. 2	2. 2	
2	30	2. 1	2. 1	
2	24	2.0	2.1	
2	22	1.9	2. 0	
2	20	1.9	2.0	
2	19	1. 95	2.0	
1	43	1.5	1. 5	
0	47	0.8	0. 7	

B.—Pendulum	suspended la	the knife ne	arest the cente	r of aravitu.

	Ф		Amplitude of the movement of the scale.  1 div. $=_{7000}$ in.			
			Observed.	Calculated.		
	c	,	Divisions.	Divisions.		
:	2	39	1. 0	1.0		
	2	34	0. 9	1. 0		
	2	29	0. 9	0. 9		
:	2	25	0. 9	0. 9		
	2	22	0.8	0. 9		
	2	14	0. 8	0. 8		
	2	12	0.8	0. 8		
	2	06	0.7	0. 8		
	2	04	0. 75	0. 8		
	1	57	0. 75	0. 7		
	1	51	0. 75	0. 7		

In making these observations, I saw distinctly the little subsidiary vibration at the end of each oscillation arising from the second term of the formula.

Finally, I swung the pendulum on two supports of different flexibility—one was the metallic tripod, by Repsold, to which refer the flexure measurements given above; the other was made by fixing the upper part of the Repsold tripod to a thick wooden plank by means of bronze bolts passing through the three holes through which the feet pass. These holes are conical, and the bolts fit exactly. I put on each bolt between the head of the support and the plank a leaden washer, so that, in tightening the bolts and compressing the washers, great stability was obtained and at the same time a horizontal position. The plank (which was 5 centimeters thick) was cut in order to make a place for the pendulum, and it was placed by force between a stone wall and a brick pillar. A slit was then cut, in which a pulley of an Atwood machine was placed to measure the flexure.

Experiments on the flexure of this support.

Новокем, Мау 21, 1877.

	Distance of scale be- fore +, behind of the center of knife in English inches.	Distance of scale to suspension plane in English inches + above, — below.	Flexure in millime- ters under a weight 1 kilogramme.	Temperature C.	Observer.
			and the second second		1 1 4 4
	Inches.	Inches.	mm.	e	
	+ 1.2	- 1.3	+0.0052	18. 3	ES.
-	+ 1.2	- 1.3	+ .0052	18.9	ES.
	+ 1.2	+39.5	0425	20.0	C. S. P.
	+13.2	+ 39, 5	0367		C. S. P.

It follows that for this apparatus  $S^1=0^{\rm mm}.0031$ , and that the difference between the values of S for the two supports is  $0^{\rm mm}.0309$ . Now I find  $\frac{\pi^2 l}{g}=1.0125$  sidereal seconds and  $l=1^{\rm m}$ . Hence, we conclude that the difference of  $\frac{\pi^2 l}{g}$  according as the pendulum oscillates on one or the other supports must be equal to

$$\frac{\pi^2 l}{g} \frac{\mathbf{M} (\mathbf{S} - \mathbf{S}^1)}{l} = \frac{81}{80} \times 6.308 \times 0.0309 = 0.000197$$

I swung the pendulum three times on the less solid support and once on the most solid to verify the theory. I observed 10 consecutive passages of the pendulum across the vertical at intervals of 5 minutes, using a relay that I invented for this purpose.

#### A.—Oscillations on the Repsold metallic support.

HOBOKEN, April 1, 1877.

PENDULUM SUSPENDED BY THE KNIFE NEAREST THE CENTER OF GRAVITY.

Nu	nber of oscillations.	Interval by chronometer.	Reduction to infinitely small arc.	Corrected interval.	Period.
	· · · · · · · · · · · · · · · · · · ·	8.	8.	8.	8.
	300	301. 9652	0.0130	301. 9522	1.006507
	296	297. 9408	-0.0084	297. 9324	528
	298	299. 9533	0. 0060	299. 9473	535
	* Mean				1. 0065238
	PENDULUM	SUSPENDED BY THE	KNIFE FARTHEST FR	OM THE CENTER OF GI	RAVITY.

	8.	8.		8.	
296	297. 9094	-0.0092		297, 9002	1.006420
302	303, 9374	-0.0081		303, 9295	389
296	297, 9060	0. 0066	!	297. 8994	417
Mean					1, 0064067
Mean					1

Hence, we have

$$T_1^2 = 1^{\circ}.0128544$$
  
 $T_2^2 = 1^{\circ}.0130902$ 

And since  $h_1: h_2=101: 44$  we have

$$\frac{\pi^2 l}{g} = \frac{T_1^2 h_1 - T_2^2 h_2}{h_1 - h_2} = 1.013 \left( 1 - \frac{101 \times 0.0001456 + 44 \times 0.0000902}{57} \right) = 1.012672$$

This value is to be corrected for rate of chronometer and temperature. The chronometer lost  $0^{\circ}.86$  per day, which gives a correction to  $T^{2}$  of  $+0^{\circ}.000020$ . The temperature during the time the heaviest mass was above was  $12^{\circ}.7$  in the mean, and  $12^{\circ}.9$  when this mass was below. Hence, to reduce to  $13^{\circ}$  we must apply a correction of

$$0.1 \times 101 - 0.3 \times 44 = 0.0000186 = -0.000001$$

Hence we conclude

$$\frac{\pi^2 l}{g}$$
 at 13° C.=1.012691

April 7, 1877

PENDULUM SUSPENDED BY THE KNIFE FARTHEST FROM THE CENTER OF GRAVITY.

Num	ber of oscillations.	Interval by chronometer.	Reduction to infinitely small arc.	Corrected interval.	Period.
		8.	8.	8.	€,
	290	291. 8794	0. 0103	291. 8691	1. 006445
	296	297. 9131	0, 0086	297. 9045	434
	298	299. 9241	0. 0073	299. 9168	482
	298	299. 9241	0.0060	299. 9181	437
	358	360. 3090	0. 0058	366, 3032	434
	Mean				1.0064357

## PENDULUM SUSPENDED BY THE KNIFE NEAREST THE CENTER OF GRAVITY.

	<b>8.</b>	8.		<b>6</b> .
288	289. 9026	0. 0132	289. 8894	1.006560
98	299. 9648	0.0092	299. 9556	562
100	301. 9760	0. 0067	301. 9693	564
98	299. 9564	0, 0051	299. 9513	548
98	299, 9591	0.0037	299. 9524	552
Mean				1. 0065578

<sup>\*</sup> When we have a series of equal consecutive intervals if n is the number of intervals and i is the position of one of them we should, in taking the mean, give to this interval the weight -i (i-1).

Hence we have

$$T^{2}_{1}=1^{s}.0129128$$

$$T^{2}_{2}=1^{s}.0131586$$

$$\frac{\pi^{2}l}{g}=1^{s}.012723$$
Daily correction,  $+0^{s}.44$ 
Temp. 15°.8 (both positions)
$$-000052$$

$$\frac{\pi^{2}l}{g}$$
at 13° C.= 1.012681

April 8, 1877.

# PENDULUM SUSPENDED BY THE KNIFE NEAREST THE CENTER OF GRAVITY.

	Number of oscil- lations.	Interval by chro- nometer.	Reduction to infi- nitely small are.	Corrected time.	Period.
•	!	8.	8.	8.	
	298	299. 9647	-0.0175	299. 9472	1. 006534
	298	299. 9549	-0.0111	299. 9438	523
	298	299. 9539	-0.0080	299, 9459	530
i	298	299. 9484	-0.0055	299. 9429	520
1	298	299. 9481	<b>-0.</b> 0039	299, 9442	526
1	Mean	••••••			1. 0065261

# PENDULUM SUSPENDED BY KNIFE FARTHEST FROM CENTER OF GRAVITY.

	8.	8.	8.	8.
298	299. 9229	-0.0066	299. 9163	1. 006431
298	299. 9213	-0.0058	299. 9155	426
297	298. 9125	-0.0049	298. 9076	423
299	300. 9236	-0.0042	300. 9194	419
298	299. 9174	-0.0035	299. 9136	422
Mean				1. 0064246

 $\begin{array}{c} s. \\ T_1^2 = 1.0128905 \\ T_2^2 = 1.0130948 \\ \hline \frac{\pi^2 l}{g} = 1.012733 \\ \text{Daily correction, } -0.41 \\ \text{Temp. heavy end up, } 13^\circ.2 \\ \text{Temp. heavy end down, } 13^\circ.5 \\ \end{array} - .000016$ 

$$\frac{\pi^2 l}{g}$$
 at 13°=1.012708

Hence the three experiments on the Repsold support give for the value of  $\frac{\pi^2 l}{g}$  at 13° C.

April 1, 1.012691 April 7, 1.012681 April 8, 1.012708

Mean, 1.012693

### B.—Experiments made on the stiffest support.

Новокем, Мау 14, 1877.

# PENDULUM SUSPENDED BY THE KNIFE FARTHEST FROM THE CENTER OF GRAVITY.

Mea	n insta trans	ant of 10 its.	Interval of 298 oscillations.	Reduction to infinitely small arc.	Corrected in- terval.	Interval of 298 oscillations.	Reduction to infinitely small arc.	Corrected interval.
h.				8.	8.	8.	8.	s
	m.	8. 00. 4007	8.	8.		٠.	٥٠	••
14		22.4307						
	7	22.8245					• • • • • • • • • • • • • • • • • • • •	
	11	22, 3337	299. 9030	-0.0132	299. 8898			
	12	22. 7213	**************			299. 8968	-0.0126	299. 8842
	16	22. 2313	299, 8976	-0.0110	299. 8866			
	17	22. 6209				299, 8996	0.0106	299. 8890
	22	22. 5145				299. 8936	0. 0087	299. 8849
	23	22, 9017				· ·		
	27	22, 4055				299, 8910	-0.0074	299. 8836
	28	22. 7949	299, 8932	0. 0072	299. 8860			
	33	22. 6896	299. 8947	-0.0060	299. 8887			

#### PENDULUM SUSPENDED BY THE KNIFE NEAREST THE CENTER OF GRAVITY.

Mean instant of 10 transits.				Reduction to infin- itely small arc.	Corrected intervals.	
h.	m.	8.	8.	8.	8.	
15	53	22, 4041			·	
	58	22.3579	299. 9538	<b>—0.0198</b>	299. 9340	
	3	22.3058	299. 9479	0. 0131	299. 9348	
	8	22, 2531	299. 9473	0. 0087	299. 9386	
	13	22, 2119	299, 9588	0, 0062	299. 9526	
	18	22, 1554	299. 9435	- 0. 0044	299, 9391	
	23	22, 1011	299, 9457	0.0031	299, 9426	

Mean  $T_2 = 1^{\circ}.0065104$ .

Hence we find-

$$\begin{array}{c} s. \\ T_1^2 = 1.0127144 \\ T_2^2 = 1.0130632 \\ \frac{\pi^2 l}{g} = 1.012445 \\ \end{array}$$
 Daily corr. to chron.  $+$  2\*.59  $-$  .000060 Temp. heavy end down, 14°.18  $\left. \begin{array}{c} - .000010 \\ \hline - .000010 \end{array} \right.$  Temp. heavy end up,  $-$  15°.00  $\left. \begin{array}{c} \frac{\pi^2 l}{g} \end{array}$  at 13° =1.012495

Comparing this value with the one obtained with the other support we find a difference of 0.000198. The difference, according to the computations of the experiments on flexure, ought to have been 0.000197,\* which shows a sufficient agreement.

Yours, most faithfully,

[Signed]

C. S. PEIRCE,

Assistant United States Coast Survey.

In the original publication, owing to an erroneous value for the mass of the pendulum, this is erroneously calculated as 0.000191. The agreement of the experiments with theory is, therefore, much better than was supposed.

ON THE INFLUENCE OF INTERNAL FRICTION UPON THE CORRECTION OF THE LENGTH OF THE SECONDS PENDULUM FOR THE FLEXIBILITY OF THE SUPPORT.

It has been shown by Prof. A. M. Mayer that the only sensible resistance to the motion of a tuning-fork is proportional to the velocity. In the case of a slowly vibrating body the chief effect is probably due to that lagging of the strain after the stress, which Weber has called the elastic after effect (Nachwirkung). The influence of the former mode of resistance upon the period of oscillation of a pendulum oscillating on an elastic tripod is easily calculated. The same thing cannot, in my opinion, be effected for the other kind of resistance, in the present state of our knowledge; nevertheless, the main characteristics of the motion may be made out. Put

- t, for the time;
- $\varphi$ , for the instantaneous angle of deflection of the pendulum;
- s, for the instantaneous horizontal displacement of the knife-edge from its position of equilibrium, in consequence of the flexure of the support;
- t, for the length of the corresponding simple pendulum;
- h, for the distance from the knife edge to the center of mass of the pendulum;
- g, for the acceleration of gravity;
- $\gamma$ , for the ratio of g to the statical displacement of the point of support, which would be produced by a horizontal force equal to the weight of the pendulum;
- a, for the coefficient of internal friction supposed proportional to the velocity.

Then the differential equations are

$$lD^{2}_{\iota}\varphi + D^{2}_{\iota}s = -g\varphi$$
  
$$hD^{2}_{\iota}\varphi + D^{2}_{\iota}s = -\gamma s - aD_{\iota}s$$

The solution of these equations will be of the form (using @ for the Neperian base and @ for the ratio of circumference to diameter):

$$\varphi = \mathbf{A}_{1} \langle \hat{\phi}^{z},' + \mathbf{A}_{2} \langle \hat{\phi}^{z},' + \mathbf{A}_{3} \langle \hat{\phi}^{z},' + \mathbf{A}_{4} \langle \hat{\phi}^{z},' \rangle 
s = \mathbf{B}_{1} \langle \hat{\phi}^{z},' + \mathbf{B}_{2} \langle \hat{\phi}^{z},' + \mathbf{B}_{3} \langle \hat{\phi}^{z},' + \mathbf{B}_{4} \langle \hat{\phi}^{z},' \rangle$$
(1)

where  $z_1$ ,  $z_2$ ,  $z_3$ ,  $z_4$  are the roots of the equation

$$(l-h)z^{4}+alz^{3}+(\gamma l+g)z^{2}+agz+\gamma g=0,$$

where, for each subscript letter,

$$B = -(l + \frac{g}{2^2}) A,$$

and where four arbitrary constants are determined by the initial conditions.

The roots of the biquadratic equation are all imaginary, and may be written

$$z_1 = -\xi_1 + \eta_1 \sqrt{-1}$$
  $z_3 = -\xi_2 + \eta_2 \sqrt{-1}$   
 $z_2 = -\xi_1 - \eta_1 \sqrt{-1}$   $z_4 = -\xi_2 - \eta_2 \sqrt{-1}$ 

Expressing the coefficients in terms of the real and imaginary parts of the roots, the equation becomes

$$z^{4}+2(\xi_{1}+\xi_{2})z^{3}+(4\xi_{1}\xi_{2}+\xi_{1}^{2}+\xi_{2}^{2}+\eta_{1}^{2}+\eta_{2}^{2})z^{2} +2[(\xi_{1}^{2}+\eta_{1}^{2})\xi_{2}+(\xi_{2}^{2}+\eta_{2}^{2})\xi_{1}]z+(\xi_{1}^{2}+\eta_{1}^{2})(\xi_{2}^{2}+\eta_{2}^{2})=0.$$

If the terms in  $z^3$  and z were neglected, that is, if a were neglected, the solution of the false equation so obtained would be as follows (where observe the varying sign of  $\eta_1$ ):

$$\begin{split} \text{False } z^2 &= -\tfrac{1}{2} (4 \mathcal{E}_1 \mathcal{E}_2 + \mathcal{E}_1^2 + \mathcal{E}_2^2 + \eta_1^2 + \eta_2^2) \\ &\quad \pm \tfrac{1}{2} (4 \mathcal{E}_1 \mathcal{E}_2 + \mathcal{E}_1^2 + \mathcal{E}_2^2 - \eta_1^2 + \eta_2^2) \sqrt{1 + 4} \, \frac{4 \mathcal{E}_1 \mathcal{E}_2 \eta_1^2 - \mathcal{E}_1^2 (\eta_2^2 - \eta_1^2) - \mathcal{E}_1^2 \mathcal{E}_2^2}{(4 \mathcal{E}_1 \mathcal{E}_2 + \mathcal{E}_1^2 + \mathcal{E}_2^2 - \eta_1^2 + \eta_2^2)^2} \end{split}$$

Now, in the actual case,  $\eta_2$  will be at least 100 times  $\eta_1$ ,  $\mathcal{E}_2$  will be quite large, and  $\mathcal{E}_1$  very small. We may, therefore, neglect the square of the fraction under the radical; and we have very closely

$$\begin{aligned} \text{False } z_1^2 = \text{false } z_2^2 = -\eta_1^2 + \frac{4\xi_1 \xi_2 \eta_1^2 - \xi_1^2 (\eta_2^2 - \eta_1^2) - \xi_1^2 \xi_2^2}{4\xi_1 \xi_2 + \xi_1^2 + \xi_2^2 - \eta_1^2 + \eta_2^2} \\ \text{False } z_3^2 = \text{false } z_4^2 = -\eta_2^2 - \xi_1^2 - \xi_2^2 - 4\xi_1 \xi_2 \frac{4\xi_1 \xi_2 \eta_1^2 - \xi_1^2 (\eta_2^2 - \eta_1^2) - \xi_1^2 \xi_2^2}{4\xi_1 \xi_2 + \xi_1^2 + \xi_2^2 - \eta_1^2 + \eta_2^2} \\ \text{False } z_1 = -\text{ false } z^2 = \eta_1 \left( 1 - \frac{1}{2\eta_1^2} \frac{4\xi_1 \xi_2 \eta_1^2 - \xi_1^2 (\eta_2^2 - \eta_1^2) - \xi_1^2 \xi_2^2}{4\xi_1 \xi_2 + \xi_1^2 + \xi_2^2 - \eta_1^2 + \eta_2^2} \right) \sqrt{-1}. \end{aligned}$$

We thus see that, by neglecting the resistance, we get for the value of  $z_1$  a quantity which requires only a minute correction in order to give the imaginary part of the true  $z_1$ . The same thing is not true for  $z_3$  and  $z_4$ . Now,  $\eta_1$  is  $\odot$  divided by the principal period of oscillation of the pendulum upon the flexible stand. This is the quantity which we wish to determine; the others have only to be known approximately for the purpose of calculating the small correction to this. The logarithmic decrement of the amplitude of oscillation of the pendulum in the unit of time, so far as it is due to internal friction, is the quantity  $\mathcal{E}_1$ . After these two quantities have been approximately ascertained, we may approximate to the quantity  $(\mathcal{E}_2^2 + \eta_2^2)$  by means of the equation

$$(\mathcal{E}_1^2 + \eta_1^2) (\mathcal{E}_2^2 + \eta_2^2) = \frac{\gamma g}{l - h}.$$

Then, by eliminating a between the two equations

$$2(\xi_1 + \xi_2) = \frac{al}{l-h},$$

$$2[(\xi_1^2 + \eta_1^2)\xi_2 + (\xi_2^2 + \eta_2^2)\xi_1] = \frac{ag}{l-h},$$

we obtain  $\mathcal{E}_{2}$ , and consequently  $\eta_{2}$ . The values so obtained must satisfy the equation

$$4\mathcal{E}_{1}\mathcal{E}_{2} + \mathcal{E}_{1}^{2} + \mathcal{E}_{2}^{2} + \eta_{1}^{2} + \eta_{2}^{2} = \frac{\gamma l + g}{l - h}.$$

Before proceeding to the consideration of the elastic after effect, I propose to apply the equations thus obtained to the calculation of the correction of the seconds' pendulum for the flexure of the stand, supposing the internal friction to be proportional to the velocity.

For the pendulum used by me we have the approximate values—

l=1.00; h (heavy end up)=0.30; h (heavy end down)=0.70; g (New York)=0.993 ×  $\odot^2$ =9.89;  $\gamma = \frac{1}{0.0002125} = 4706$ ;  $\eta_1 = 1.00$ .

The accompanying table shows that  $\mathcal{E}_1$ =0.000008. From this we calculate that with heavy end up  $\mathcal{E}_2$ =0.08,  $\eta_2$ =257; with heavy end down  $\mathcal{E}_2$ =0.17,  $\eta_2$ =392. From this it appears that the correction of  $\eta_1$  is absolutely insensible, or, in other words, the effect of resistance (supposed proportional to the velocity) vanishes. That this is nearly, in fact, the case for my instrument is shown by the circumstance that the times of oscillation upon stands of different rigidities agree with the values calculated in leaving the internal friction out of account.

U. S. Coast Survey. Pendulum. Decrement of Arc due to internal friction of brass of tripod. Pendulum was swung on brass tripod in Paris, Geneva, and Kew. On a stand ten times as stiff in Hoboken. The times of decrement given are the SUM of the times with the heavy end up and heavy end down.

Half amplitude.	Time decre	Stiff stand.	Time short- ened by inter- nal friction.	Ratio of shortening.	Decrement in one second.	Decrement due to internal friction in one second.	Mean arc.	Natural logarith- mic decrement due to internal friction.
100′	1073*	1095•	+22s	. 022	0.0186	. 00023	90′	. 0000025
80 70	706	762	+56	. 080	.0142	. 00114	75	. 0000152
. 50	1927	1969	+42	. 020	. 0104	. 00037	60	. 0000062
40	1377	1254	Reject.				Mean	. 000008

The last interval is probably affected by an error in the graduation of the scale used on one of the stands.

M. Plantamour proposes to determine the effect of the internal friction of the pendulum stand upon the correction for flexure, by means of the difference between the statical and dynamical flexure. He has made numerous observations, which, according to his own interpretation of them, would show that, if a pendulum be supported in a certain inclined position until the stand has had time to take its position of equilibrium under this force, and then be let go, the ratio of the amplitude of oscillation of the stand to that of the pendulum is not the initial one, but is very different from that. If this were the case, the motion of the stand and pendulum could not be represented, even approximately, in the form (1), for by those equations the logarithmic decrement of the oscillation of the stand is the same as that of the pendulum. It is true that the two parts of the oscillation (nearly in the natural periods of the pendulum and of the stind) have different logarithmic decrements; and, as the ratio of their amplitudes is not the same for the stand and for the pendulum, a certain change in the total relative amplitude might occur in this way, but only an excessively minute one, nothing like what M. Plantamour thinks he has observed. But it is so improbable that the motions of the stand and pendulum depart much from the forms (1) that if would be wrong to accept M. Plantamour's results until they are confirmed by a purely optical observation free from any possible influence from the machinery attached to the stand. Such an observation has been made by me, and, though I admit it was rather rough, it is entirely opposed to M. Plantamour's conclusions. Should the latter be confirmed, they would totally nullify the attempt to correct for the effect of flexure, as they would show the inapplicability of the analysis which has been proposed for the solution of that problem without affording us much hope of being able to replace it; and it would seem to be necessary in that case to reject all the work which has been done with the reversible pendulum.

If the pendulum were started in the manner proposed, and if for any cause the amplitudes of pendulum and stand were altered in different ratios, there would be a perpetual force at work tending to restore the old ratio, so long as the phases of the motion were the same in the pendulum and stand. But, if the phases differed, a part of this force would go to diminishing the amplitudes, and would act so strongly in this way that there would be a rapid decrement on account of this circumstance. Suppose, for instance, that in the differential equations we were to put instead of  $D_t^2 s_1$ , where s is the value of s at a time later than t by a constant. The result of this would be (neglecting terms involving a) that instead of the square of the exponent of the Naperian base being the sum of two negative quantities, one of them very small compared with the other, the smaller of these quantities would be multiplied by an imaginary root of unity. This would have but little effect on the imaginary part of the exponent of base, which determines the period; but it would add a considerable real part, which would represent a corresponding decrement of arc.

It seems difficult to conceive of a force which should greatly change the relative amplitudes of oscillation of the pendulum and stand, without at the same time producing an enormous decrement of the amplitude of oscillation, such as certainly does not exist. It is for those who believe that the existence of such a force has been experimentally proved to show how great an effect it would have upon the period of oscillation. M. Plantamour supposes that the formula given by me in my paper, "De l'influence de la ffexibilité du trépied sur l'oscillation du pendule à reversion," would still apply to such a case; but I am unable to see upon what ground.

Meantime, in the present state of the question, it appears to me that we must appeal to direct experiment to determine the difference between the time of oscillation on a stiff and on a flexible stand. Such experiments were given by me in the paper above mentioned, and I have since greatly multiplied experiments on a stiff stand, with the general result there announced, namely, that the difference is slightly greater than my theory supposes (owing, perhaps, to neglecting the energy of movement of the support), and not smaller, as M. Plantamour's views would require.

ON THE EFFECT OF THE VERTICAL ELASTICITY OF A PENDULUM SUPPORT.

Let s = the amount of depression of the support below its mean position;

c = the force of restitution;

l = the length of the simple equivalent pendulum;

h = the distance of the point of support from the center of mass of the pendulum;

 $\varphi$  = the angle which the axis of the pendulum makes with the vertical;

r = the perpendicular distance of a given particle of the pendulum from the knife-edge;

 $\omega$  = the angle which r makes with the plane through the knife-edge and the axis of the pendulum.

The horizontal velocity of a particle m is

$$r\cos(\varphi+\omega)\,\mathrm{D}_{\iota}\varphi.$$

The vertical velocity is

$$r\sin(\varphi+\omega)D_{t}\varphi-D_{t}s$$
.

The vis viva of the particle is, then,

$$\frac{1}{2}mr^2(D_i\varphi)^2 - mr\sin(\varphi + \omega)D_i\varphi \cdot D_is + \frac{1}{2}m(D_is)^2.$$

The vis viva of the pendulum is

$$\frac{1}{2}\mathbf{M}lh(\mathbf{D}_{r}\varphi)^{2} + \mathbf{M}h\sin\varphi\mathbf{D}_{r}\varphi\cdot\mathbf{D}_{r}s + \frac{1}{2}\mathbf{M}(\mathbf{D}_{r}s)^{2}.$$

The potential energy of the pendulum is

$$Mg(h-h\cos\varphi)+\frac{1}{2}es^2.$$

Lagrange's equations are consequently

$$Mlh D_t^2 \varphi - Mh \sin \varphi D_t^2 s = -Mgh \sin \varphi,$$

$$-Mh \sin \varphi D_t^2 \varphi - Mh \cos \varphi (D_t \varphi)^2 + MD_t s = -es.$$

If S be the amount by which the stand is statically compressed by the weight of the pendulum, then c=Mg:S, so that the equations become

$$lD_{t}^{2} \varphi - \sin \varphi D_{t}^{2} s = -g \sin \varphi, \tag{1}$$

$$-h\sin\varphi D_{t}\varphi - h\cos\varphi (D_{t}\varphi)^{2} + D_{t}^{2}s = -\frac{g}{8}s.$$
(2)

It is evident that small changes in  $\varphi$  will affect s insensibly, so that in determining s we may assume

$$\sin \varphi = \varphi, \quad \cos \varphi = 1, \quad \varphi = \Phi \cos \sqrt{\frac{g}{it}}.$$

The second equation then becomes

$$D^2_{,s} + \frac{g}{S}s = -\frac{gh\Phi^2}{l}\cos 2\sqrt{\frac{g}{l}}.t,$$

whence

$$s = \frac{hS}{4S - l} \Phi^2 \cos 2 \sqrt{\frac{g}{l}} t + C \cos \sqrt{\frac{g}{S}} (t - t_0).$$

The second term may obviously be neglected, and 4S may be neglected in comparison with l, so that

$$s = -\frac{hS}{l} \Phi^{2} \cos 2 \sqrt{\frac{g}{l}} t,$$

$$D^{2}, s = \frac{4ghS}{l^{2}} \Phi^{2} \cos 2 \sqrt{\frac{g}{l}} t = \frac{4ghS}{l^{2}} (2\varphi^{2} - \Phi^{2}).$$

Then the first equation becomes

$$D^2, \varphi = -\frac{g}{l} \left( 1 + 4 \frac{Sh}{l^2} \Phi^2 \right) \varphi + \frac{g}{6l} \left( 1 + 48 \frac{Sh}{l^2} \right) \varphi^2,$$

after substituting  $\varphi = \frac{1}{6}\varphi^3$  for sin  $\varphi$ , &c. Or, more briefly,

$$\mathbf{D}_{e}^{2}\varphi = -g'\varphi + \frac{1}{6}g''\varphi^{3}.$$

Putting  $\varphi = \sqrt{\frac{g'}{g''}}\theta$ , we get

$$\mathbf{D}_{t}^{2}\theta = -g'\theta + \frac{1}{6}g'\theta^{3},$$

whence

$$\begin{split} \mathbf{T} &= \bigcirc \sqrt{\frac{l}{g'}} \bigg( 1 + \frac{1}{16} \ \Phi^2 \bigg) \\ &= \bigcirc \sqrt{\frac{l}{l}} \bigg( 1 + \frac{1}{16} \frac{g''}{g'} \Phi^2 \bigg) \\ &= \bigcirc \sqrt{\frac{l}{g}} \bigg( 1 - \frac{2\mathbf{S}h}{l^2} \ \Phi^2 \bigg) \left( 1 + \frac{1}{16} \ \Phi^2 + \frac{3\mathbf{S}h}{l^2} \ \Phi^2 \right) \\ &= \bigcirc \sqrt{\frac{l}{g}} \bigg( 1 + \frac{1}{16} \ \Phi^2 + \frac{\mathbf{S}h}{l^2} \ \Phi^2 \bigg). \end{split}$$

If 
$$\frac{S}{l} = .0001$$
,  $\frac{h}{l} = .7$ , and  $\Phi = .05$ , then  $\frac{Sh}{l^2}\Phi^2 = .000000175$ .

so that the effect of the vertical elasticity of the support is insensible in ordinary cases.